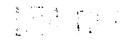
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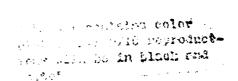


Technical Report 1317 December 1989

# Water Entry Structural Technique (WEST):

An Analytical Technique to Determine Frangible Nosecap Behavior During Water Entry

P. A. Jung R. C. Shaw





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## **ADMINISTRATIVE INFORMATION**

This work was performed for the Office of the Chief of Naval Research, Independent Exploratory Development Programs (IED), Arlington, VA 22217, under program element 060293. The work described was performed by members of NOSC Code 931.

Released by P. A. Jung, Head Structural Mechanics, Analysis, and Design Branch Under authority of C. L. Ward, Jr., Head Design and Development Division

# SUMMARY

## **OBJECTIVE**

Develop an efficient and accurate computational method for rational design of frangible nosecaps for air- and surface-launched undersea weapons.

## RESULTS

The powerful geometry and finite element model (FEM) pre- and post-processor PATRAN1, a potential-flow computer code that can calculate dynamic pressure-time histories of an arbitrary entry body called ENTRY, and the nonlinear finite element analysis (FEA) code ABAQUS have been effectively linked. This linkage allows rapid and accurate assessment of the state of stress and deformation of missile nosecaps intended to break up at water entry. This technique has been dubbed the Water Entry Structural Technique (WEST).

Two sample nosecap analyses are included in this report. One, a 90-degree cone, is an example of a known shape, previously experimentally characterized for water entry behavior. This shape serves as the control shape for results evaluation. The other is an engineering example of a nosecap that can be used on missiles like the vertical launch ASROC (VLA).

Since no single computer at the Naval Ocean Systems Center (NOSC) has the correct combination of processing power and graphics capability to host the various parts of WEST, portions of the codes are hosted on various computers at different locations within NOSC. The PATRAN pre- and post-processor is hosted on the NOSC Code 936-owned VAX 11/785 computer, named FLIPPER, as is the water entry portion of this code called ENTRY. The nonlinear FEA code ABAQUS is hosted on both FLIPPER and the General Purpose Computer Center (GPCC)-owned Convex minisupercomputer STINGRAY. A small computer code called YADAP, used for plotting results of pressure-time histories, is hosted on a personal computer (PC).

An abbreviated operating manual for WEST has been written and is included as appendix G.

#### CONCLUSIONS

The objective of this in-house Independent Exploratory Development (IED) effort has been met. The code linkage has been validated through comparison with experimental work. WEST is a valuable analytical tool that reduces the design cycle time for frangible nosecaps used for water entry.

<sup>&</sup>lt;sup>1</sup>PATRAN is a registered trademark of PDA Engineering, Inc.

## RECOMMENDATIONS

The code linkage process described in this report is suitable for a scientific engineering application within a Naval laboratory such as NOSC. It should not be considered a "production" code suitable for commercial application, as there are several "rough edges" within the execution of WEST. In particular, the limitations of the water entry portion of the code must be recognized during the development of the initial geometry and finite element model (FEM) within the pre-processor portion of WEST. While arbitrary shapes can be accommodated in the water entry process, there are limitations pertaining to mesh density, spacing, and numbering that preclude executing any old FEM that happens to describe a particular geometry of a nosecap.

The user must have a background in the PATRAN pre- and post-processor. This IED project does not attempt to reduce or foreshorten the complex task of translating nosecap drawings into "PATRANese". A FEA background is essential to create viable FEMs that will not only execute throughout WEST, but will truly produce "right" answers at the conclusion of the process. As with any FEA technique, it is often as much art as science, and not something that can be learned overnight by reading this report.

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# INTRODUCTION

Improvements to the analytical procedure described for the VLA nosecap can be made in the following areas:

- (a) ABAPAT translator revision. The PDA Engineering Release 3.0 ABAPAT translator has numerous "bugs" in it. In particular, while the outer fiber normal stresses are computed by ABAQUS and are contained in the FILENAME.FIL postprocessing file, ABAPAT cannot translate all of them into the FILENAME.STPiIj.NOD files needed for complete investigation of the results. According to PDA Engineering, Inc. this deficiency will be corrected in Version 4.0, which will soon be released.
- (b) Non-linear material model. ABAQUS is equipped to accommodate nonlinear stress-strain curves. As the plastic materials typically used in nosecaps are nonmetallic, increased accuracy would result by taking advantage of this feature. For the work accomplished for this IED project, linear-elastic materials properties were used.
- (c) **Nonlinear geometry model**. In addition to material nonlinearities, ABAQUS can accommodate geometric nonlinearities. These should be used in the area of the fin-shell interface of the VLA nosecap, to improve the accuracy of the relative deformations of the structural elements.

# **BACKGROUND**

Early investigations into the phenomena of water entry were concerned with the behavior of seaplane hulls upon landing. At the heart of these investigations was the determination of the surface pressures, forces, and moments on the entering object. Because of the mathematical difficulty of the problem, these investigations relied on many assumptions. Rather than attempt to rewrite a compilation of approaches found in the literature, an excerpt from Wardlaw, Morrison, and Baldwin (1977) follows.

(NOTE: Superscripted numbers in the following quoted passage refer to the original authors' work. For complete citations see Wardlaw et al. (1977), and the bibliography at the end of this report.)

Attempts to analyze the water-entry problem originate (circa 1929) with the work of Von Karman.¹ Comprehensive surveys of this field are provided by May,² Thingpen,³ Szebehely,⁴ and Moran.⁵ The main thrust of early work follows the formulation developed by Von Karman and Wagner.⁶ In this approach, a potential flow model is used and forces are calculated by the added mass concept. The submerged portion of the body is often fitted or replaced by another with the same surface cross-sectional area for which a closed form solution is available. A linearized version of the free-surface boundary conditions is applied to determine the surface shape. Most of the theories are

restricted to vertical entry of simple geometries. In recent years, computational efforts have been made to obtain a solution using the nonlinear boundary conditions. An early example of such work is that of Chu and Falconer. A relaxation method was used to solve the potential problem for arbitrary bodies. This project was abandoned due to problems with excessive computational time and surface contact discontinuities. The same formulation for the vertical entry of cones has been treated by Weber8 using a distribution of source dipoles. More recently, Shere and Vander Vorst9 and Vander Vorst and Rogers10 have used the marker and cell method to develop a detailed viscous model of vertical cone entry.

Although the description of the pressures, forces, and moments on entry bodies is of major importance to designers of water-entry bodies, the ability to characterize these loads is only part of the solution to the general engineering design problem of frangible nosecaps. Once the water-entry loads are known, the structural performance of the entry body must be obtained, and the optimization of the body made to accommodate these loads.

The application of traditional analytical tools to this type of engineering design has been supplemented in the last 15 years by the FEA approach. FEA codes are currently available, such as MSC/PAL-II,2 CASA/GIFTS,3 and NISA II-PC4 which run on PCs on a design engineer's desktop. More powerful versions of these codes, such as MSC/NASTRAN5 execute on higher power workstations, mainframe, and supercomputer platforms.

These codes, however, (especially those hosted on PC platforms) are normally used for linear analyses, which are necessarily limited to small displacements and a constant relationship between stress and strain in the material. Unfortunately, structures designed to fail (frange) upon water impact, such as torpedo nosecaps used in weapon systems for air- or surface-launched antisubmarine warfare, cannot be analyzed by the linear-elastic method. Due to the nature of the design, these structures undergo relatively large strains during the entry phenomena, well beyond the application of linear stress analysis. In addition, these structures are necessarily fabricated from some sort of engineering plastic, which typically has an extremely nonlinear stress-strain curve. Also, resolution of stresses within complicated nosecap geometries can result in large problem sizes. Lastly, these structures usually possess some form of nonlinear contact boundary condition, which cannot be handled by linear-elastic theory. These limitations become insurmountable when dealing with linear FEA codes.

In the early 1980s, NOSC was tasked to design a frangible nosecap for the VLA. Because of the highly streamlined profile of the nosecap required for Mach 2 flight, the subsequent pressure profile generated over the nosecap surface upon water entry was significantly smaller than on the antisubmarine rocket (ASROC) nosecap,

<sup>&</sup>lt;sup>2</sup>MSC/PAL-II is a registered trademark of the MacNeal-Schwendler Corporation.

<sup>&</sup>lt;sup>3</sup>CASA/GIFTS is a registered trademark of Computer-Aided Structural Analysis, Inc.

<sup>&</sup>lt;sup>4</sup>NISA II-PC is a registered trademark of Engineering Mechanics Research Corporation.

<sup>&</sup>lt;sup>5</sup>MSC/NASTRAN is a registered trademark of the MacNeal-Schwendler Corporation.

upon which the initial design was based (Jung, 1984). The VLA nosecap showed a marked propensity to remain intact upon water entry, rendering the torpedo payload useless.

A new design concept was subsequently developed by NOSC, in collaboration with Loral Systems Group, the prime contractor for VLA. In this design, an ogival-shaped plastic shell is filled with segmented rigid foam. The tip of the shell is cut off at approximately the 5-inch-diameter station to form a "hollow point" nosecap shell. A multipound, lead-filled, aluminum nosetip is attached to the front of the shell. The nosetip is designed to remain in place from launch through airframe separation, to provide an aerodynamic profile. At parachute deployment, the nosetip is to fall away, revealing a high-drag, hollow-nosecap shell designed to break apart at water entry. Although complicated by a difficult and restrictive design envelope, a workable design was developed.

In 1988, an effort to find alternative designs was initiated by Jung and Plapp (1988). Two designs were investigated: an ogival shell stiffened with tapered foam columnar supports, and an ogival shell with blade stiffening. Wardlaw's, et al. (1977) computer code ENTRY was used to determine the entry body pressure profile. The linear analysis capabilities of MSC/NASTRAN were used to generate the subsequent stress and displacement time histories of the nosecaps. The analysis revealed two shortcomings: First, the manual coding of ENTRY to produce a suitable pressure-time distribution on the VLA profile for FEA was extremely laborious and error-prone. Second, the linear capabilities of MSC/NASTRAN did not accurately predict the structural response of the nosecaps to the dynamic water entry load. Significant concerns were voiced by Jung and Plapp (1988) to the effect that potentially viable alternatives to the "hollow point" VLA nosecap were being discarded due to the lack of suitable analytical tools to properly develop the design of these alternatives.

This in-house IED project was developed to assuage that concern, and significantly improve NOSC's capabilities to design frangible nosecaps for water entry.

# **APPROACH**

As extensively documented in the literature, the mathematical approach to the determination of the water-entry loads on a nosecap is extremely complex computationally. Therefore, some type of computer code is necessary to solve the frangible nosecap design problem. A literature search revealed only a few codes, such as PHOENICS6 and PISCES-3DE,7 that can solve fast transient compressible flow phenomena to characterize the pressure-time history during water entry. Unfortunately, these are not public-domain softwares, and are consequently extremely expensive to own or lease. As a goal of this project was to obtain an inexpensive solution to the water entry structural problem, these codes were not pursued.

In the late 1970s, an engineering code was developed by the Naval Surface Weapons Center/White Oak Laboratory (NSWC/WOL) for calculating pressures and

<sup>&</sup>lt;sup>6</sup>PHOENICS is a registered trademark of CHAM of North America, Inc.

<sup>&</sup>lt;sup>7</sup>PISCES-3DE is a registered trademark of Physics International Company, a California company.

loads at high speed water entries. Rather than attempt to rewrite in original form the authors' intent, an excerpt from Wardlaw et al. (1977) is included:

...The present approach differs from such efforts (Von Karman, May, Moran, et al.) in several important aspects. Application of Hess and Smith's numerical solution method allows arbitrary bodies to be treated and makes it possible to calculate pressures on the surface of the body. These pressures are themselves of interest and can be integrated to provide both force and moment information..."

The flow field about the entry body is described by Laplace's equation. The boundary condition

$$-\nabla\phi\cdot e_n = V_E\cdot e_n\tag{1}$$

is applied on the entry body and  $\phi=0$  to the effective planar surface. This surface is assumed to rise at the rate  $(C_w-1)$   $V_E$  sin  $(\theta)$ . Pressures are calculated from successive solutions at differing depths using the unsteady Bernoulli's equation, which is cast in a reference frame moving with the model

$$C_p = \left[ 2 \frac{\partial \phi}{\partial t} - Z \overline{V}_E \cdot \nabla \phi - (\nabla \phi)^2 \right] / V_E^2$$
 (2)

The preceding equation produces pressure and force coefficients that are independent of model scale and entry velocity. The value of these two parameters must be simulated through an appropriate choice of the rate of surface rise (i.e., picking the correct  $C_w$  value).

To implement the potential flow solution, the nose of the entry body is approximated with a series of planar, quadrilateral elements. Each of these elements is defined by four points or nodes lying on the body surface. The computation proceeds by inserting the model into the water in a series of steps, each at a depth,  $\Delta h$ , greater than the previous one. At every step, the group of elements comprising the submerged portion of the model is redefined and arranged into a form amenable to the potential flow calculative procedure. The nodes defining a particular element are checked to determine whether they are above or below the water line. Elements with all four nodes above the water surface are discarded, whereas those elements with all four nodes below it are included without change. Elements which are intersected by the water surface are redefined with an upper edge coincident with it. A constant source strength is assumed distributed over the surface of each element, and the value of this strength is determined by satisfying eq. 1 at the centroid of each element. The boundary condition,  $\phi = 0$ , is satisfied by locating an image of each element of opposite source strength above the water surface.

At each depth, the pressure coefficient,  $C_p$ , is evaluated at element centroids using eq. 2, which is a body-fixed frame of reference. The quantity  $\nabla \phi$  appearing in this equation is determined directly at each depth, but must be calculated using the value of  $\phi$  at the same body location in adjacent steps. In most cases, this requirement does not pose a problem, since  $\phi$  is calculated at the element centroids that are

at fixed locations on the entry body. Defining  $\phi_{c_n}$  to be the value of  $\phi$  at a specific element centroid at step n and using the central differences,

$$\frac{\partial \phi}{\partial t} = \frac{\partial h}{\partial t} \frac{\partial \phi}{\partial h} = V_E \sin \theta \ C_w \left[ \phi \frac{C_{n+1} - \phi}{2\Delta h} \ C_{n-1} \right]. \tag{3}$$

The foregoing procedure cannot be applied directly to elements that are intersected by the water surface. Under these conditions, the element centroid location on the entry body changes slightly from step to step and certain approximates must be introduced.

The computational method has been applied to the oblique entry of disk cylinders, spheres, and to the vertical entry of cones and spheres. In each case  $C_w$  is assigned a value consistent with existing experimental data. The computed pressure coefficients are in good agreement with measurements on the wide variety of bodies studied. The calculated pressures reflect not only basic body geometry, but also position on the model surface. The peak pressures tend to be somewhat larger than the measured ones. Fortunately, these act over very small areas and thus do not affect the calculated loads significantly....

This computational method is described in detail by Wardlaw et al. (1977), in which the computer code ENTRY is described, written in FORTRAN for a Control Data Corporation (CDC) computer. As this is public domain software, and thus available at no cost to NOSC, it was obtained from NSWC/WOL and installed by Wardlaw and Youngs from NSWC/WOL on the NOSC computer FLIPPER.

Describing the geometry of the entry body in a form that ENTRY can read can be a time-consuming and error-prone task. No graphics pre-processor exists for ENTRY. There is no post-processor to enable graphical display of the resulting pressure-time slices on the geometry either. As it was the intent of this IED task to develop a general purpose computational capability, it was decided to alter ENTRY to enable it to interface with the commercial pre- and post-processor PATRAN. Unfortunately, PATRAN is not available as public domain software. This compromise was accepted upon receipt of assurances from the GPCC for long-term support for PATRAN at NOSC.

There are a number of large, nonlinear FEA codes on the open market such as DYCAST/GC,8 and MARC.9 Unfortunately, most of them are extremely expensive to obtain, either on a purchase or a lease basis. Some, such as DYNA-3D, are available as public domain software, but are so user "unfriendly" as to make them virtually useless to the less-than-dedicated user.

ABAQUS, 10 a general purpose FEA program, has special emphasis on nonlinear structural engineering applications. It has extensive material, element, and procedure libraries. Material models include plasticity for metals as well as rubber

<sup>&</sup>lt;sup>8</sup>DYCAST/GC us a regustered trademark of the Grumman Corporate Research Center.

<sup>&</sup>lt;sup>9</sup>MARC is a registered trademark of MARC Analysis Research Corporation.

<sup>&</sup>lt;sup>10</sup>ABAQUS is a registered trademark of Hibbitt, Karlson, & Sorenson, Inc.

elasticity. It features both user-specified and automatic control of step size and a simple, compact problem-definition language. In addition, the David Taylor Research Center (DTRC) makes this code available to all Naval laboratories through the Naval Engineering Software Support (NESS) Office, for a nominal annual fee. This code was chosen for implementation in this IED program.

Figure 1 describes the linkages and program flow to interconnect PATRAN with ENTRY and ABAQUS. Figure 1 also describes the various translators that were written to facilitate this linkage. The ability to execute this data flow is the final product of this IED project.

Figure 1 illustrates the flow of information, from the geometry and FEM preprocessor PATRAN (black figures) through the water entry code ENTRY (red figures), and into the FEA code ABAQUS for stress and displacement analysis (green figures), then back again to PATRAN to view the resultant stress and displacement.

# FRANGIBLE NOSECAP ANALYSIS

To illustrate this process, a step-by-step example for a 90-degree (included angle) cone-shaped nosecap is presented (at times referred to by bullet numbers).

The analysis process begins with the construction of the geometry of the nosecap via the PATRAN Phase I modeling technique on FLIPPER. This geometry model is then meshed with nodes, and four-noded quadrilateral shell elements to produce the Phase II FEM within PATRAN (bullet 1). Figure 2 is a hardcopy image of the cone, modeled as a half-symmetric nosecap, created within PATRAN. This image shows 100 nodes, and 84 four-noded quadrilateral elements which form the Phase II finite element PATRAN model. Figure 3 is a side view of the nosecap, showing the 90-degree included angle. Figure 4 is an expanded view of the tip of the nosecap, showing the four quadrilateral elements used to mesh the flat tip of the nosecap, and node 100 at the center of the truncated tip.

Note that one of the peculiarities of the ENTRY code is that it will not execute a pressure-time history on a pointed (zero radius) entry body. Thus, it is necessary to create a very small flat tip on any pointed geometry to accommodate this quirk.

Upon completion of this Phase II FEM, a request for neutral file generation is made within PATRAN. This neutral file contains only the Phase II FEM, and is named PATRAN.OUT (bullet 2). This completes the mesh definition portion of the problem, and PATRAN is exited. Exiting of PATRAN results in the generation of the default PATRAN data file PATRAN.DAT (bullet 3), which must be manually renamed to FILENAME.PAT (bullet 4). ("FILENAME" is a user-optional identifier for the particular nosecap.) This completes the initial PATRAN pre-processing portion of the analysis task. All files generated to this point now reside in the user's space on FLIPPER.

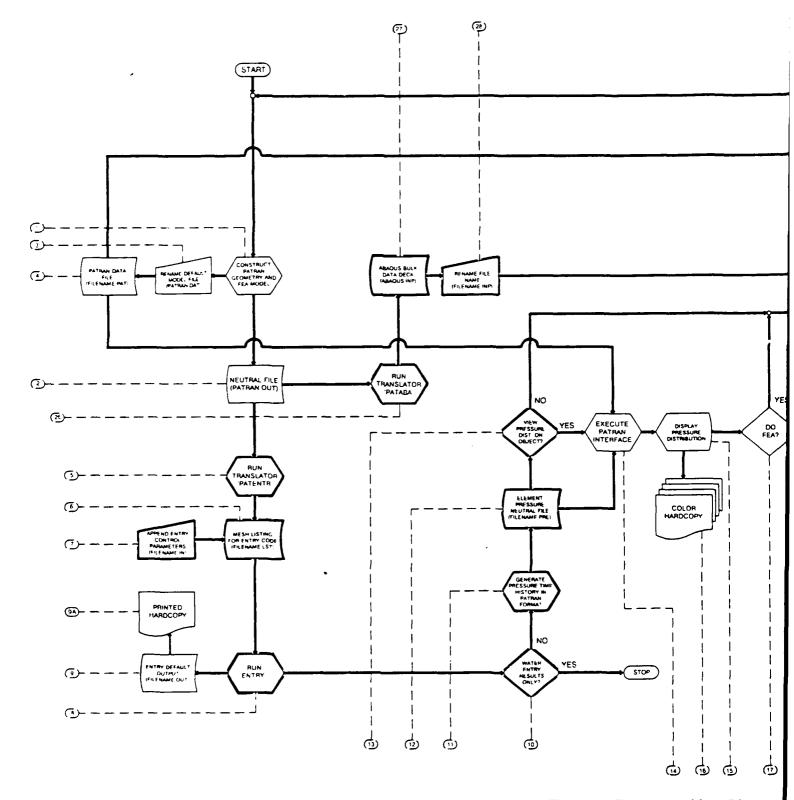
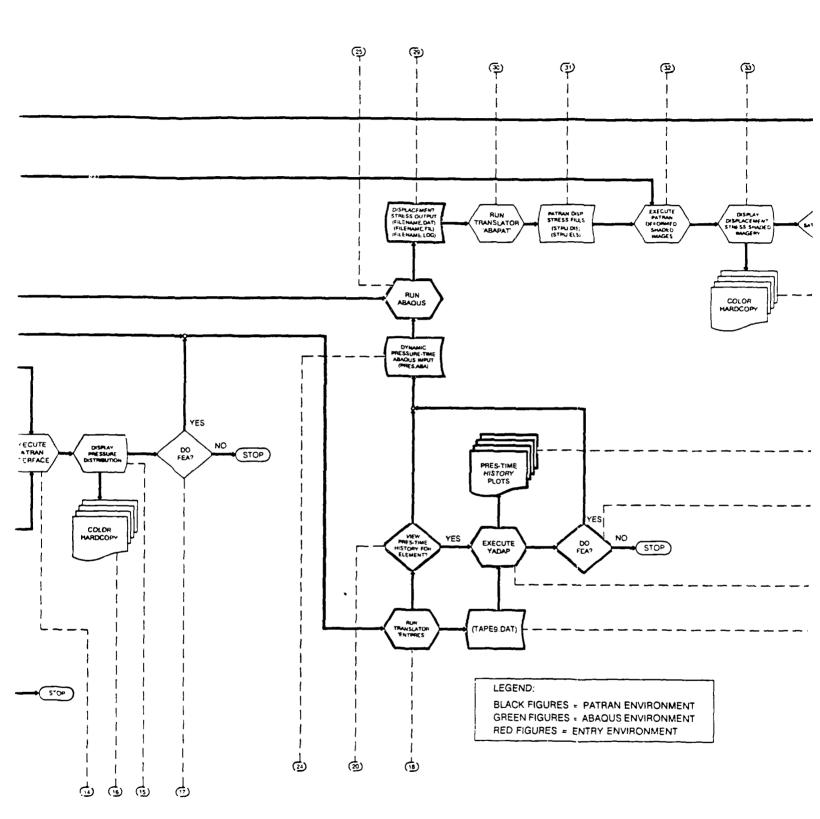


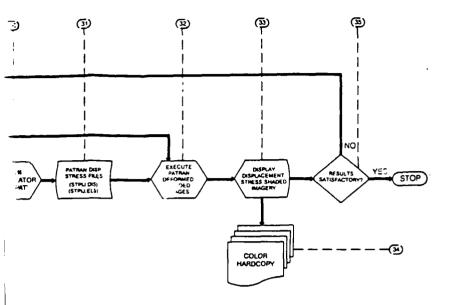
Figure 1. Flow chart of frangible nose

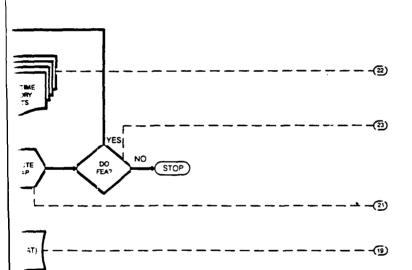
1:13



. Flow chart of frangible nosecap analysis at water entry.

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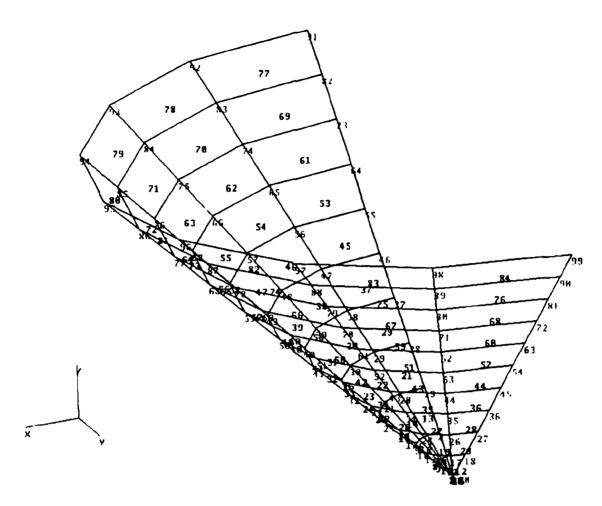


Figure 2. Half-symmetric water-entry and FEA model of a 90-degree cone.

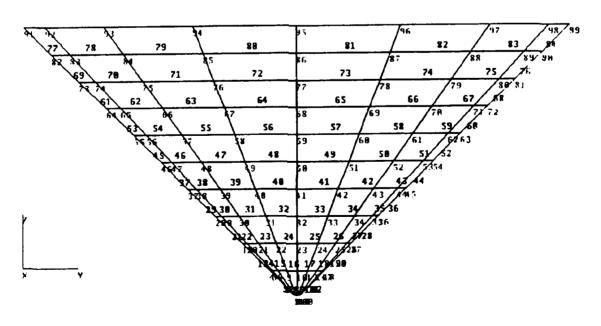


Figure 3. Side view of 90-degree blunted cone model.

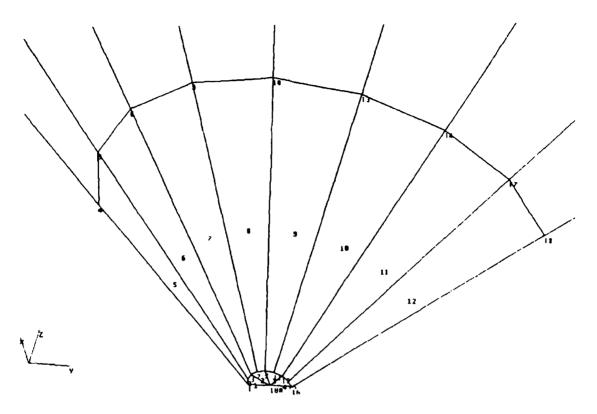


Figure 4. Enlarged view of the tip area of the 90-degree blunted cone.

Bullet 5 is a translator program written for this project called PATENTR. PATENTR resides on, and is executed from the operating system of FLIPPER to translate the nodal definition and connectivity generated within PATRAN in neutral file format (PATRAN.OUT, bullet 2) into a nodal definition and connectivity format readable by the ENTRY code. The output of PATENTR is a file called FILE-NAME.LST (bullet 6). Using the FLIPPER text editor, a file containing ENTRY control parameters (entry velocity, angle, wetting factor, etc.) is then appended to FILENAME.LST and renamed to FILENAME.IN (bullet 7). The sample ENTRY input file CONE90.IN is included in appendix A for reference. At this point, ENTRY is ready for execution.

ENTRY is executed from the FLIPPER operating system (bullet 8), using FILENAME.IN for the ENTRY input file. This process automatically produces the ENTRY default output file FILENAME.OUT (bullet 9), and the file FILENAME.PRE (bullet 12), which are saved to the user's space on FLIPPER. The output file FILENAME.OUT may be edited by the FLIPPER text editor or printed at the user's discretion. It contains a complete listing of the water entry parameters, the nodal definition and connectivity of the geometry, pressure-time history of each element, integrated forces, and moments over the entire nosecap. The file CONE90.OUT is included in appendix B as an example of a typical ENTRY output file.

Within ENTRY is an option to produce only the file FILENAME.OUT, and exit the entry process (bullet 10). This option would be chosen early in the analysis process, for error checking, steady-state drag coefficient calculations, or use of ENTRY for some other purpose. If this option is not exercised, then ENTRY uses a

portion of the ENTRY code (bullet 11) to produce the file FILENAME.PRE (bullet 12). This option discards all extraneous information printed in FILENAME.OUT, and rearranges the pressure-time history on each element into a format that can be read by PATRAN. FILENAME.PRE is created in the user's space on FLIPPER.

At bullet 13, a decision is made whether or not to view the pressure distribution displayed at the centroids of the elements of the nosecap as a function of time during the entry event. If desired, PATRAN is reentered at bullet 14. This execution of PATRAN calls upon the PATRAN data file FILENAME.PAT (bullet 4), and the formatted results file FILENAME.PRE (bullet 12) to produce color displays of the pressure distribution overlaid on the FEM. These graphics images, one per time-step, are displayed on a suitable color graphics terminal (bullet 15). These images may be captured on hardcopy (bullet 16) if desired.

Figures 5 through 15 are typical hardcopy images which illustrate this step. These figures show the pressure distribution on the 90-degree nosecap during the water entry event. Figure 5, at 0.16 ms after vertical entry, shows elements 1 through 4 fully wetted. The pressure, displayed at the centroid of the quadrilateral elements 1 through 4, is 24.1 psi. Figure 6, at 0.80 ms after entry, shows the pressure on the flat tip rising to 129.0 psi, and the pressure on the ring of elements 5 through 12 wetted with a pressure of 131.0 psi at their centroids. Figure 7, 1.61 ms after entry, shows the nosecap wetted to the upper edges of elements 13 through 20. The tip pressure has decayed to 106.0 psi, while the pressure on elements 5 through 12 has decayed to 95.7 psi, and the pressure on elements 13 through 20 has risen to 135.0 psi. By following figures 5 through 15, the progression of the nosecap entry into the water, as well as the progression of the pressure distribution up the nosecap, can be seen. At figure 15, full submergence has occurred after 8.06 ms. The pressure has decayed from a high of 129.0 psi at the tip, to a value of 79.2 psi around the ring of elements 53 through 60, and has not yet decayed to steady state along the upper ring of elements 77 through 84, which have just been wetted. The pressure peak has passed up the nosecap, while the earlier-wetted elements have reached their steady state drag phase pressure profiles.

Referring back to figure 1, bullet 17 is a decision whether or not to proceed with the FEA portion of this analysis task. If the color hardcopies of the pressure distribution, such as figures 5 through 15, are satisfactory to complete the task at hand, and no FEA work is intended on the nosecap FEM, then PATRAN may be exited at bullet 17 and the task completed. However, if the pressure distributions are acceptable to the analyst, and additional FEA work is required, then PATRAN is exited at bullet 17 and the WEST process rejoined.

The translator ENTPRES (bullet 18), another special-purpose translator written for this project, resides on FLIPPER. Execution of ENTPRES from the operating system of FLIPPER produces a file called TAPE9.DAT (bullet 19), and the file PRES.ABA (bullet 24), both stored in the user's space on FLIPPER.

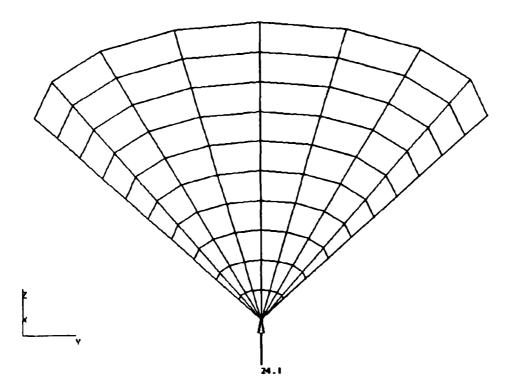


Figure 5. Ninety-degree blunted cone impacting water at 100 ft/s, pressure distribution at t = 0.1612 ms (step 1, initial wetting).

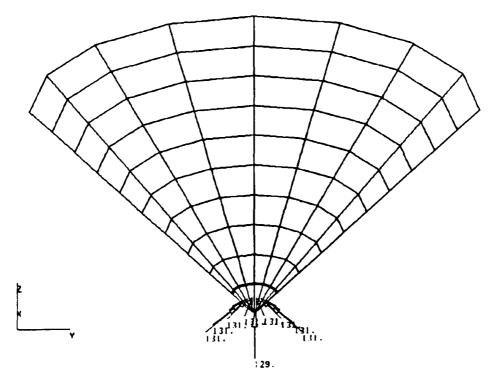


Figure 6. Pressure distribution on 90-degree blunted cone at t = 0.8068 ms (step 2).

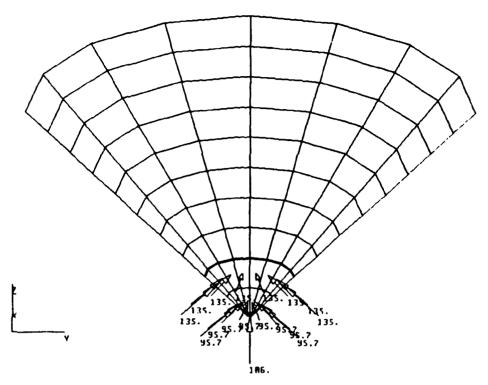


Figure 7. Pressure distribution on 90-degree blunted cone at  $t \approx 1.6119$  ms (step 3).

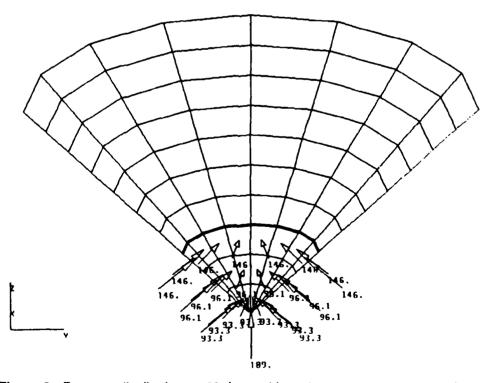


Figure 8. Pressure distribution on 90-degree blunted cone at  $t \approx 2.4179$  ms (step 4).

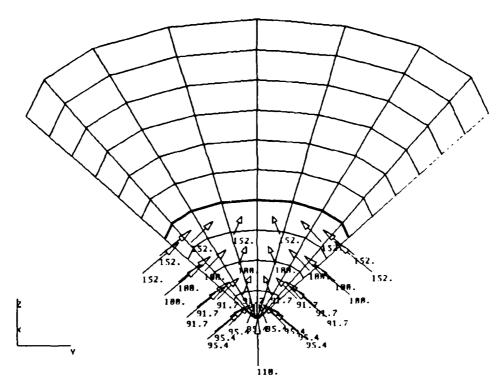


Figure 9. Pressure distribution on 90-degree blunted cone at t = 3.2238 ms (step 5).

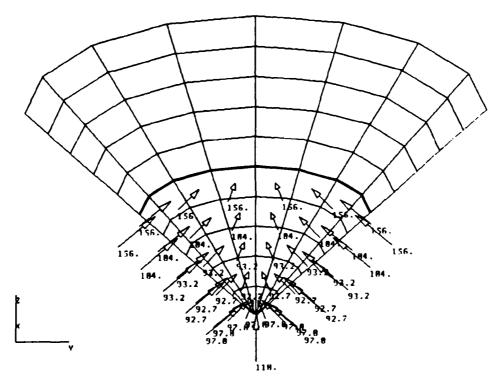


Figure 10. Pressure distribution on 90-degree blunted cone at t = 4.0298 ms (step 6).

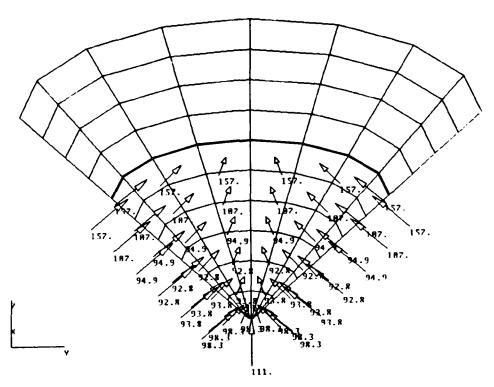


Figure 11. Pressure distribution on 90-degree blunted cone at t = 4.8357 ms (step 7).

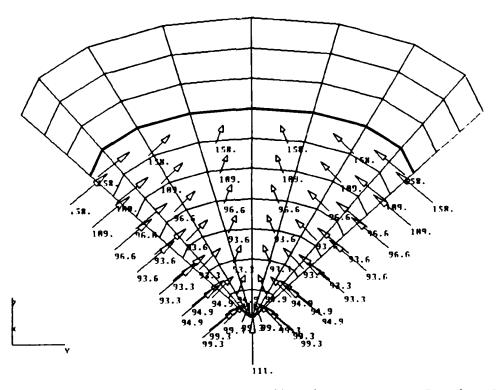


Figure 12. Pressure distribution on 90-degree blunted cone at t = 5.6417 ms (step 8).

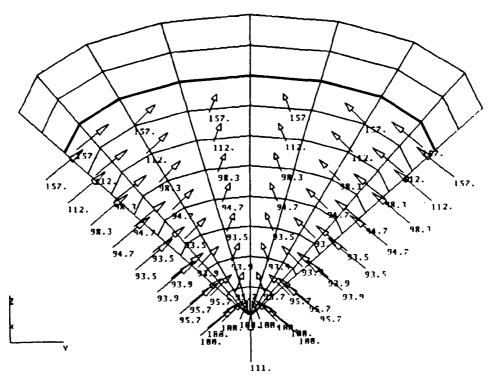


Figure 13. Pressure distribution on 90-degree blunted cone at t = 6.4476 ms (step 9).

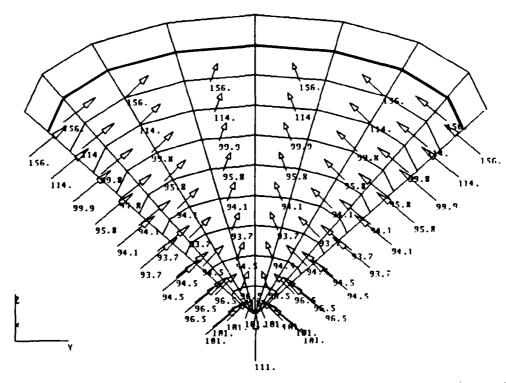
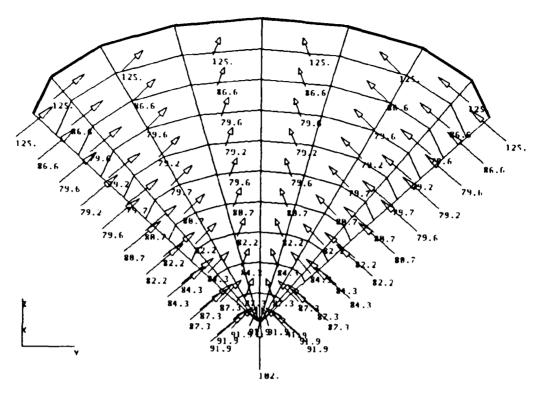


Figure 14. Pressure distribution on 90-degree blunted cone at t = 7.2536 ms (step 10).



**Figure 15.** Pressure distribution on 90-degree blunted cone at t = 8.0595 ms (step 11, fully submerged).

At bullet 20, a decision is made whether or not to view the pressure-time history on an elemental basis. If the decision is made to do so, then the user must log off FLIPPER and go to the personal computer that hosts YADAP (bullet 21). YADAP is an interactive plotting package that plots the pressure on each element as a function of time. Through the General Communications Backbone (GCB), a remote login to FLIPPER must be made. The file TAPE9.DAT (bullet 9) must then be transmitted via the MCP communications protocol from the user's space on FLIPPER to the user's hard disk on his PC. After executing YADAP, the elemental time-history plots can be captured as hardcopy images on a suitable graphics printer or plotter (bullet 22).

Figure 16 illustrates this portion of the analytical process, showing pressuretime histories for "rings" or "groups" of elements. Element group 1 consists of elements 1 through 4 on the flat face, element group 2 is the ring of elements 5 through 12, and so forth. These YADAP plots show, on an elemental basis, how the pressure peaks at a later time the farther up the nosecap the element is located, as well as the reduction in peak pressure the farther from the nosecap tip the element is located.

Another decision point is reached at bullet 23. Here, if the analyst decides that there is enough information about the problem, he/she can exit the process. However, if the analyst is satisfied with the pressure distribution time-histories to date, (figure 16) but wishes to do the FEA on the nosecap, he/she returns to the main program flow in the program stream.

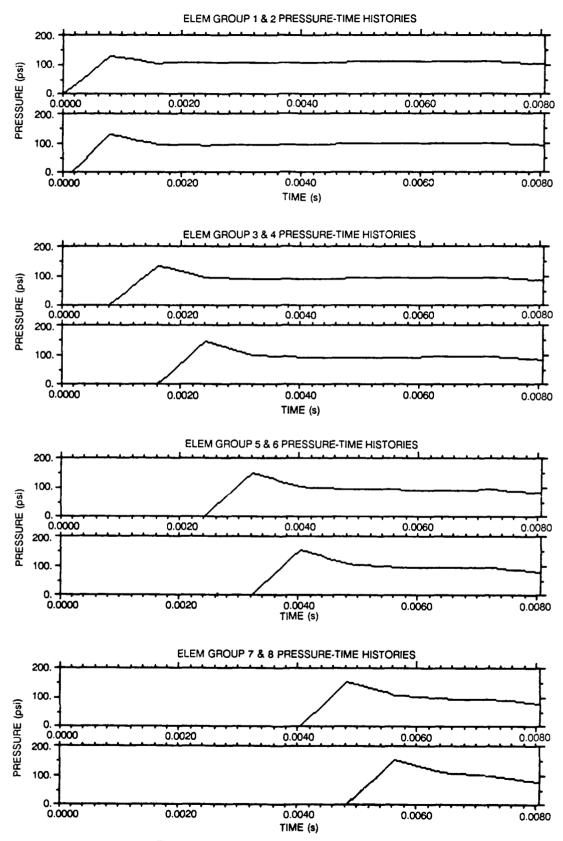


Figure 16. Pressure-time histories (0.000-0.008 s).

Earlier in the process, the translator ENTPRES (bullet 18) produced a file called PRES.ABA (bullet 24), which was stored in the user's space on FLIPPER. This file is a portion of the "history" input file needed by ABAQUS (bullet 25) for the FEA of the nosecap. To execute ABAQUS and produce the FEA of the nosecap, it is necessary to execute the translator PATABA (a PDA Engineering translator, part of the PATRAN software) from FLIPPER (bullet 26). This translator operates on the PATRAN neutral file PATRAN.OUT (bullet 2) previously created and stored on FLIPPER, and produces the ABAQUS model input file called ABAQUS.INP (bullet 27). This file must then be manually renamed to FILENAME.INP (bullet 28). After appending it to the file PRES.ABA (bullet 24), it is possible to complete the FEA portion of this process by executing ABAQUS in bullet 25.

ABAQUS is executed from the operating system of FLIPPER, using FILE-NAME.INP as the input file. Successful execution of ABAQUS in bullet 25 on FLIPPER produces several permanent files of the analyst's choice (bullet 29), and a myriad of temporary files which are erased at program halt. These files are stored in the user's space on FLIPPER. Typically, the following are the files which are retained at this point:

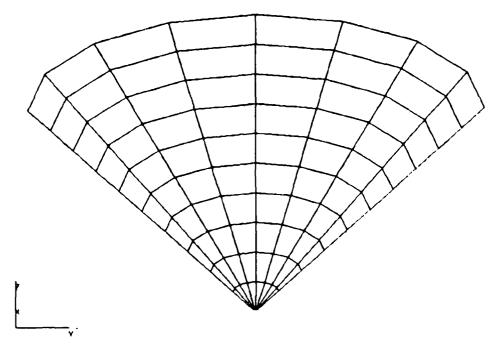
FILENAME.INP FILENAME.DAT FILENAME.FIL FILENAME.LOG input file output file post-processing output file accounting file

The sample ABAQUS input file for the 90-degree cone (CONE90.INP) is shown in appendix C.

Upon completion of ABAQUS, the translator ABAPAT (another PDA Engineering translator, bullet 30) is called upon to operate on the post-processing output file FILENAME.FIL (bullet 29). ABAPAT produces the PATRAN displacement, elemental stress, and average nodal stress files (bullet 31) called STPiIj.DIS, STPiIj. ELS, and STPiIj.NOD respectively. In these files, i is the step number, and j is the increment number within each step. Obviously, a complex nonlinear dynamics problem can have many steps, and numerous increments within each step. Thus, this portion of the code tends to produce massive amounts of data, that the analyst is well advised to prescreen for need.

Once the displacement and stress files are complete in bullet 31, the post-processing of the results can begin. In bullet 32, after executing ABAPAT, the PATRAN environment is re-entered. Here, after calling up the PATRAN model file FILENAME.PAT (bullet 4), displaced geometry and stress contours for each time increment during the entry process can be created. These images can be viewed on the appropriate graphics terminal (bullet 33) or output as color hardcopy in bullet 34.

For example, figure 17 is a hardcopy image of the displacement of the 90-degree nosecap 0.16 ms after entry. As the tip is just wetted, no appreciable deformation of the nosecap structure has occurred. Figure 18, at 0.80 ms after entry, shows the beginnings of deformation of the nosecap under the pressure distribution previously illustrated at this same time-step in figure 6. By following the progression from figures 17 through 23, the deformation of the nosecap in response to the pressure-time history can be clearly seen. This deformation is greatly enlarged for clarity.



**Figure 17.** Deformed shape of 90-degree blunted cone at t = 0.1612 ms (step 1, initial wetting).

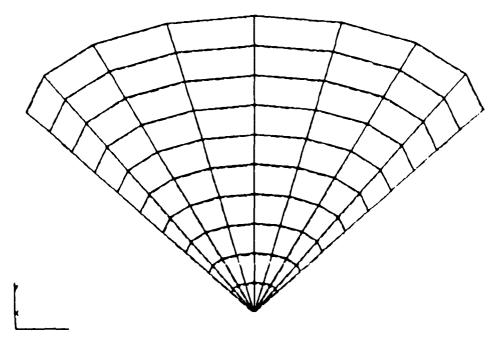


Figure 18. Deformed shape of 90-degree blunted cone at t = 0.8060 ms (step 2).

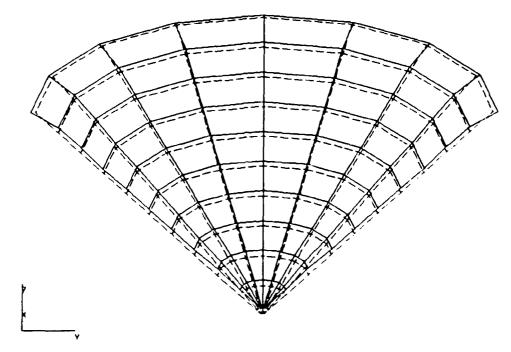


Figure 19. Deformed shape of 90-degree blunted cone at t = 1.6119 ms (step 3).

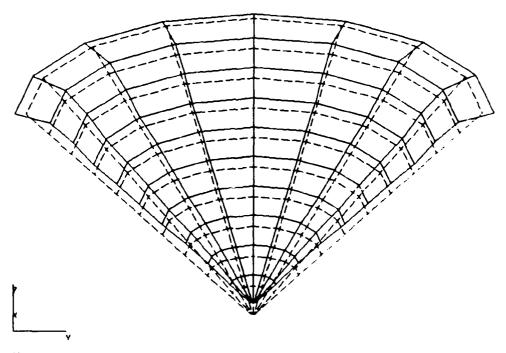


Figure 20. Deformed shape of 90-degree blunted cone at t = 2.4179 ms (step 4).

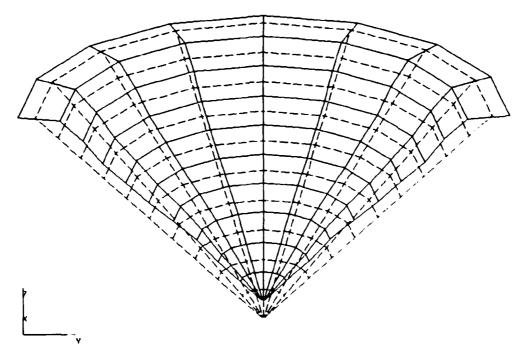


Figure 21. Deformed shape of 90-degree blunted cone at t = 3.2238 ms (step 5).

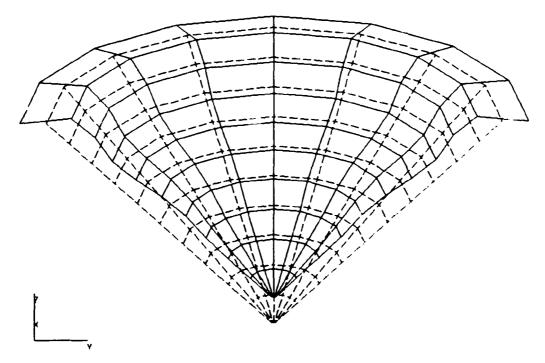


Figure 22. Deformed shape of 90-degree blunted cone at t = 4.0298 ms (step 6).

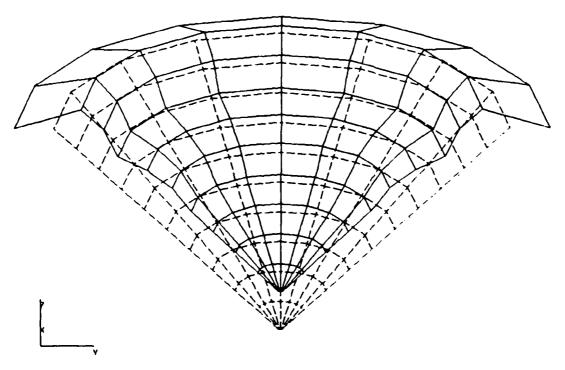


Figure 23. Deformed shape of 90-degree blunted cone at t = 4.8357 ms (step 7).

Deformations of elastic bodies result in associated stresses. Figure 24, a maximum distortion energy theory (Von Mises) stress plot, shows the state of stress of the nosecap at time-step 2, 0.80 ms after water entry. As only the tip of the nosecap has been wetted, the Von Mises stresses have only built to a low value on the nosecap. Note that, since no effort was made to accurately model the upper supported edge of this nosecap mode, the actual stress values reported in the stress spectrum along the right edge of the figure are meaningless. They serve only as relative values during the event.

Figures 25 and 26, time-step 3 at 1.61 ms after entry, indicate that the stress levels in the blunted tip have risen to a level exceeding  $2.72 \times 104$  psi.

Figures 27 through 30 are a progression of the Von Mises stresses through the nosecap during the water-entry process, at the same time-steps as the previously displayed deformation histories (figures 17-23). The stress levels along the edge can be seen to fall during the entry process, and then rise again as the stress wave approaches this boundary. As the stress wave follows the pressure pulse up the shell, the boundary condition imposes an inordinate amount of bending into the shell at the upper edge. This bending skews the stress contouring to the point where the analysis is considered unrealistic beyond the point where the nosecap is fully wetted.

As this nosecap has been included only to illustrate the flow of information through this analytical process, the final few figures indicate some of the details of this discipline that require extreme attention by the analyst. Blind faith in computer-generated numbers easily leads to unrealistic expectations from the FEA approach and dangerously incorrect answers.

Assuming the analyst is satisfied with the progress to date, production of these displacement and stress contours completes the program flow at bullet 33. The propensity of the nosecap to break up at some point in the entry sequence can be determined at this time and a decision made on optimization of the nosecap structure. Obviously, if the results are not satisfactory, the program can be started over (bullet 35), or exited at this point.

# RESULTS

Jung and Plapp's (1988) technical report (TR 1221) dealt with a frangible nosecap design for the VLA program. In that document, it was reported that a frangible one-piece nosecap was not suitable for the current VLA, due to the difficult, restrictive design envelope.

The approach used to develop this conclusion was a manual design optimization of the nosecap shell. The water entry pressure profile was generated using ENTRY, and the resultant stress analysis completed using the linear-elastic capabilities of the MacNeal/Schwendler version of MSC/NASTRAN.

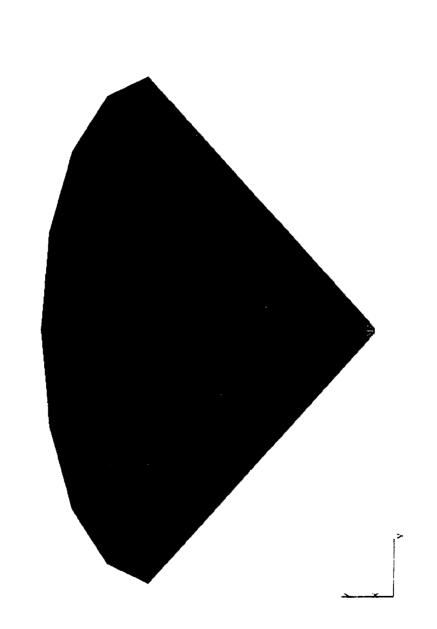
To further illustrate the engineering application of the WEST developed in the current IED project, it was decided to review the water entry portion of the VLA nosecap analysis.

Figure 31 is the nodal pattern of a half-symmetric 1.25-caliber Von Karman ogival nosecap water entry FEM, suitable for use on the VLA missile. This nosecap consists of 262 nodes on the exterior of the shell. Figure 32 is a plot of the 228 four-noded quadrilateral elements which form the elemental connectivity for the water entry model, while figure 33 shows an enlarged view of the tip of the nosecap FEM. These figures were produced within the PATRAN portion of WEST (bullet 1, figure 1).

Appendix D is the resultant ENTRY input, entitled CAP90.IN;6. This is the result of bullet 7 in figure 1 and is used in the execution of the ENTRY portion of WEST in bullet 8.

Appendix E is the hardcopy output from ENTRY (bullet 9a). Figures 34 through 40 show the plotted resultant pressure distribution as a function of time for the ogival nosecap undergoing vertical entry at a velocity of 130.0 ft/s during the time period 0.0 to 3.92 ms. Note that only even time-steps after step 2 are shown for brevity.

These pressure-time histories are shown as element groups (elements grouped as rings in the axial direction) in figure 41. This figure shows how the pressure peak decays in time during water entry.



2251.\_

1941.

2568.

2869.

4187.

4726.

4416.

3798.

3488.

3179.

1813.

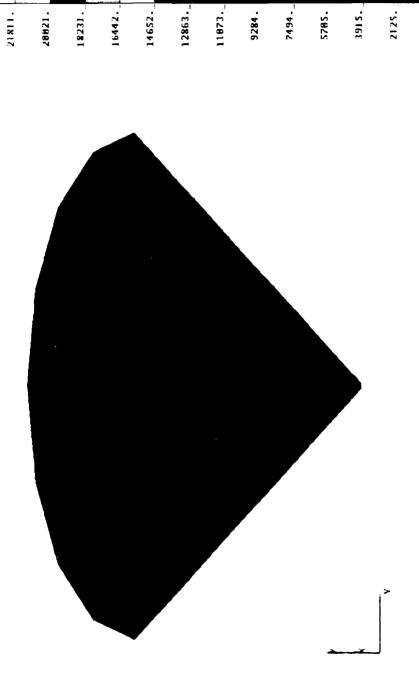
1323.\_

1632.

395.

85.3

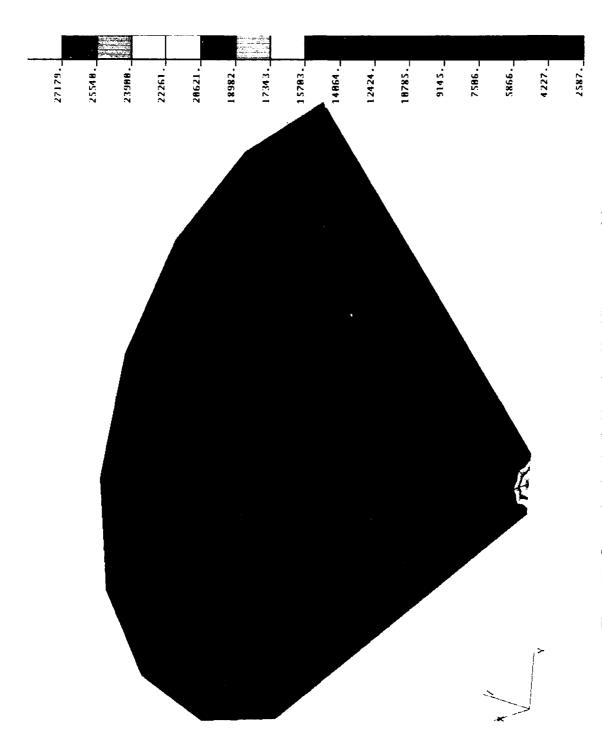
Figure 24. Stress plot showing inside surface Von Mises stresses at t=0.8060 ms (step 2).



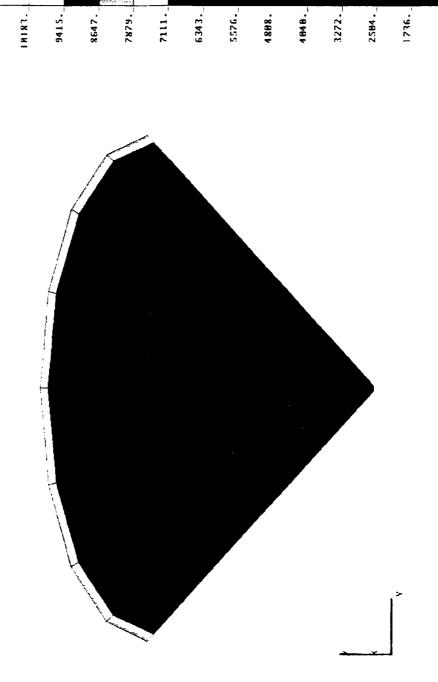
.62127

23600.

Figure 25. Stress plot showing inside surface Von Mises stresses at t = 1.6119 ms (step 3).



**Figure 26.** Stress plot showing inside surface Von Mises stresses of tip area at t = 1.6119 ms (step 3).

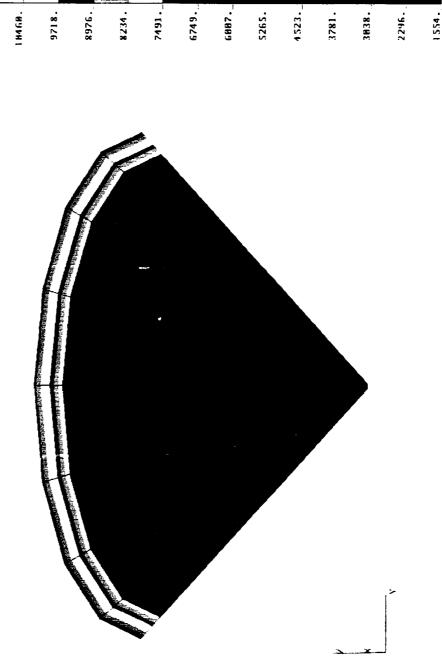


11718.

18951.

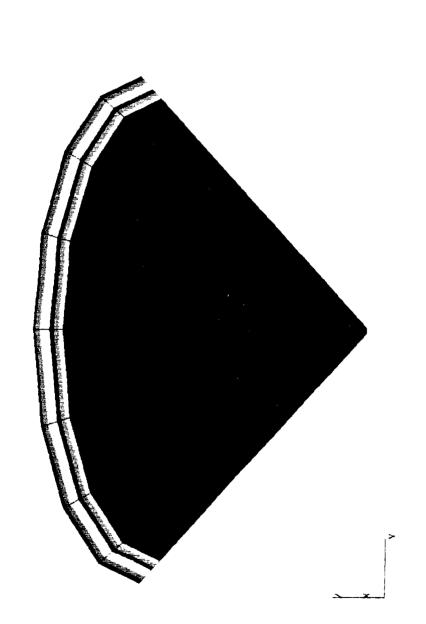
8647.

Figure 27. Stress plot showing inside surface Von Mises stresses at t=2.4179~ms (step 4).



11944.

**Figure 28.** Stress plot showing inside surface Von Mises stresses at t = 3.2238 ms (step 5).



79896

18883.

12879

18862.

19259.

16866.

15669.

14473.

13276.

8489.

7293.

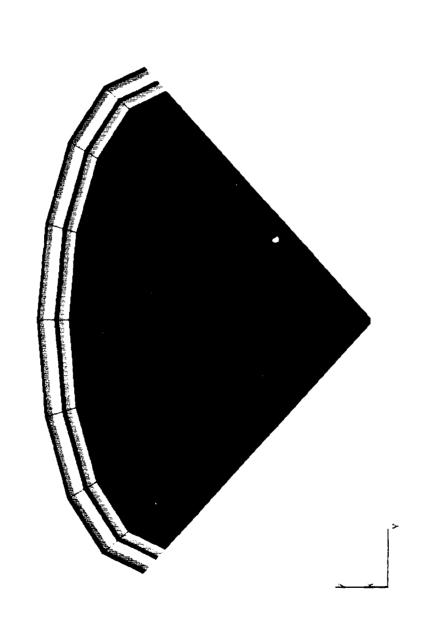
-.9689

4988.

2586.

3783.

Figure 29. Stress plot showing inside surface Von Mises stresses at t = 4.0298 ms (step 6).



14146.\_

12346.\_

18546.

6946.\_

8746.

5146.\_

3346.

1.546.

17746.

19546.

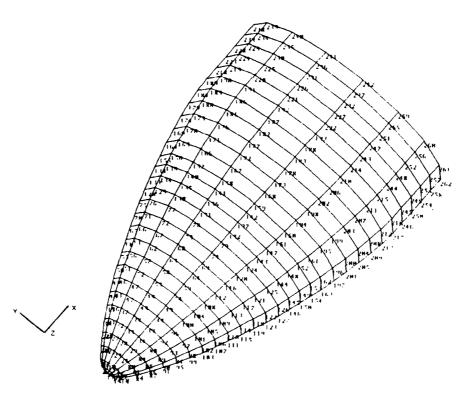
26746.

28546.

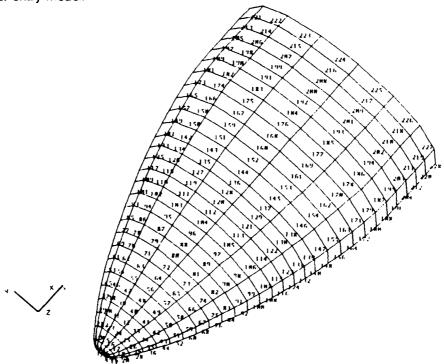
24946.

23146.

**Figure 30.** Stress plot showing inside surface Von Mises stresses at t = 4.8357 ms (step 7).



**Figure 31.** Nodal point pattern of half-symmetric 1.25-cal. Von Karman ogival nosecap water-entry model.



**Figure 32.** Element-numbering pattern of half-symmetric 1.25-cal. Von Karman ogival nosecap water-entry model.

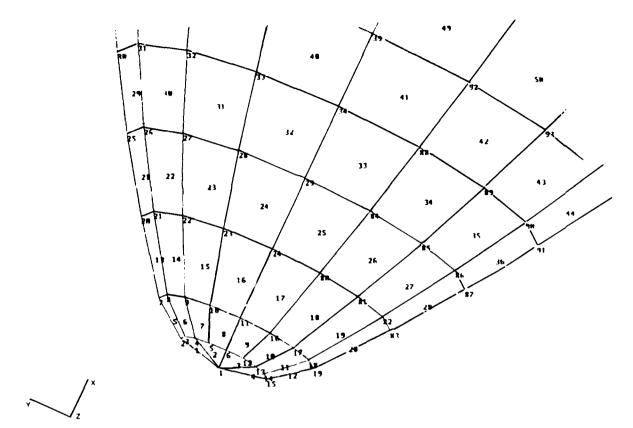


Figure 33. Enlarged view of the tip area of ogive model.

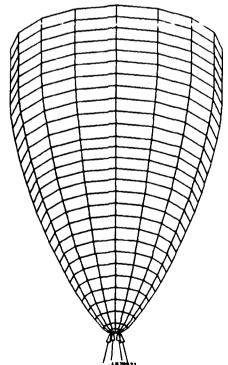


Figure 34. Vertical water-entry nosecap impacting water at 130 ft/s, pressure distribution at t = 0.0445 ms (step 1, initial wetting).

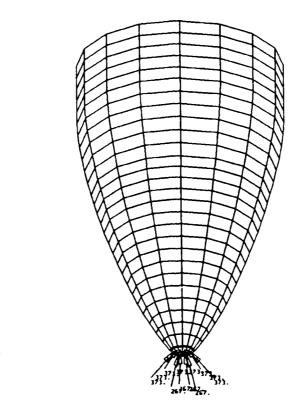


Figure 35. Pressure distribution on vertical water-entry nosecap at t = 0.1770 ms (step 2).

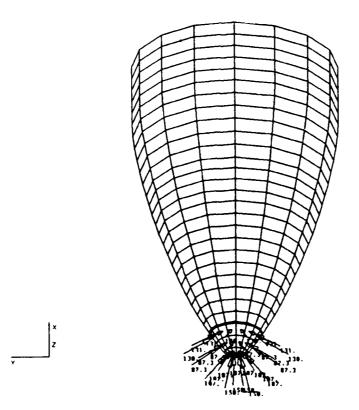


Figure 36. Pressure distribution on vertical water-entry nosecap at t = 0.8440 ms (step 4).

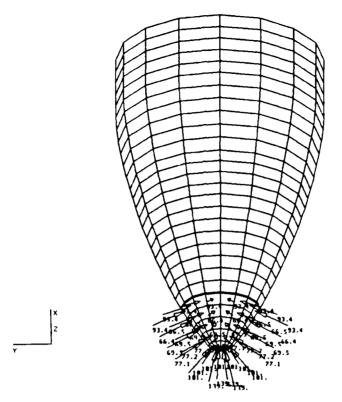


Figure 37. Pressure distribution on vertical water-entry nosecap at t = 1.5624 ms (step 6).

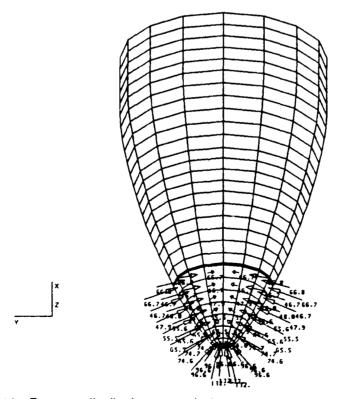


Figure 38. Pressure distribution on vertical water-entry nosecap at t = 2.3226 ms (step 8).

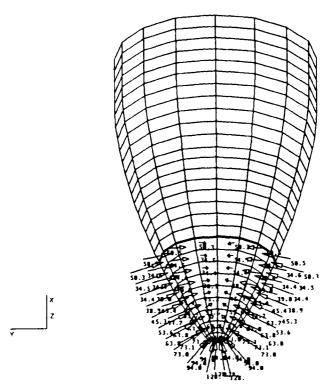


Figure 39. Pressure distribution on vertical water-entry nosecap at t = 3.1128 ms (step 10).

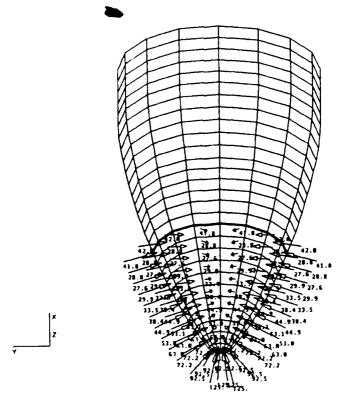


Figure 40. Pressure distribution on vertical water-entry nosecap at t = 3.9200 ms (step 12).

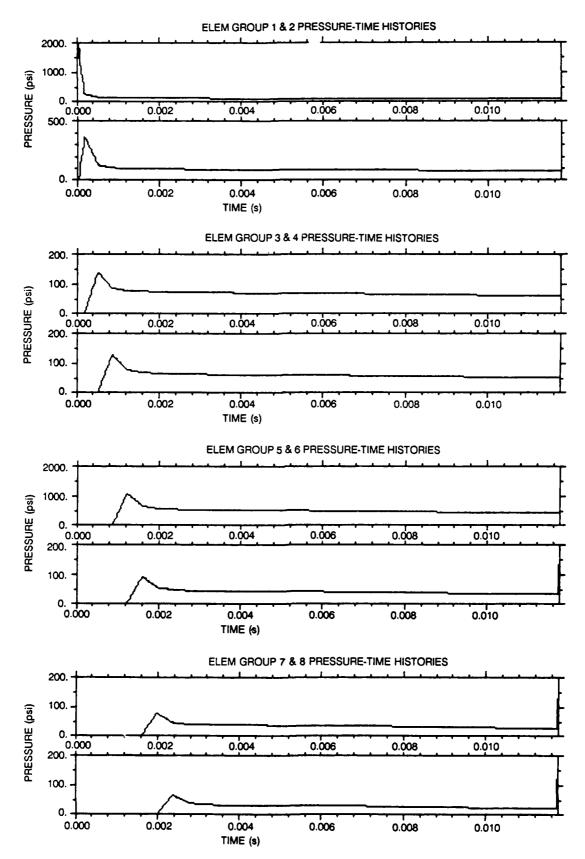


Figure 41. Pressure-time histories (0.000-0.010 s).

To increase computational efficiency, it is possible to take advantage of symmetry in the FEM generated in bullet 1. Figure 42 is a nodal plot of a quarter-symmetric FEM of the nosecap. Notice that this FEM includes two "blade stiffeners" inside the nosecap, which are also composed of four-noded quadrilateral and three-noded shell elements, none of which are wetted during the water entry process. By re-entering PATRAN, it is possible to modify the figures 31 and 32 FEMs to reduce the nodal and elemental count, and add the internal stiffeners, all without compromising the pressure distribution information generated to date. As this is a nongeneric detail of the FEM process, it is not specifically shown in figure 1, but can be accommodated by the experienced user in the WEST process. In addition, the elemental resolution in the circumferential direction has been increased from four to eight elements per 90-degree segment. As reversed bending is expected in the unsupported regions in the center of the nosecap, this increased resolution will improve the computational accuracy of the FEA.

The figure 42 FEM, after being transferred to neutral file format in bullet 2, translated into ABAQUS input format in bullet 26, and after being appended with the pressure-distribution information generated in bullet 24, was used to produce the ABAQUS input file CAP90.INP;10. This input deck is included as appendix F of this report.

Execution of ABAQUS in bullet 25 produces the output files CAP90.DAT, CAP90.FIL, CAP90.LIS, etc. The post-processing file CAP90.FIL is operated on by the results translator, ABAPAT, in bullet 30 to produce the hardcopy deformation plots shown in figures 43 through 52, from 0.04 ms to 6.26 ms after water entry.

Obviously, after the initial wetting shown in figure 43, negligible deformation has taken place, despite the extreme pressure on the tip of the nosecap (see figure 34).

The progression of deformation in the nosecap shell with respect to time during water entry is shown in figures 43 through 52. Notice that a "hump" develops just upstream from the two blade stiffeners, which is first visible in figure 46 at 1.56 ms after water entry. In addition, a concavity in the shell forms between the stiffeners, first visible in figure 47 at 2.32 ms after entry. These deformations become progressively more severe the further the nosecap is immersed.

As previously illustrated with the 90-degree cone example, deformation of a structure body is accompanied by associated stress levels within the structure. In this stress analysis process of the nosecap, the nosecap shell is predicted to fail as the stress in the shell material reaches some measure of a failure criterion. In the case of nonmetallic brittle materials in a nonuniform geometrical arrangement such as a nosecap, it is appropriate to use the maximum principal stress theory for a failure criterion. This theory asserts that failure or fracture of a material occurs when the maximum principal stress at any point in the structure reaches a critical value regardless of the other stresses. The critical value of stress is usually determined in a tensile experiment, where the failure of a specimen is defined to be either excessively large elongation or fracture. Failure is characterized by the separation, or the cleavage, fracture. This mechanism of failure differs drastically from a ductile fracture typical of metallic materials, which is accompanied by large deformations due to slips along the planes of maximum shearing stress.

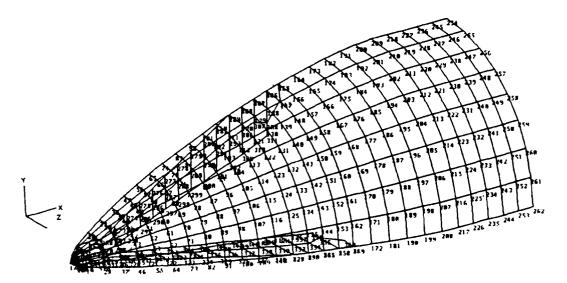
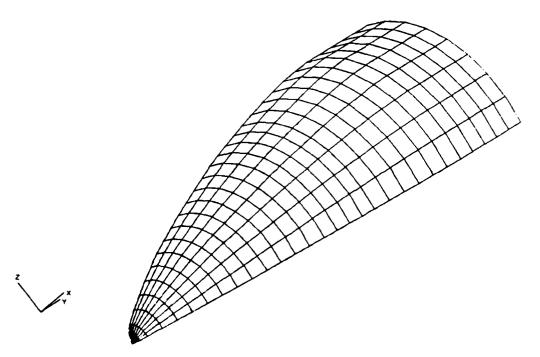


Figure 42. Nodal point pattern of a nosecap quarter-symmetric model with internal blade supports.



**Figure 43.** Progression of deformation of the nosecap shell, at t = 0.0445 ms (step 1, initial wetting).

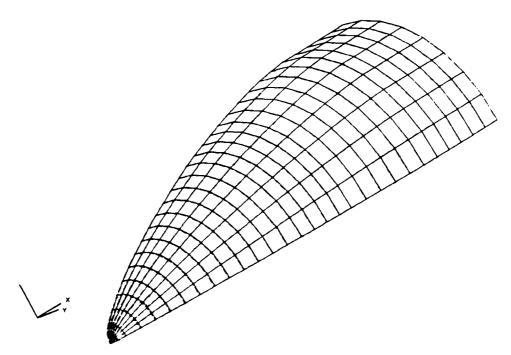


Figure 44. Deformed shape of shell at t = 0.1770 ms (step 2).

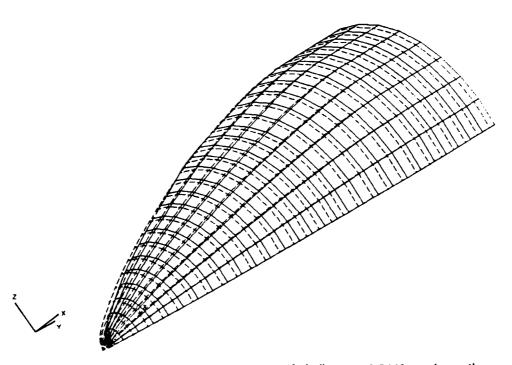


Figure 45. Deformed shape of shell at t = 0.8440 ms (step 4).

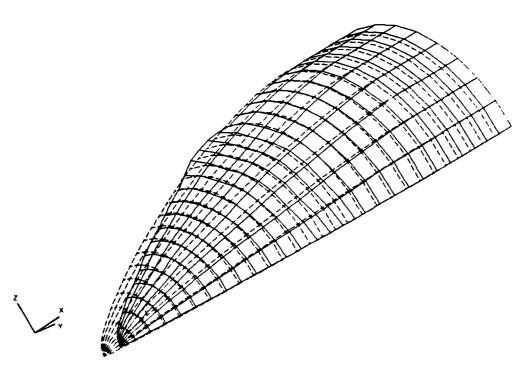


Figure 46. Deformed shape of shell at t = 1.5624 ms (step 6).

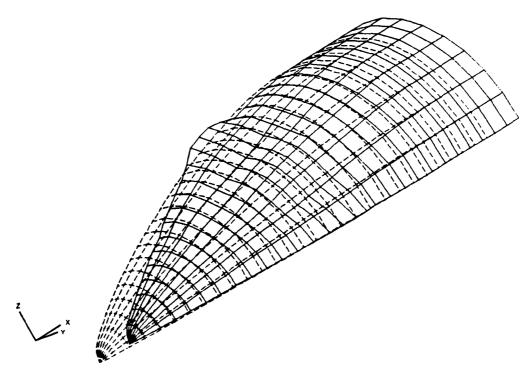


Figure 47. Deformed shape of shell at t = 2.3226 ms (step 8).

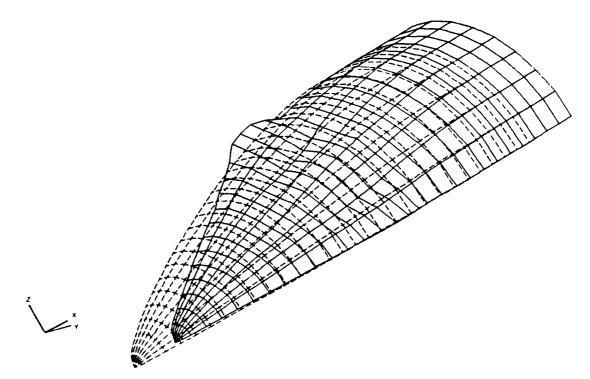


Figure 48. Deformed shape of shell at t = 3.1128 ms (step 10).

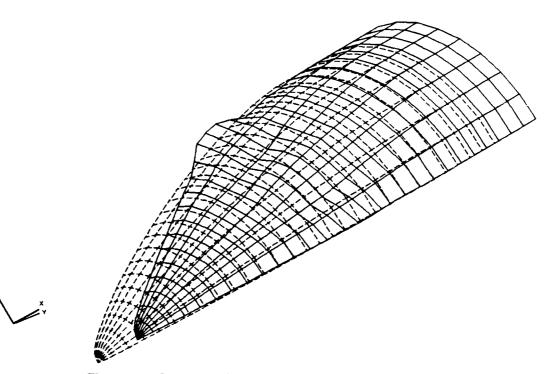


Figure 49. Deformed shape of shell at t = 3.9200 ms (step 12).

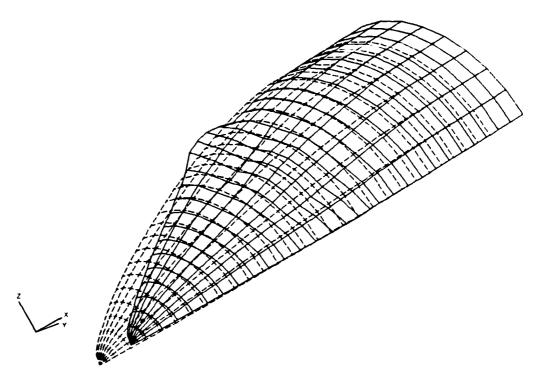


Figure 50. Deformed shape of shell at t = 4.7335 ms (step 14).

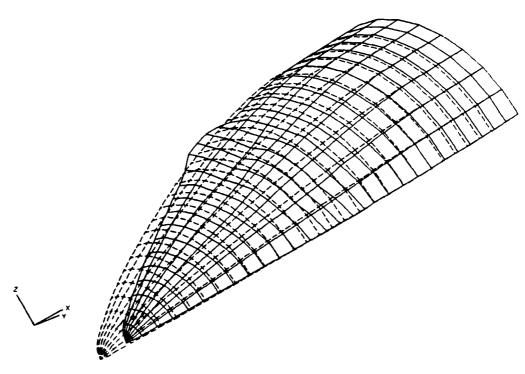


Figure 51. Deformed shape of shell at t = 5.5041 ms (step 16).

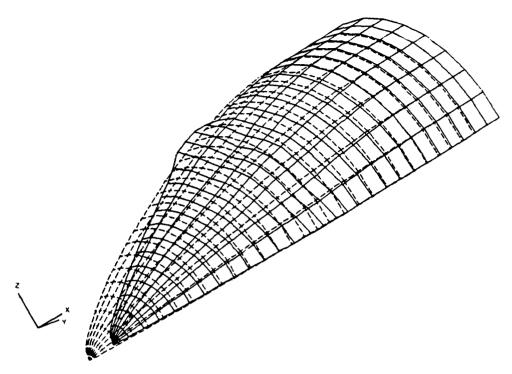


Figure 52. Deformed shape of shell at t = 6.2642 ms (step 18).

The ultimate tensile strength (UTS) of the fiber-reinforced PHENOLIC material for the VLA nosecap has an average UTS in the transverse (hoop) direction of the nosecap of  $5.68\times10^3$  psi (5.68 KSI) and a UTS of 7.57 KSI in the longitudinal (axial) direction of the nosecap. Therefore, failure of the nosecap can be predicted any time the value of any of the three principal stresses exceed these values. Note that, as this is a thin shell, the through-the-thickness stress gradient is virtually zero, and will not be reported.

Figures 53 through 62 are stress contour plots of the inner fiber first principal stress of the nosecap, from initial entry to 6.26 ms after entry. Nodal values are shown in the color bar to the right of the plots, which are averaged by PATRAN throughout the elements to produce the color contours shown. To reduce repetition, only five time-steps are reported.

Figure 53, at 0.04 ms after entry, shows the maximum first principal stress occurring in the second and third rings of elements from the tip. As the nosecap is barely wetted at this time-step (see figure 35), the deformation has not had a chance to propagate to any great extent, and the stress level has only risen to -4303 psi.

Figure 54 indicates that the maximum stress has risen to -4565 psi in several areas along the midspan between the tip and the stiffener interface. The concavity in the shell along the center meridian has started to develop significant stresses of its own.

Figure 55, at 3.11 ms after entry, indicates that the stress level in the midspan area along the edges of the FEM have reached a stress level of -11,648 psi, while the center meridional stress level has increased to approximately -5400 psi.

In figure 56, at 4.73 ms after entry, the stress in the midspan along the edges has fallen to -11,327 psi, with a corresponding decrease to -5286 psi along the center meridian. The point of maximum stress has moved up the nosecap in the approximate location of the water surface.

Figure 57, at 6.26 ms after entry, indicates that the midspan stress has continued to fall, along with the meridional stress. As previously shown in figure 41, the magnitude of the pressure distribution decreases with time as the nosecap enters the water. As the shell diameter increases with distance from the tip, the stress level would be expected to increase with time, had the pressure distribution been constant with time. However, this is obviously not the case, and the stress level in the shell has been shown to peak at some time after entry, but before the shell is fully wetted.

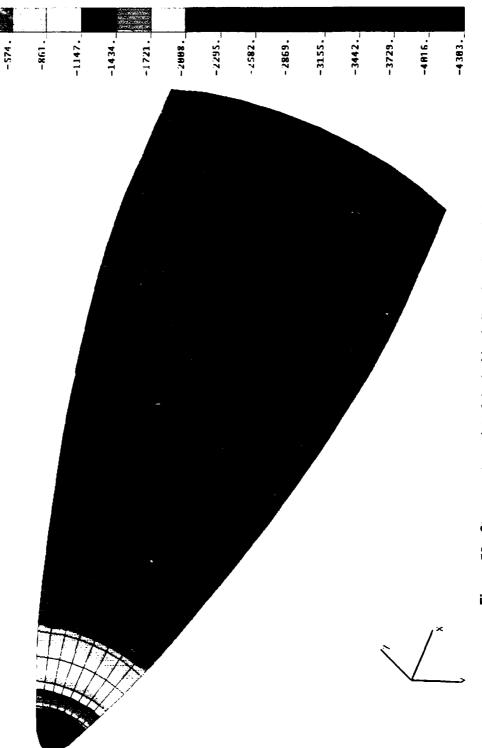
To reduce redundancy, the second principal stress distribution is not reported.

Figures 58 through 62 are stress contour plots of the third principal stresses, from 0.18 to 6.26 ms after water entry. In general, these follow the same trends as reported in figures 53 through 57. However, the third principal stresses are all tensile (positive in magnitude) stresses, and are greatly influenced by the reverse bending that occurs in the equatorial direction around the shell.

A cross-sectional view of the nosecap along the axial meridians reveals some bending behavior occurring near the "hump" in the shell. However, the predominant deformation is compressive, yielding predominantly compressive normal stresses in the axial direction. A cross-sectional view of the nosecap along the hoop direction equators reveals considerable reverse bending response in the hoop direction. It is this bending response that dominates the stress field used to compute the third principal stresses.

As the direction cosines of the normal stresses are not reported by ABAQUS, it is extremely difficult to describe the directions of the principal stresses. Although care was taken in this analysis to insure that the elemental numbering scheme used resulted in a uniformly "outboard normal" for each element, ABAQUS sorts the resultant principal stresses by magnitude. Therefore, it is not possible to state that the first principal stresses are aligned in predominantly axial direction, or that the third principal stresses are predominantly aligned in the hoop direction. In addition, the deformation in response to the pressure-time history is extremely complex, both due to the changing loads, and the presence of the blade stiffeners. The principal stresses are simply compared to UTS in the maximum principal stress theory. If a failure is indicated in one of the three principal directions, it is not important to know in what direction that stress lies.

Figures 53 through 62 show that a dynamic state of stress exists within the nosecap during water entry. To attempt to simplify the large amount of data collected, figures 63 through 66 were condensed from the data. These are plots of the state of stress as a function of time for four points on the nosecap shell, aligned with the 90-degree meridian and the blade stiffener. Node 28 is located near the tip of the shell, node 73 in the midspan between the tip and the blade-shell interface, node 118 near the lower edge of the blade stiffener, and node 136 near the middle of the blade-shell interface (see figure 42).



-287.

-.8668

**Figure 53.** Stress contour plot of the inside shell surface first principal stresses at t = 0.0445 ms (step 1).



Figure 54. Inside shell surface first principal stresses at t = 1.5624 ms (step 6).



-777.

Figure 55. Inside shell surface first priftipal stresses at t = 3.1128 ms (step 10).



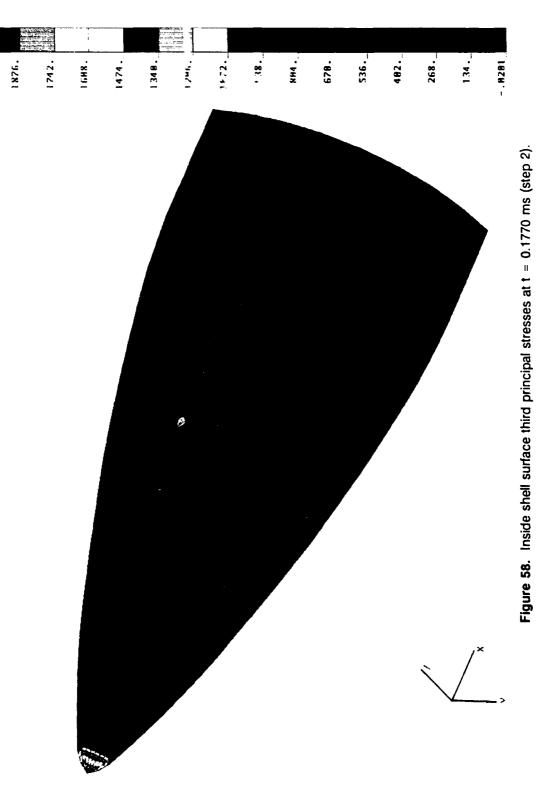
-755.

Figure 56. Inside shell surface first principal stresses at t = 4.7335 ms (step 14).



-527.

Figure 57. Inside shell surface first principal stresses at t = 6.2642 ms (step 18).



63

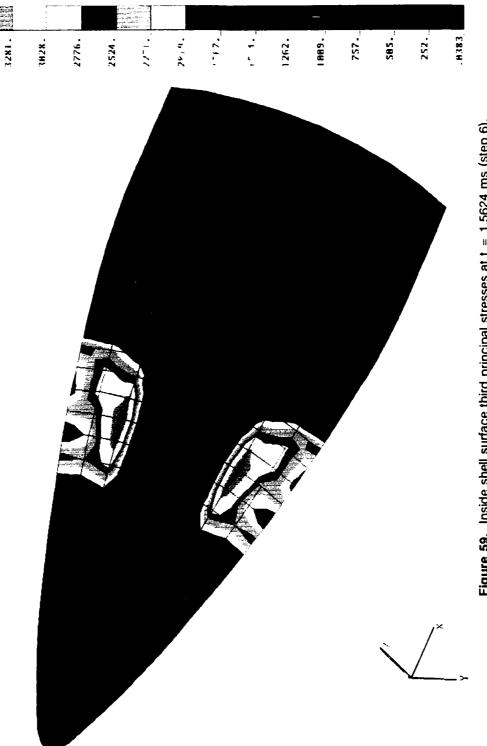


Figure 59. Inside shell surface third principal stresses at t = 1.5624 ms (step 6).

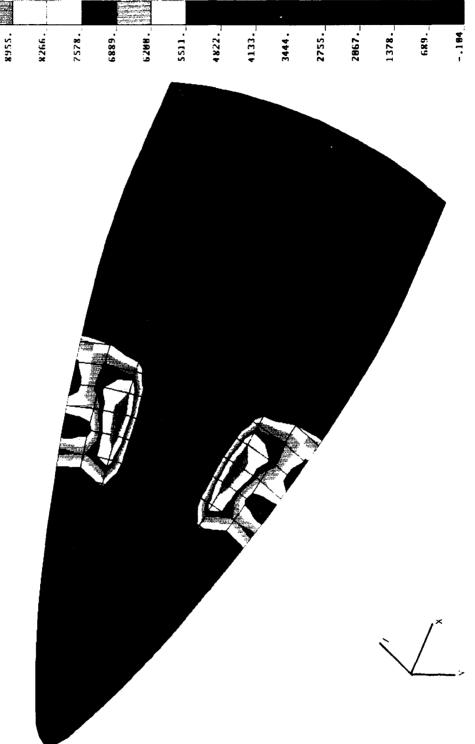


Figure 60. Inside shell surface third principal stresses at t = 3.1128 ms (step 10).

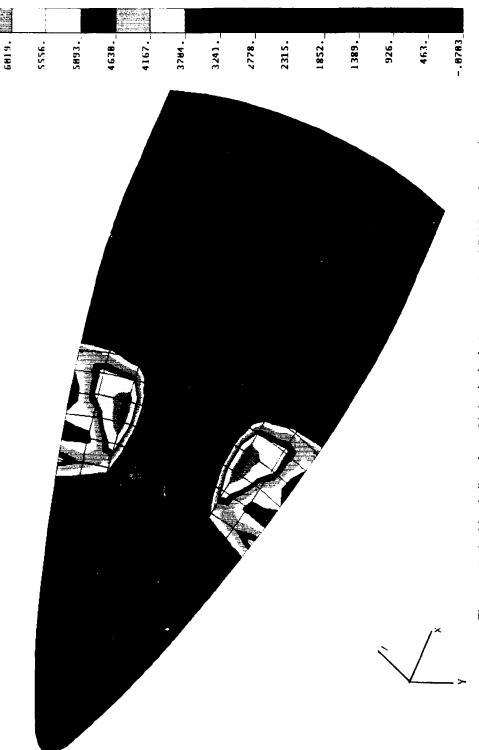
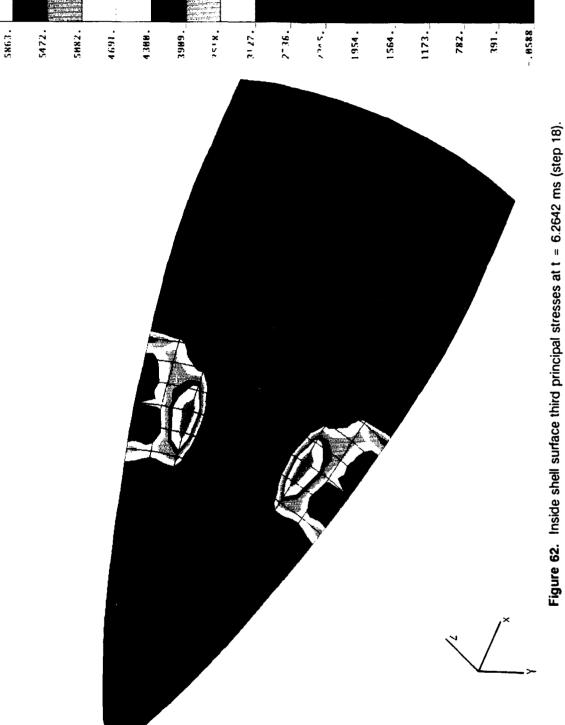


Figure 61. Inside shell surface third principal stresses at t = 4.7335 ms (step 14).



These stress-time histories indicate an obvious periodicity of the data. This could be indicative of the period of the moving stress wave within the shell, or the lowest order response of the shell. It is also likely that there are considerable higher frequency stress waves propagating through the structure that either the nodal spacing in the FEM was unable to resolve, or the time steps used in the dynamic analysis were too large. In any case, there is no need to lend any large degree of engineering significance to this periodicity.

Figures 63 and 64 are the plots of the first principal stresses at nodes 28, 73, 118, and 136 on the inner and outer fibers respectively. Note that the inner fiber stress reaches a maximum value of -12,560 psi at 4.32 ms after water entry, while the outer fiber first principal stress for the same node has a value of 0 psi at the same time. This apparent anomaly is due to the method that ABAQUS uses to sort principal stresses. Although not shown, it is suspected that the second principal stress at this time period and nodal location has a nonzero stress value of some negative value, which has a great deal more engineering significance than the zero value shown at node 118 at this time-step.

Figures 65 and 66 show the third principal stresses for the same nodes on the inner and outer surfaces respectively. In figure 65, it is seen that the third principal stress at both nodes 118 and 136 exceed the value of UTS shortly after 3.0 ms after water entry, reaching a peak value of 7312 psi.

At the same time, the third principal stress on the outer surface in figure 66 reaches a maximum value of 21,240 psi at node 136, while the value at node 118 rises to 12,650 psi. Each of these values greatly exceed the UTS of 5680 psi, shown as a horizontal line in figures 65 and 66.

Figure 67 is a maximum envelope of the stress-time history of the entire nosecap. This indicates the maximum value of the second principal stress (17,856 psi), and the maximum value of the third principal stress (21,129 psi) occur simultaneously at 3.11 ms after water entry. The corresponding minimum values are shown in figure 68.

The UTS of the material is 5.68 KSI (Dimelfi, July 1988), while the ultimate compressive strength (UCS) is 30.16 KSI (Fogg, October 1989). Therefore, the minimum margin of safety of the nosecap during water entry occurs in tension at 3.11 ms after entry, and has a value of:

MS = (UTS/Max Principal Stress)-1 = (5680/21,129)-1 = -0.73

A dynamic stress analysis of this nosecap structure is a very complicated proposition. Not only are the stress levels affected by the blade-shell interface, but the presence of a dynamic load, with associated moving stress waves and the possibility of high-frequency shell responses, makes interpretation of these stress contours very difficult indeed. As this nosecap example was included to illustrate the WEST process, little attempt was made to exactly model the nosecap structure. Thus, while these results are considerably more accurate than those reported by Jung and Plapp (1988), they should only be used as an indicator of the ability of the WEST to aid the analytical process used to design this type of frangible nosecap.

## CONCLUSIONS

The objective of this IED effort has been met. The WEST has been validated through two separate examples of nosecaps, a 90-degree cone and an engineering example of a typical VLA nosecap. While both these examples described undergoing vertical entry, the WEST is equally facile at oblique entry. Note that the use of symmetrical FEM must be adjusted accordingly, as an oblique entry results in nonsymmetric loading. In addition, a greatly increased amount of data results from the analysis, as each element has a unique pressure-time history throughout the water entry process.

Development of the WEST has led to a many-fold reduction in the time required to execute a meaningful design iteration of a frangible nosecap. In addition, due to the ability to use both geometric and material nonlinearities, the potential accuracy of the solution has greatly improved as well. Note that the analytical techniques described in this report did not take advantage of these features.

While investigation of an alternate frangible nosecap for the VLA project is certainly not complete, the results from the water entry portion of that task indicate that breakup at water entry appears likely.

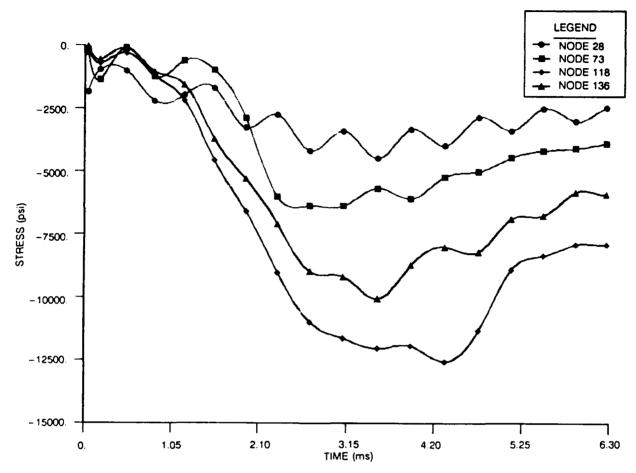


Figure 63. VLA nosecap first principal stresses of the inner surface.

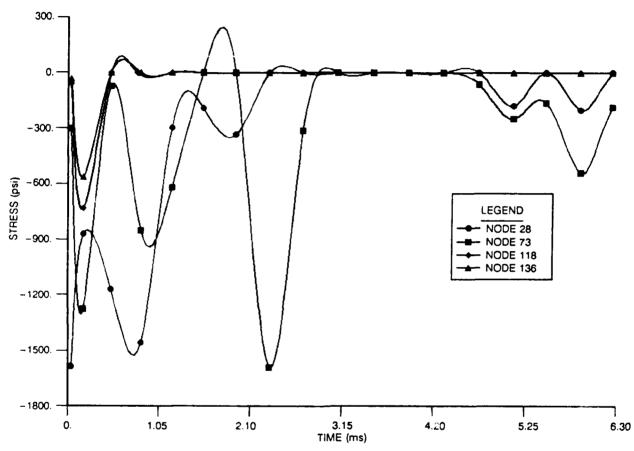


Figure 64. VLA nosecap first principal stresses of the outer surface.

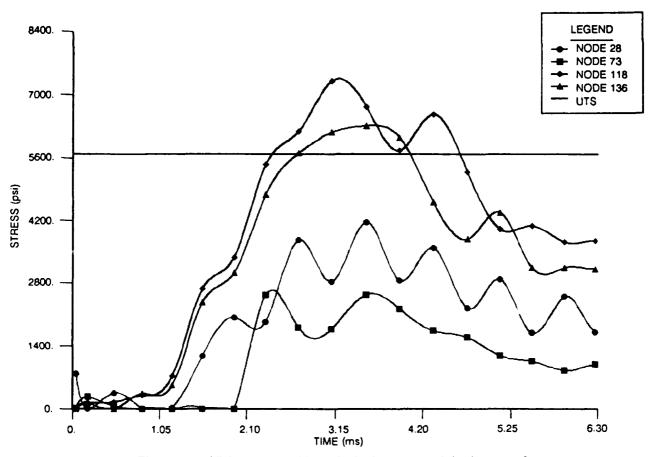


Figure 65. VLA nosecap third principal stresses of the inner surface.

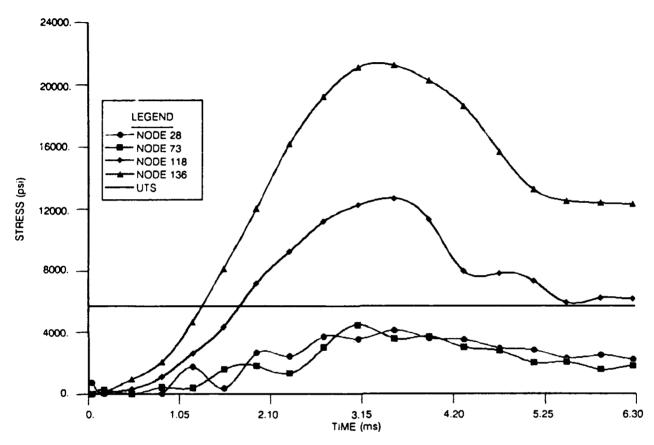
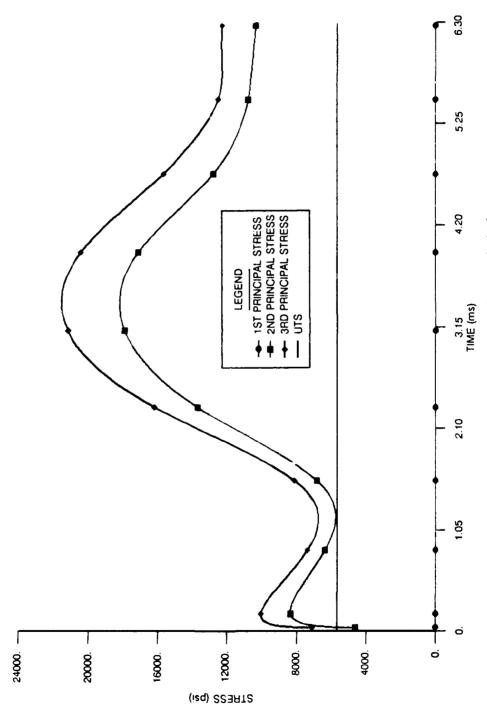


Figure 66. VLA nosecap third principal stresses of the outer surface.



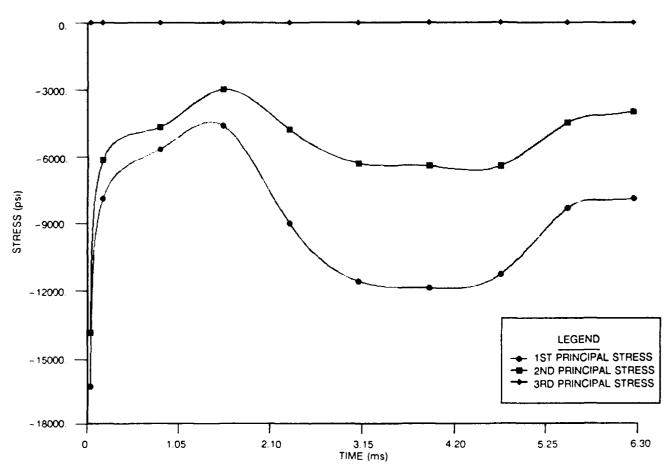


Figure 68. VLA nosecap minimum principal stresses.

## RECOMMENDATIONS

The code linkage process described in this report is only suitable for a scientific engineering application within a Naval laboratory such as NOSC. It should not be considered a "production" code suitable for commercial application, as there are several "rough edges" within the execution of WEST. In particular, care must be taken during the development of the initial geometry and FEM within the preprocessor portion of WEST, to acknowledge the limitations of the water entry portion of the code. While arbitrary shapes can be accommodated in the water entry process, there are limitations pertaining to mesh density, spacing, and numbering that preclude executing any old FEM that happens to describe a particular geometry of a nosecap.

In addition, it is assumed that the user has a background in using the PATRAN pre- and post-processor. This IED project does not attempt to reduce or foreshorten the complex task of translating nosecap drawings into "PATRANese". A FEA background is essential to create viable FEMs that not only will execute throughout WEST, but will truly produce "right" answers at the conclusion of the process. As with any FEA technique, it is often as much art as science, and not something that can be learned overnight by reading this report.

Improvements to the analytical procedure described for the VLA nosecap can be made in the following areas:

- (a) ABAPAT translator revision. The PDA Engineering Release 3.0 ABAPAT translator has numerous "bugs" in it. In particular, while the outer fiber normal stresses are computed by ABAQUS and are contained in the FILENAME.FIL post-processing file, ABAPAT cannot translate all of them into the FILENAME.STPiIj.NOD files needed for complete investigation of the results. Therefore, they are not included in this report. According to PDA Engineering, this will be fixed in Release 4.0, which is due for release within 90 days.
- (b) Nonlinear material model. ABAQUS is equipped to accommodate nonlinear materials models. As the plastic materials typically used in nosecaps exhibit considerably nonlinear stress-strain behavior, increased accuracy would result by taking advantage of this feature. For the work accomplished for this IED project, linear-elastic materials properties were used.
- (c) Nonlinear geometry model. In addition to material nonlinearities, ABAQUS can accommodate geometric non-linearities. These should be used in the area of the fin-shell interface of the VLA nosecap, to improve the accuracy of the relative deformations of the structural elements.
- (d) Mesh rezoning. As the nosecap deforms, the geometry presented to the water surface during water entry changes. Yet, in the present form of the WEST process, the pressure-time histories are considered to be developed by a constant, invariant geometry. With significant deformation, this can introduce significant error in the process. A worthwhile improvement would be to halt the deformation process at some point in the analysis

loop, remesh the deformed geometry, use this new meshed geometry as input to the ENTRY code, re-apply the revised pressure-time histories to the deformed geometry, and re-start the analysis. This iteration could be performed several times if necessary, until the analyst was satisfied that accuracy had been achieved.

Investigation of the VLA nosecap shape during the development of this WEST revealed several areas for improvement in the analytical process used to examine the nosecap structure. Because of the negative margin of safety at water entry, additional investigation into the design optimization of this nosecap should be pursued, followed by fabrication and test of sample one-piece nosecaps. The potential for success is there, and the rewards for major simplifications to the nosecap design are many.

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## **GLOSSARY**

ABAQUS An FEA computer code that is a registered trademark of Hibbitt,

Karlson, & Sorenson, Inc.

ASROC Antisubmarine rocket

ABAPAT A PDA Engineering translator

CASA/GIFTS An FEA computer code that is a trademark of Computer-Aided

Structural Analysis, Inc.

CDC Control Data Corporation

 $C_p$  Pressure coefficient

 $\nabla \phi$  The gradient operator

 $C_{\mathbf{w}}$  Wetting factor

 $\Delta h$  Increment in effective depth between successive steps

DTRC David Taylor Research Center

DYCAST/GC An FEA computer code that is a registered trademark of the

Grumman Corporate Research Center

DYNA-3D An FEA computer code

ENTPRES A NOSC-developed translator

ENTRY A computer code for the prediction of impact loads of an arbitrary

body during water entry

 $\phi_{cn}$  The value of  $\phi$  at a specific element centroid at step n

FEA Finite element analysis

FEM Finite element model

FLIPPER A NOSC Code 936-owned VAX 11/785 computer

FORTRAN A computer language

GCB General Communications Backbone

GPCC General Purpose Computer Center

IED Independent Exploratory Development (Program)

KSI 1000 pounds per square inch

φ A velocity potential function

MARC An FEA computer code that is a trademark of MARC Analysis

Research Corporation

ms Millisecond

MSC/NASTRAN An FEA computer code that is a registered trademark of the

MacNeal/Schwendler Corporation

MSC/PAL-II An FEA code that is a registered trademark of MacNeal/

**Schwendler Corporation** 

MCP A VAX command to download a VAX file to PC, or vice versa.

MS Margin of safety

NESS Naval Engineering Software Support (Office)

NISAII-PC An FEA computer code that is a registered trademark of

**Engineering Mechanics Research Corporation** 

NOSC Naval Ocean Systems Center

NSWC/WOL Naval Surface Weapons Center, White Oak Laboratory

PC Personal computer

PATRAN A registered trademark of PDA Engineering, Inc.

PHOENICS An FEA computer code that is a registered trademark of CHAM

of North America, Inc.

PISCES-3DE An FEA computer code that is a trademark of Physics

**International Company** 

PATABA Another PDA Engineering translator

PATENTR A NOSC-developed translator

psi Pounds per square inch

STINGRAY A GPCC-owned convex mini-supercomputer

UCS Ultimate compressive strength

UTS Ultimate tensile strength

VLA Vertical launch ASROC

WEST Water Entry Structure Technique

YADAP A PC computer code for display and plotting of time-histories

APPENDIX A
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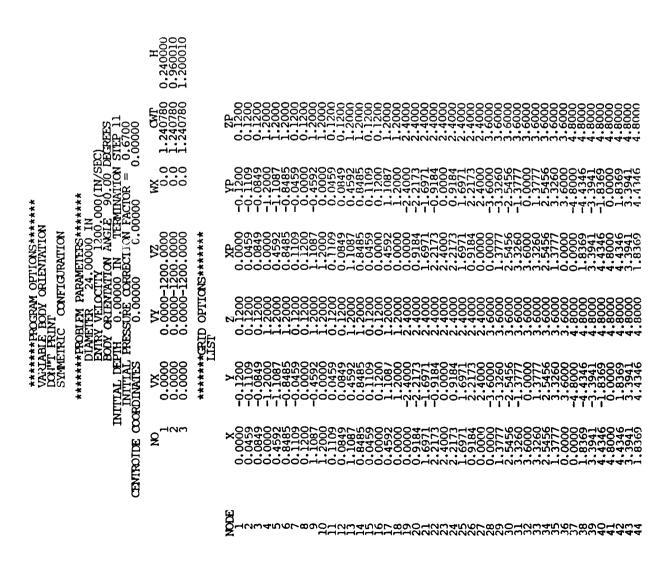
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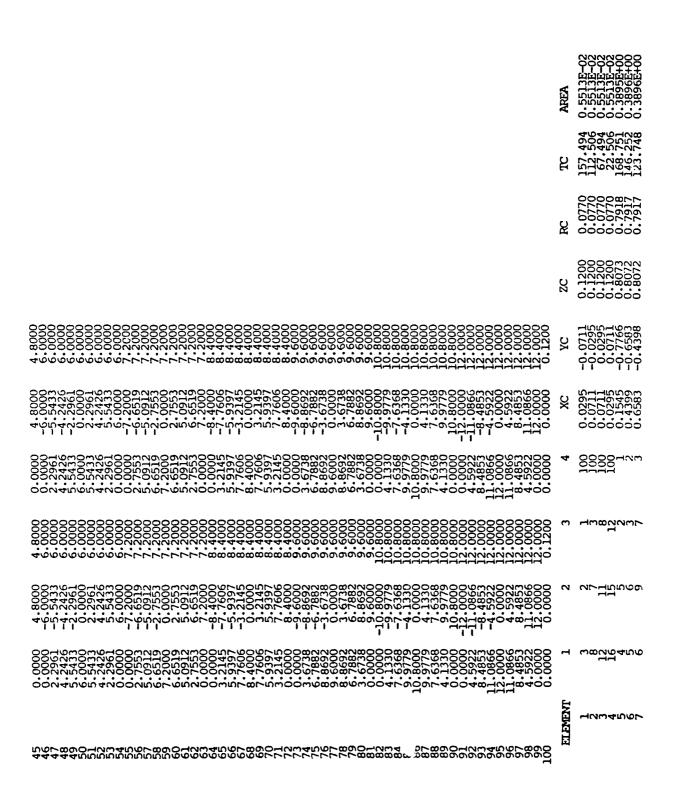
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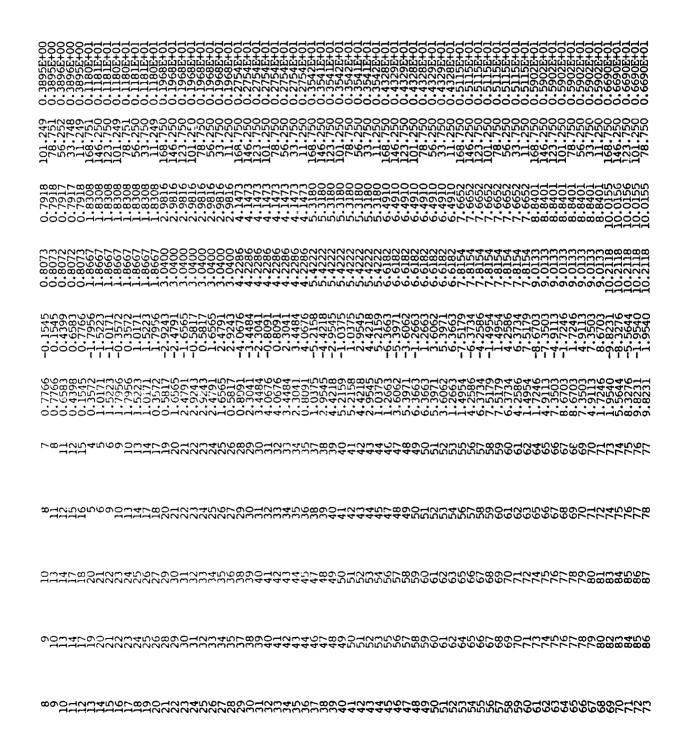
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96	97	88	87
97	98	89	88
98	99	90	89

APPENDIX B
ENTRY INPUT FILE
CONE90.OUT



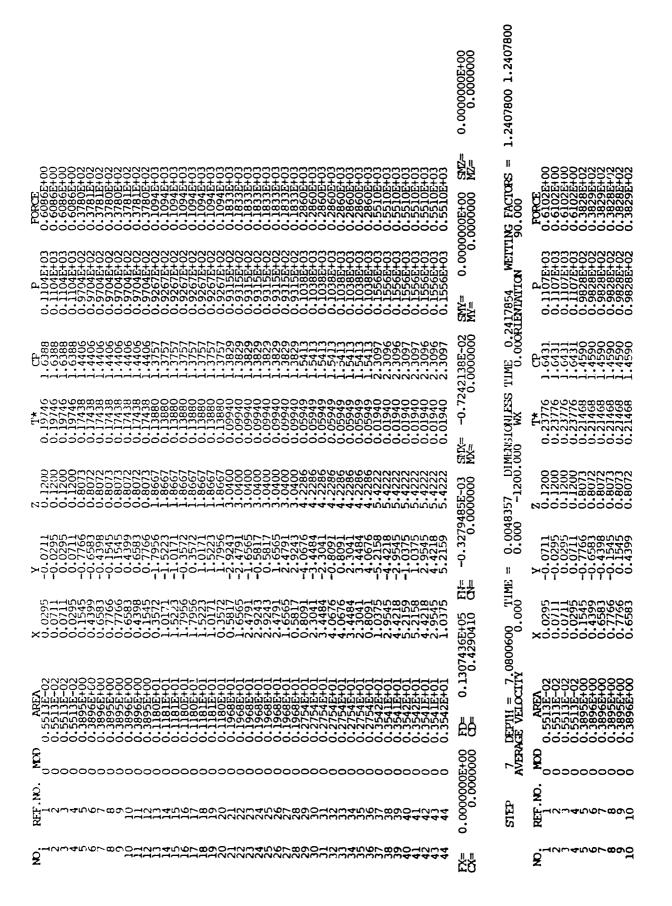


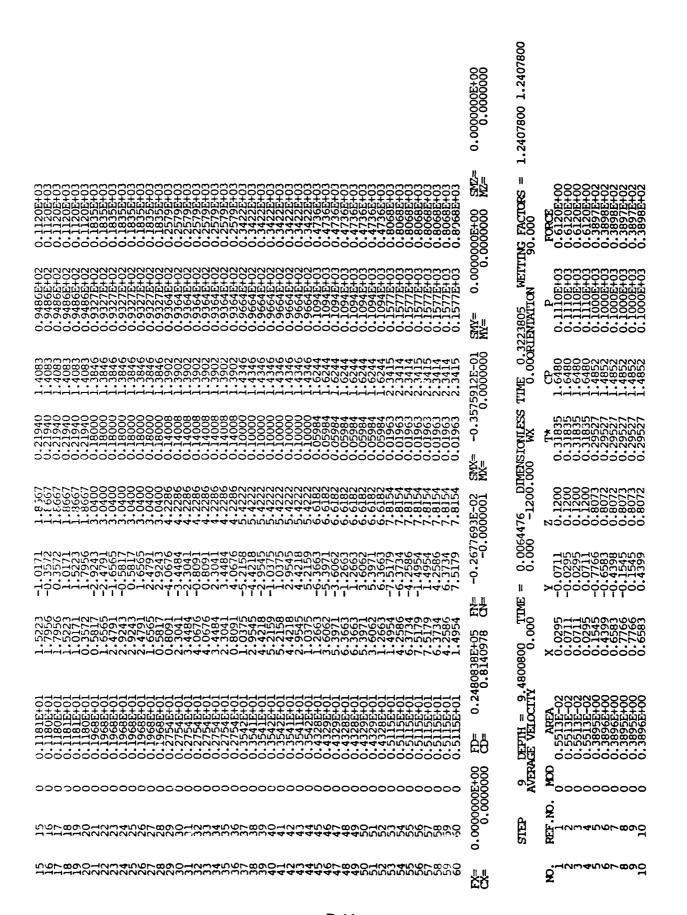


0.6690E+01 0.6690E+01 0.7477E+01 0.7476E+01 0.7476E+01 0.7477E+01 0.7477E+01 0.7477E+01													1.2407800 1.2407800		0,000000E+C0 0,00000C0	1.2407800 1.2407800
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477978999999999999999999999999999999999	CP TIME FOR	AVERAGE VELOCITY	00000000000000000000000000000000000000	88 80 113	2 DEFIH = 1.0800100 AVERACE VELOCITY 0.00 MOD 3513E-02 0.029 0 0.5513E-02 0.071											
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1180E+01 1181E+01 1181E+01 1180E+01 1968E+01 196	E+00 FD= 0.4889904E+04 FN= 0000 CD= 0.1604644 CN= 5 DEPIH = 4.6800400 TIME AVERAGE VELOCITY	00001420/20444000/2000000400440000000000	5400 ED= 0.8527767E4-04 FN= 0.000 ED= 0.2798424 CN= 0.2798424 CN=
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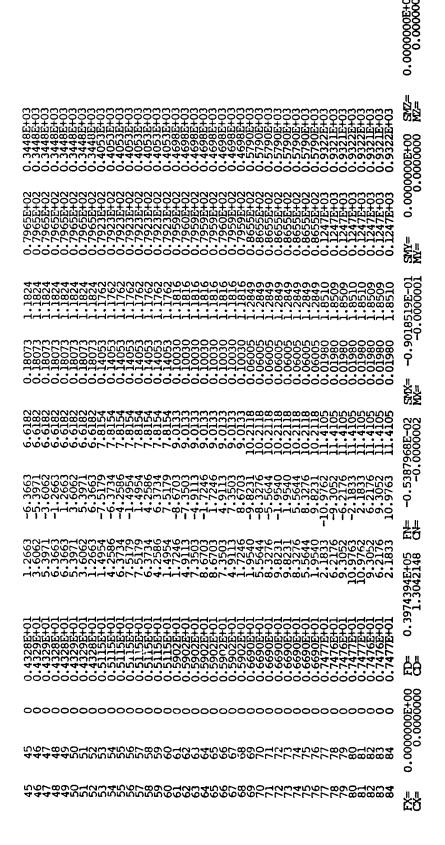


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0.25527 0.255970 0.25597	SVX= 0.8532471E-01 MX= 0.0000001
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0.000000000000000000000000000000000000	щО
11111111111111111111111111111111111111	0.0000000E+00 0.0000000
	### ###

10 DEPTH = 10.6800900 TIME = 0.0072536 DIMENSIONLESS TIME 0.3626781 WEITING FACTORS = 1.2407800 1.2407800 AVERAGE VELOCITY 0.000 -1200.000 WX 0.000RIENTATION 90.000

STEP

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11111222222222222222222222222222222222	365194E-01 0.0000000	TIME 0	
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## APPENDIX C SAMPLE ABAQUS INPUT FILE CON90.IN

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*HEADLNG
50-D BIUNTED FIBERITE CONE (12-IN LONG, 0.1-IN THICK)
** NEUTRAL FILE GENERATED ON: 24-AUG-89 17:20:26 PAT
                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                                       PATABA VERSION: 3.0
                                                                                                                                                                                                    0.000000000E+00, -0.119999997E+00, 0.459220074E-01, -0.848528072E-01, 0.000000000E+00, -0.12000007E+01, 0.459220171E+00, -0.110865557E+01, 0.848528594E+00, -0.848528266E+00, 0.110865556E+00, -0.459220074E-01, 0.119999990E+00, 0.000000000E+00, 0.119999990E+00, 0.000000000E+00, 0.1108655569E+01, -0.459220074E-01, 0.110865557E+01, 0.459220111E-01, 0.848528266E+00, 0.84852826E+00, 0.84852826E+00, 0.84852826E+00, 0.84852826E+00, 0.84852826E+00, 0.459220111E-01, 0.0000000000E+00, 0.110865557E+01, 0.459220141E+00, 0.459220141E+00, 0.459220141E+00, 0.110865548E+00, 0.459220141E+00, 0.110865548E+00, 0.000000000E+00, 0.110865569E+01, 0.0000000000E+00, 0.110865569E+01, 0.0000000000E+00, 0.120000017E+01, 0.000000000E+00, 0.120000017E+01, 0.21731191E+01, 0.221731191E+01, 0.918440163E+00, 0.221731091E+01, 0.332596636E+01, 0.332596636E+01, 0.332596636E+01, 0.332596636E+01, 0.332596636E+01, 0.332596636E+01, 0.332596636E+01, 0.33259660E+01, 0.254558442E+01, 0.33259660E+01, 0.254558442E+01, 0.33259660E+01, 0.254558442E+01, 0.33259660E+01, 0.33259660E+01, 0.254558442E+01, 0.33259660E+01, 0.254558442E+01, 0.33259660E+01, 0.254558442E+01, 0.33259660E+01, 0.254558442E+01, 0.33259660E+01, 0.229610062E+01, 0.229610062E+01, 0.229610062E+01, 0.229610062E+01, 0.229610062E+01, 0.229610062E+01, 0.229610062E+01, 0.229610
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0.997789850E+01,
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0.720000076E+01,
-0.840000057E+01,
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0.593969774E+01,
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0.840000057E+01,
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0.00000000E+00,
0.367376065E+01,
0.678822565E+01,
0.678822565E+01,
0.960000038E+01,
-0.108000002E+02,
-0.997789955E+01,
-0.413298082E+01,
0.763675308E+01,
0.763675308E+01,
0.763675356E+01,
0.997789955E+01,
0.108000002E+02,
-0.120000000E+02,
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0.10805545E+02,
-0.848528099E+01,
0.459220123E+01,
0.459220123E+01,
0.459220171E+01,
0.10865545E+02,
-0.848528099E+01,
0.108000000E+02,
-0.120000000E+00,
0.459220171E+01,
0.10865545E+02,
0.120000000E+02,
0.120000000E+00,
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0.720000076E+01
0.719999981E+01
0.720000076E+01
0.84000057E+01
0.96000038E+01
0.959999943E+01
0.108000002E+02
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0.108000002E+02
0.108000000E+02
0.120000000E+02
                                                                                                                          98, 0.459220028E+01
99, 0.000000000E+00
100, 0.000000000E+00
*NSET,NSET=SYMIT,GENERATE
19,91,9
*NSET,NSET=SYMRT,GENERATE
18,99,9
*NSET,NSET=SYMALL
1,4,16,100,SYMIT,SYMRT
*NSET,NSET=BASE,GENERATE
91,99,1
*BOUNDARY
SYMALL,XSYMM
BASE,3
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                                                                               ELEMENT DEFINITIONS
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*EIFMENT TYPE=

1, 100,
2, 100,
3, 100,
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78,
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115690147801234567901234568901234578901234678901235678901245678901 

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77, 82, 83, 92,
78, 83, 84, 93,
79, 84, 85, 94,
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81, 86, 87, 96,
82, 87, 88, 97,
83, 88, 89, 98,
*ELSET, ELSET=ELGI, GENERATE
1,4
*EISET_EISET=EIG2,GENERATE
5,12
5,12
5,12
*ELSET, ELSET=ELG3, GENERATE
13,20
*ELSET, ELSET=ELG4, GENERATE
21,28
*EISET,EISET=EIG5,GENERATE
29,36
*ELSET,ELSET=ELG6,GENERATE
 37,44
*ELSET,ELSET=ELG7,GENERATE
45,52
*ELSET-ELG8,GENERATE
53,60
*FISET,EISET=ELG9,GENERATE
 61,68
*ELSET,ELSET=ELG10,GENERATE
 69,76
*EISET,EISET=EIG11,GENERATE
 *SHELL SECTION, ELSET=ALL, MATERIAL=FIBERITE
*MATERIAL, NAME=FIBERITE
*FIASTIC
1.9E+06,0.28,72.
*DENSITY
1.682E-4
      TIME-VARYING PRESSURE LOADS ON Q-SHELL ELEMENTS
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3.224E-03 0.000E+00 4.030E-03 0.000E+00 4.836E-03-1.573E+02 5.642E-03-1.094E+02 6.448E-03-9.832E+01 7.254E-03-9.581E+01 8.059E-03-7.965E+01

*AMPLITUDE, NAME=ELG8, TIME=D, VALUE=A
1.612E-04 0.000E+00 8.060E-04 0.000E+00 1.612E-03 0.000E+00 2.418E-03 0.000E+00 3.224E-03 0.000E+00 4.030E-03 0.000E+00 4.836E-03 0.000E+00 5.642E-03-1.577E+02 6.448E-03-1.116E+02 7.254E-03-9.985E+01 8.059E-03-7.923E+01

*AMPLITUDE, NAME=ELG9, TIME=D, VALUE=A
1.612E-04 0.000E+00 8.060E-04 0.000E+00 1.612E-03 0.000E+00 2.418E-03 0.000E+00 3.224E-03 0.000E+00 4.030E-03 0.000E+00 4.836E-03 0.000E+00 5.642E-03 0.000E+00 6.448E-03-1.572E+02 7.254E-03-1.135E+02 8.059E-03-7.959E+01

*AMPLITUDE, NAME=ELG10, TIME=D, VALUE=A
1.612E-04 0.000E+00 8.060E-04 0.000E+00 1.612E-03 0.000E+00 2.418E-03 0.000E+00 3.224E-03 0.000E+00 4.030E-03 0.000E+00 1.612E-03 0.000E+00 5.642E-03 0.000E+00 6.448E-03 0.000E+00 4.030E-03 0.000E+00 4.836E-03 0.000E+00 5.642E-03 0.000E+00 1.612E-04 0.000E+00 4.030E-03 0.000E+00 1.612E-03 0.000E+00 5.642E-03 0.000E+00 3.224E-03 0.000E+00 4.030E-03 0.000E+00 4.836E-03 0.000E+00 5.642E-03 0.000E+00 6.448E-03 0.000E+00 7.254E-03-1.59E+02 8.059E-03-8.655E+01

*AMPLITUDE, NAME=ELG11, TIME=D, VALUE=A
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        *DYNAMIC
     *DINATIC
1.6120E-04, 1.6120E-04
*DIOAD, AMPLITUDE-ELGI
ELGI,P,1:
*EL FILE, POSITON-AVERAGED AT NODES
       *EL PRINT, POSITION=AVERAGED AT NODES
       *NODE FILE, GLOBAL=YES
       *NODE PRINT, GLOBAL=YES
    *FILE FORMAT, ASCII
*END STEP
*STEP, AMPLITUDE=RAMP, LINEAR, INC= 1, CYCLE= 1
TRANSIENT RESPONSE TO 2ND STEP OF WATER-ENTRY PRESSURE
       *DYNAMIC
    6.4480E-04,6.4480E-04

*DLOAD, AMPLITUDE=ELG1

ELG1, P, 1.

*DLOAD, AMPLITUDE=ELG2

ELG2, P, 1.
       *EL FILE, POSITION=AVERAGED AT NODES
       *EL PRINT, POSITION-AVERAGED AT NODES
       *NODE FILE, GLOBAL=YES
     *NODE PRINT, GLOBAL=YES
*FILE FORMAT, ASCII

*END STEP

*STEP, AMPLITUDE=RAMP, LINEAR, INC= 5, CYCLE= 1
TRANSIENT RESPONSE TO 5 SUBSEQUENT STEPS OF WATER-ENTRY PRESSURES
*DYNAMIC

8.0600E-04,4.0300E-03
*DLOAD, AMPLITUDE=ELG1
ELG1,P,1.
*DLOAD, AMPLITUDE=ELG2
ELG2,P,1.
*DLOAD, AMPLITUDE=ELG3
ELG3,P,1.
```

\*DICAD, AMPLITUDE=ELG4
ELG4, P, 1
\*DICAD, AMPLITUDE=ELG5
ELG5, P, 1
\*DICAD, AMPLITUDE=ELG6
ELG6, P, 1
\*DICAD, AMPLITUDE=ELG7
ELG7, P, 1
\*ELG7, P, 1
\*EL FILE, POSITION=AVERAGED AT NODES
SINV
\*EL PRINT, POSITION=AVERAGED AT NODES
SINV
\*NODE FILE, GLOBAL=YES
U
\*NODE PRINT, GLOBAL=YES
U
\*FILE FORMAT, ASCII
\*END SIEP

APPENDIX D
ENTRY INPUT FILE
CAP90.IN;6

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VARIABLE
DON'T PRINT
SYMMETRIC
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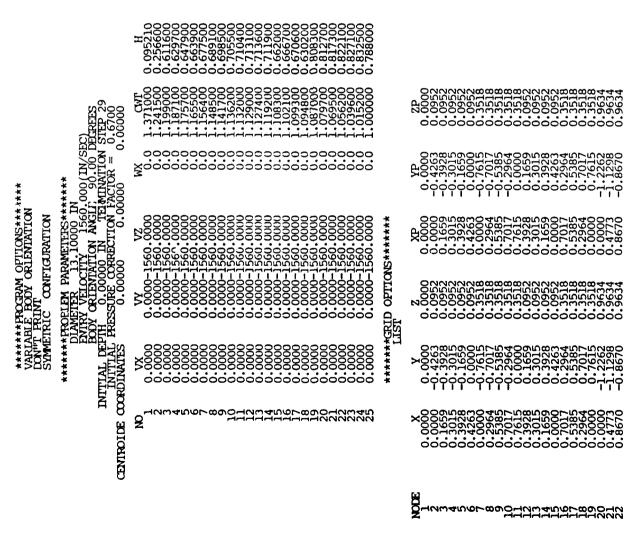
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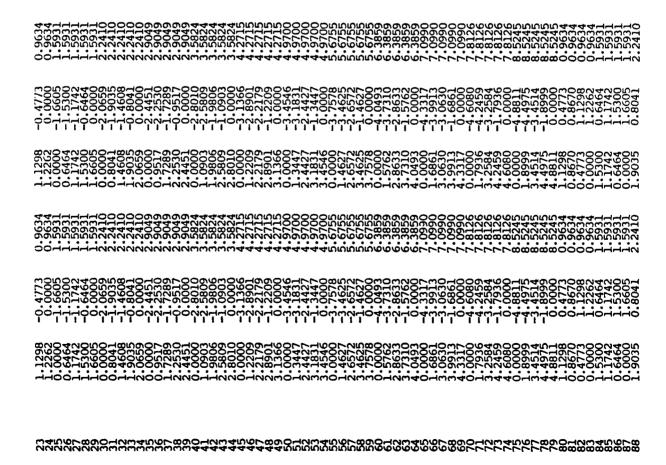
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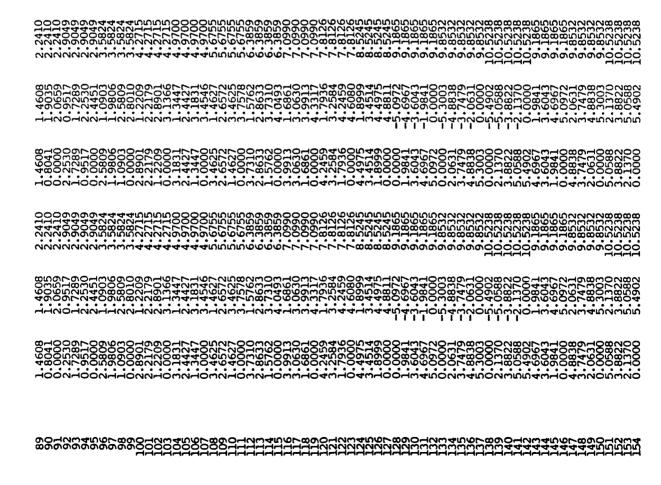
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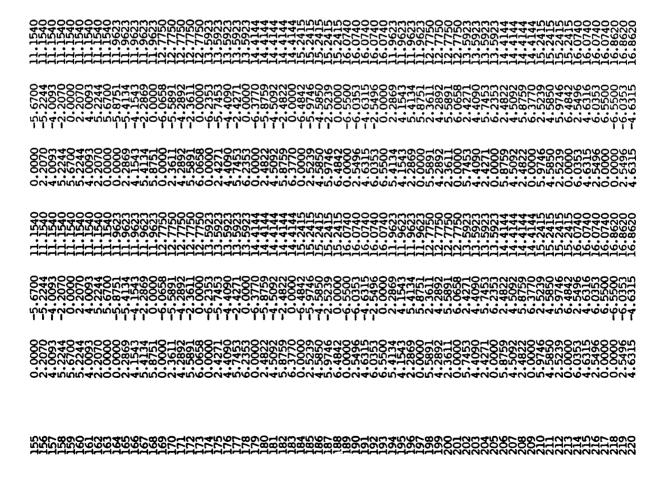
8955378999944444499994444449012012344567898900012344234567896789677777777222 1111 111 111 111 111 111 111 111 111	993333399994444499999444499123140123 1111155555500767898990111116666423445678989121340123 11111777722223	8871234889016789234512344667896789678901231234456789890111111116666661789 1111116666611111111111111111111111111	5601234890567892340123467856789012012344565678989001234234567896789678 8833333889933333999444449999444449012012344565678989001234456789678 1111666789678 1111666789678
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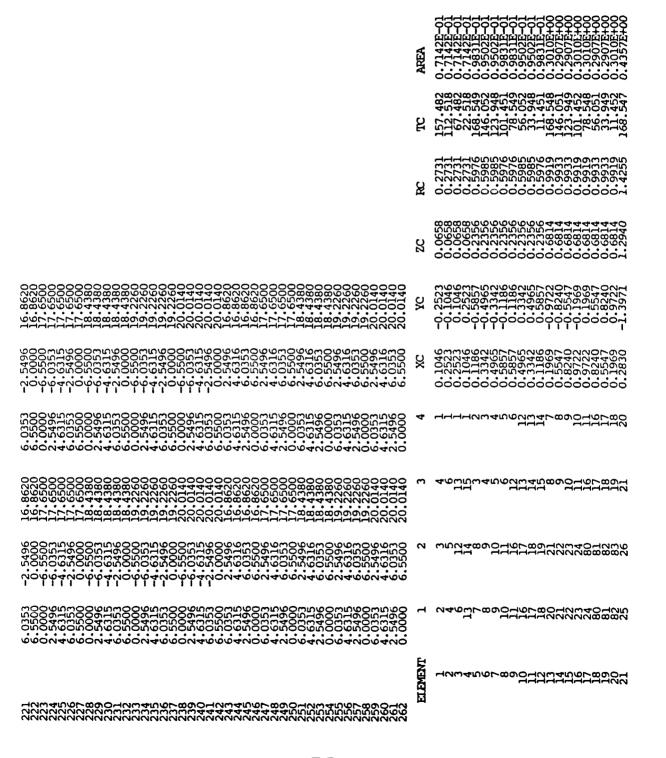
## APPENDIX E HARDCOPY OUTPUT FROM ENTRY

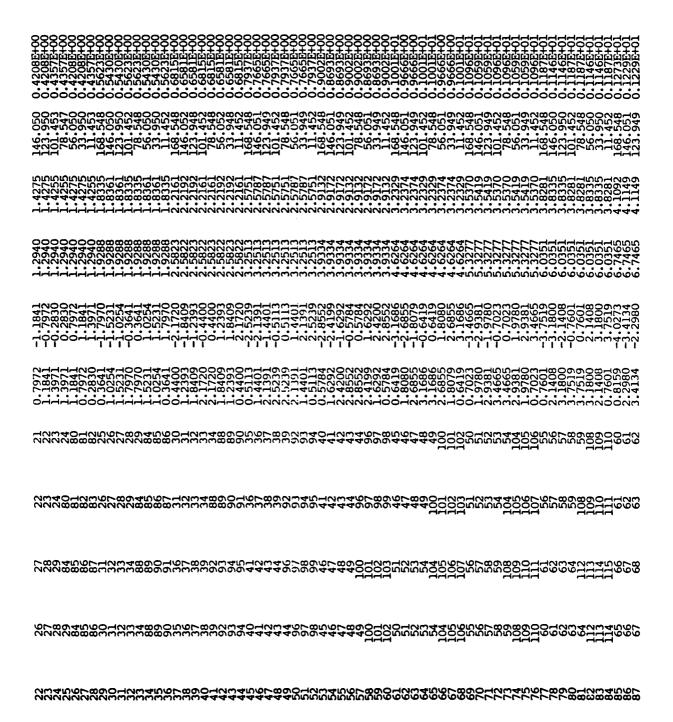


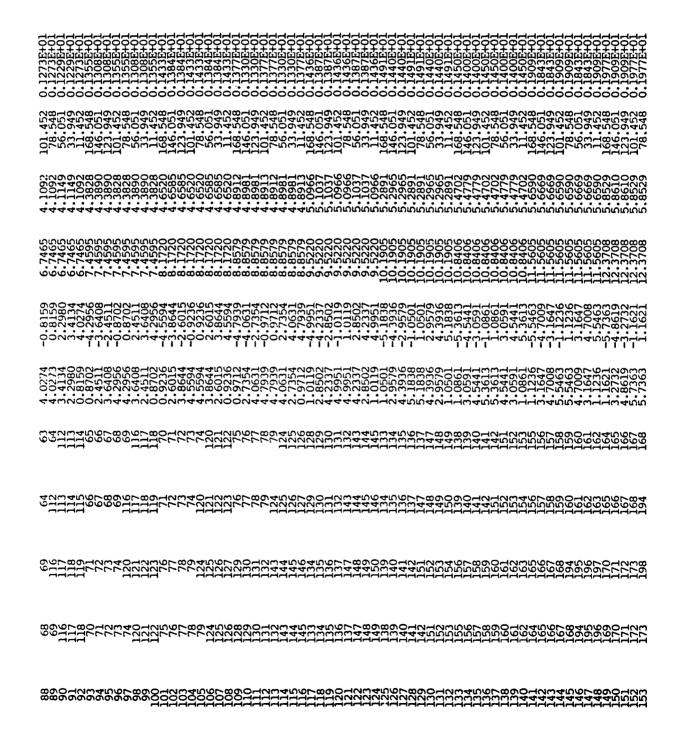


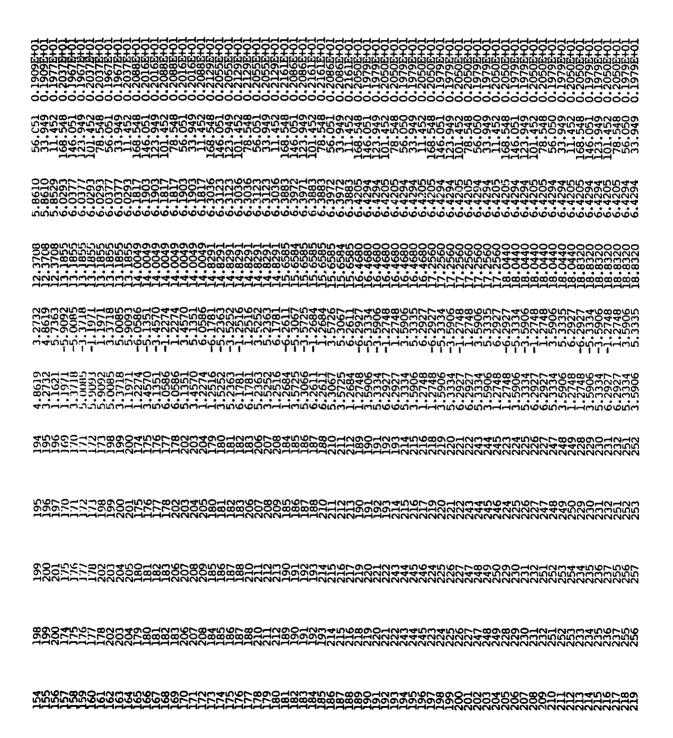






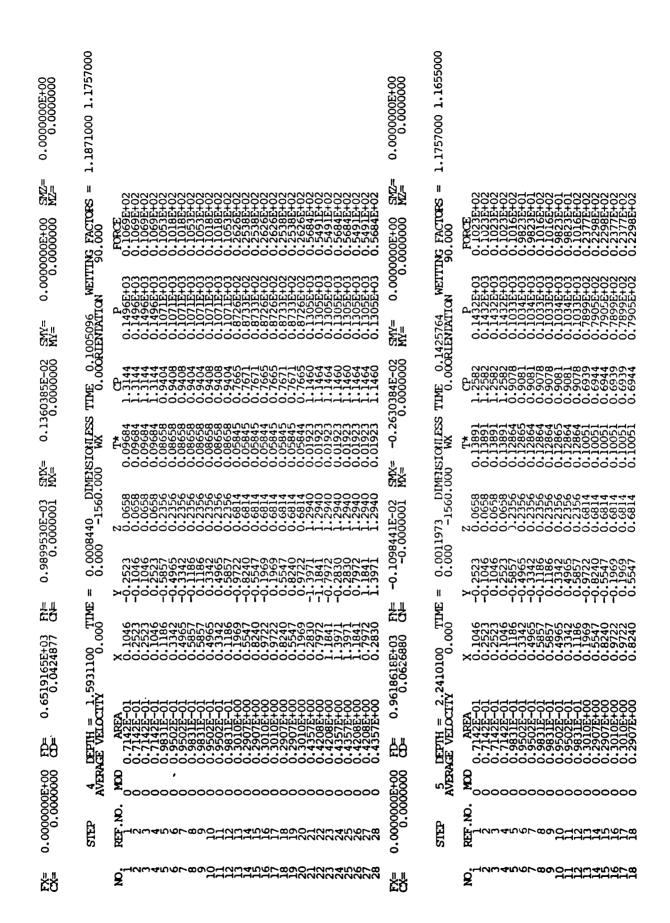






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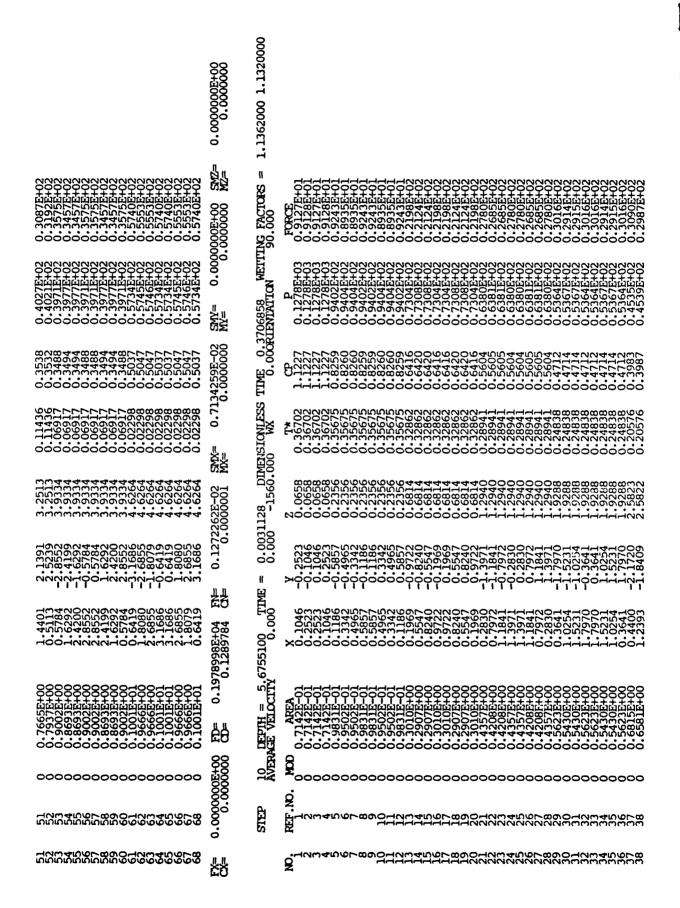
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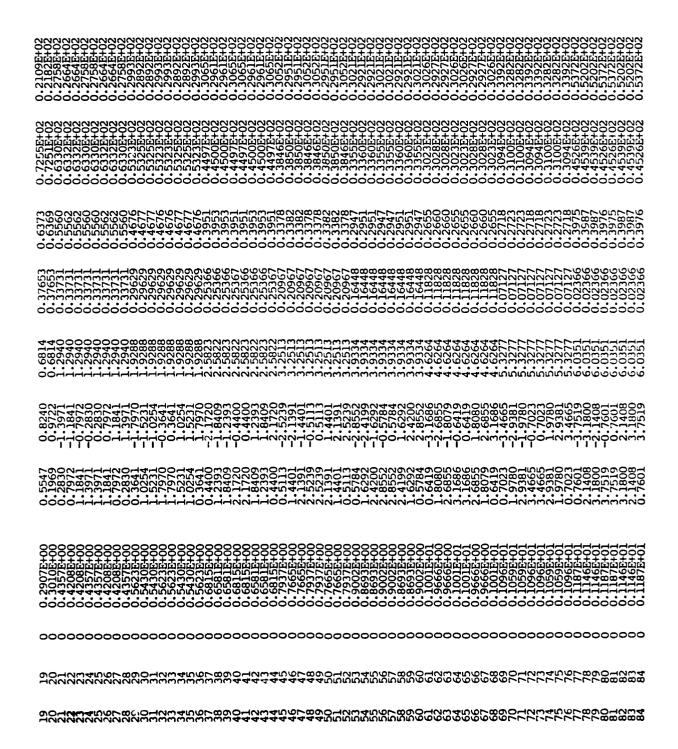
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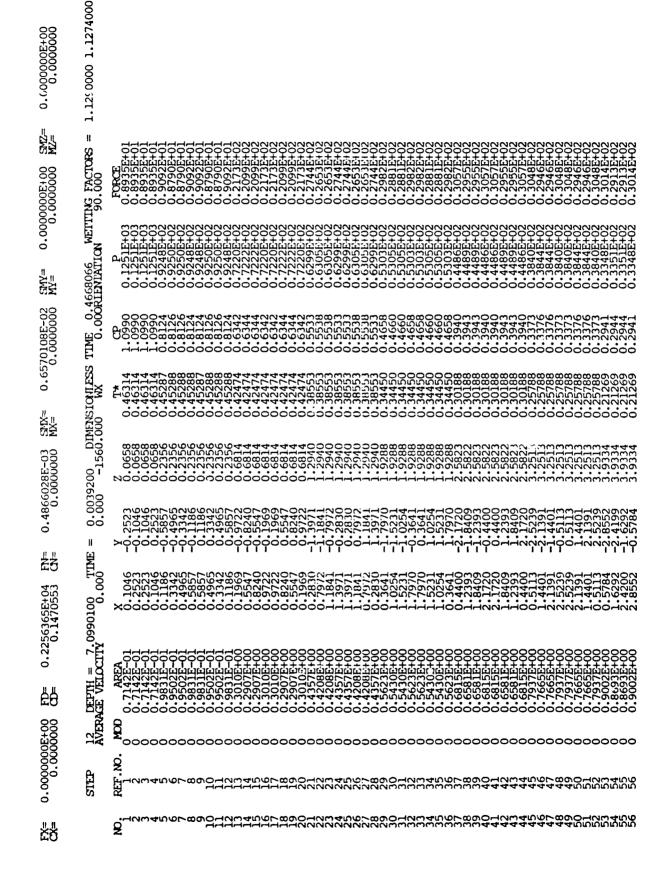
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1.8409 2.1720 2.1720 1.8409 1.239 0.4400 1.0965587		110000111111100111 7711100112722 11000111111100111 7711100112722 120011111111111111111111111111111111
0.6581E+00 0.6815E+00 0.6815E+00 0.6581E+00 0.6815E+00 TD= 0.148	00000000000000000000000000000000000000	0.000000000000000000000000000000000000
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	0.0000000000000000000000000000000000000	1.1485000 1.1417000	
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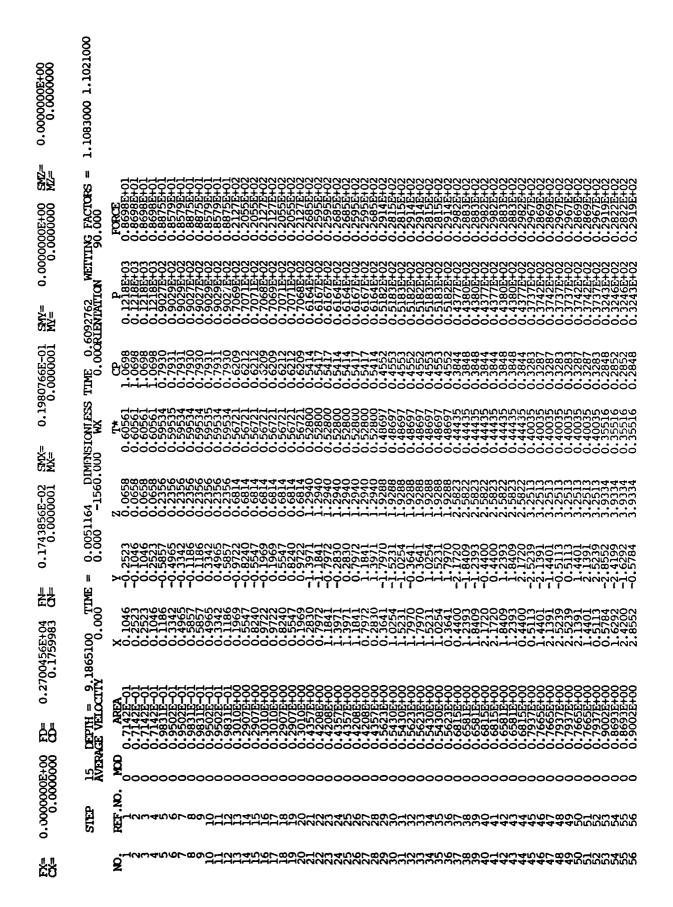


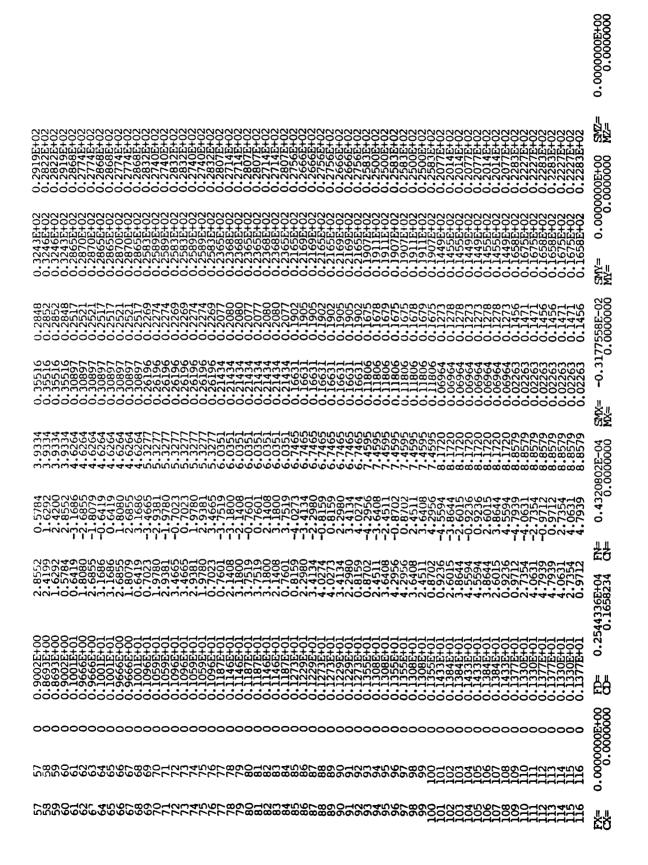
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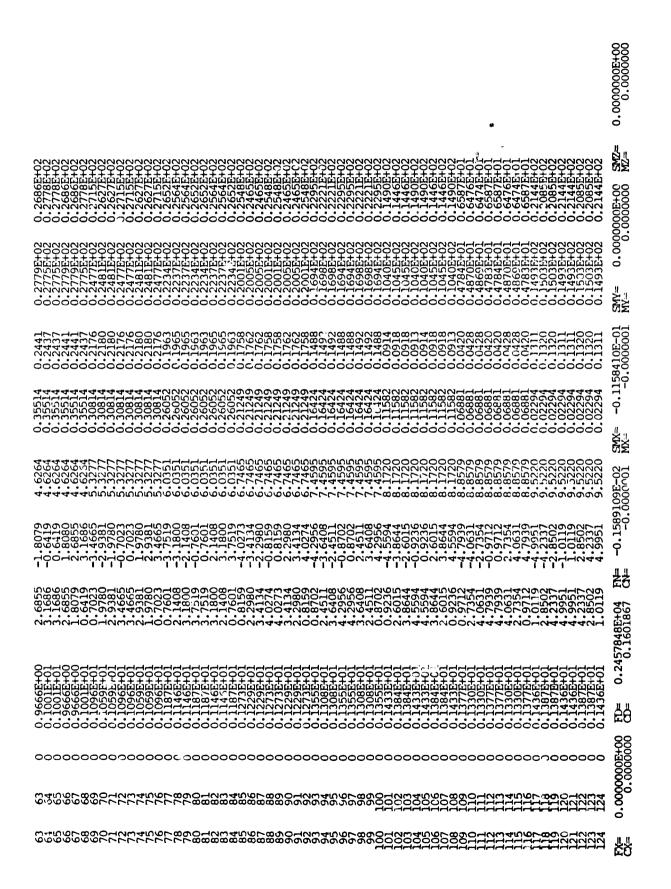
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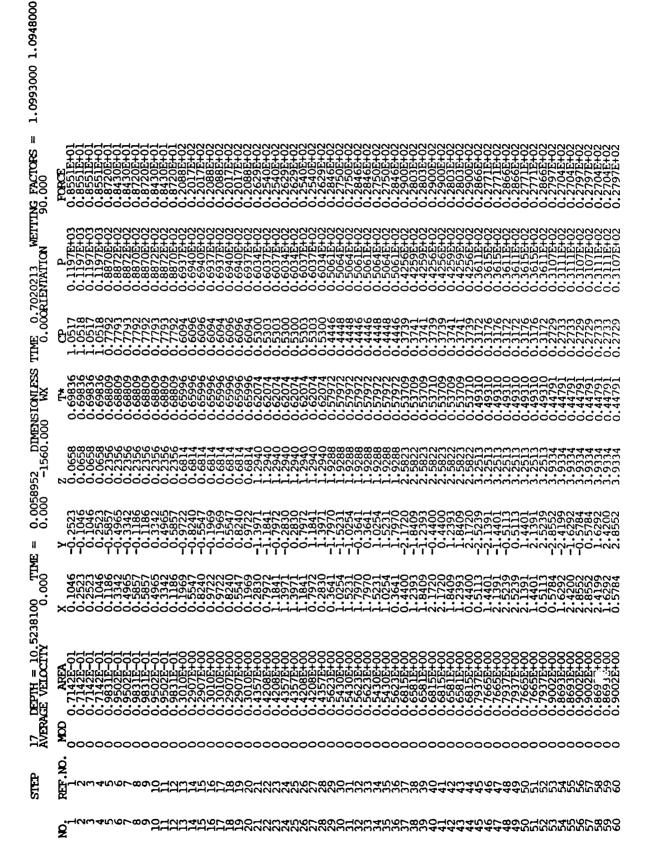
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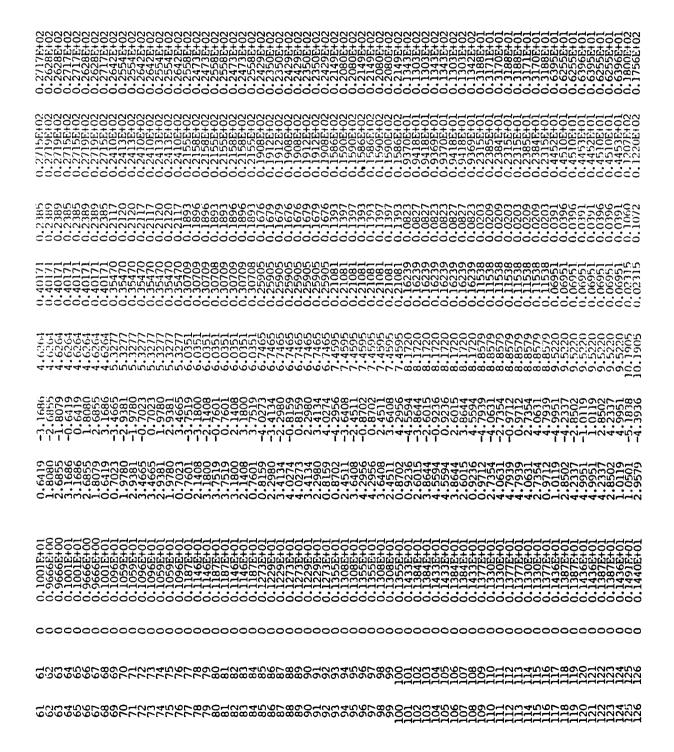
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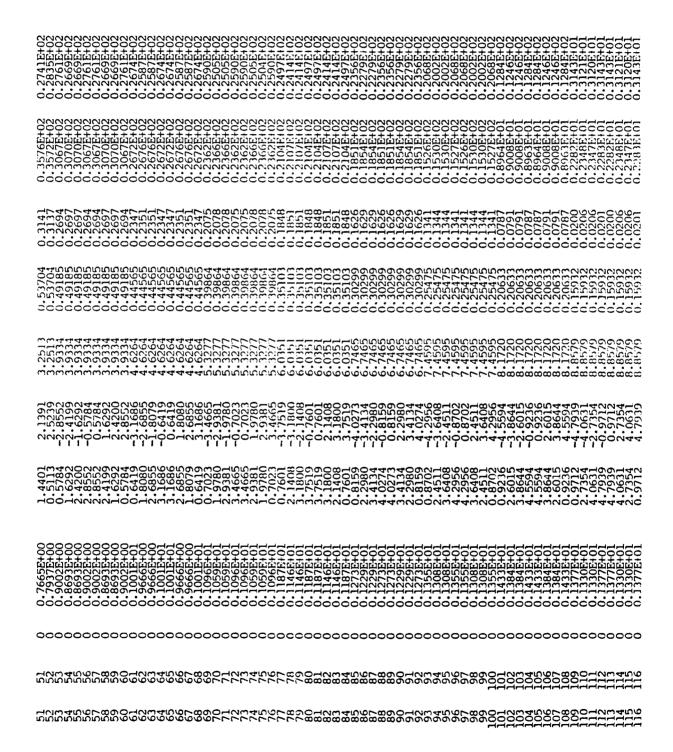


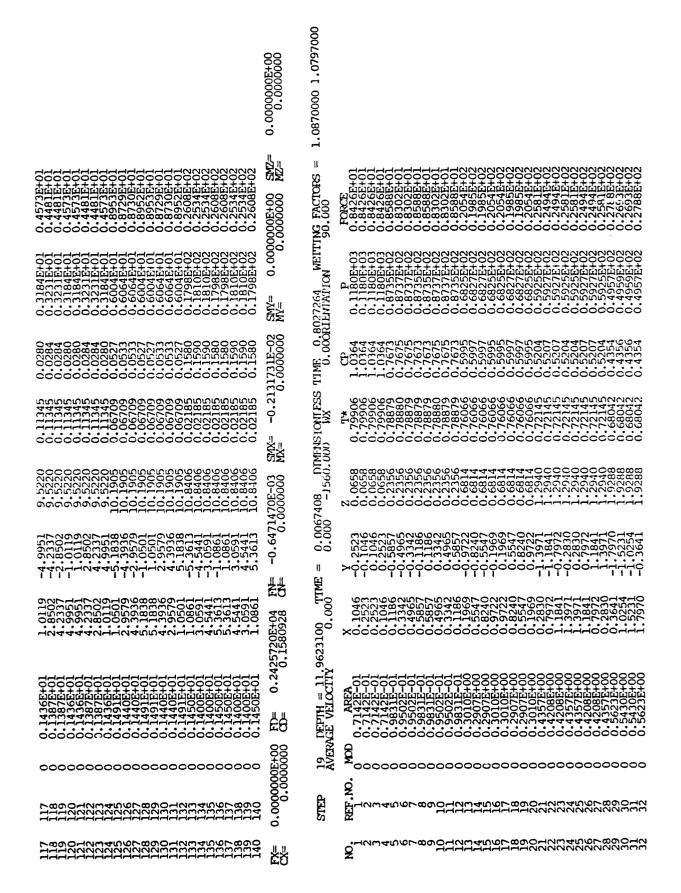






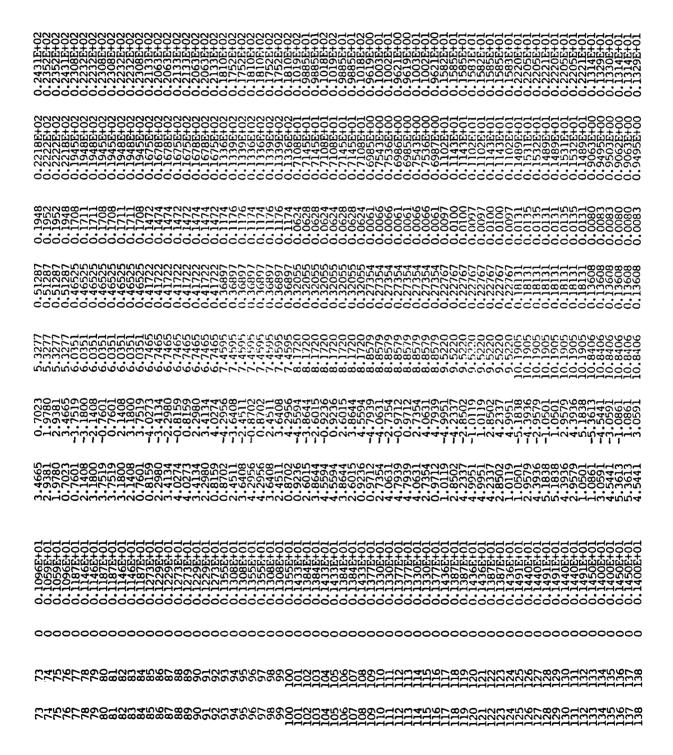
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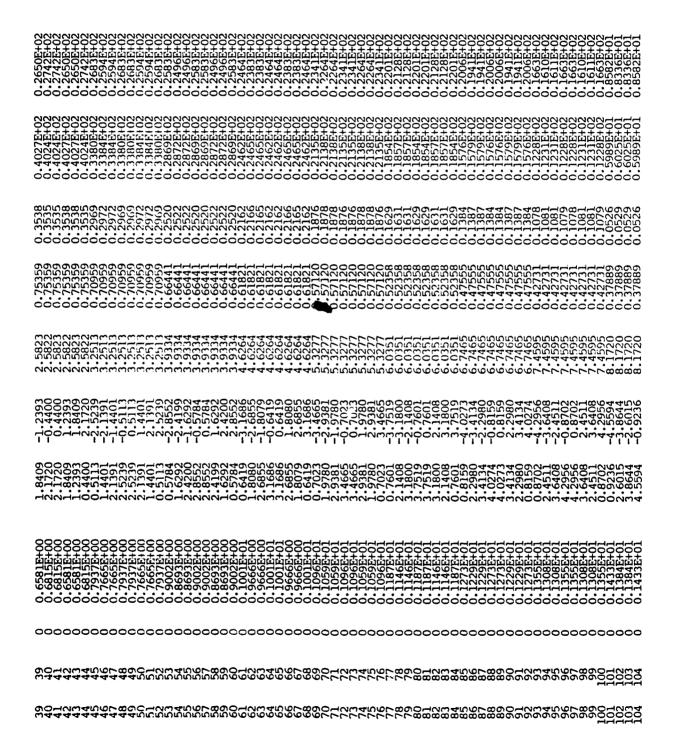


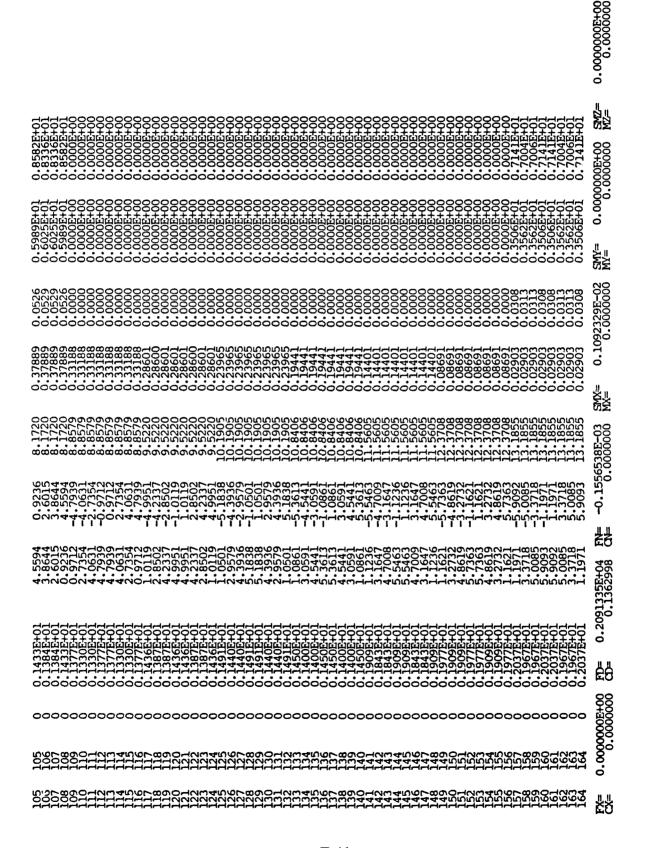
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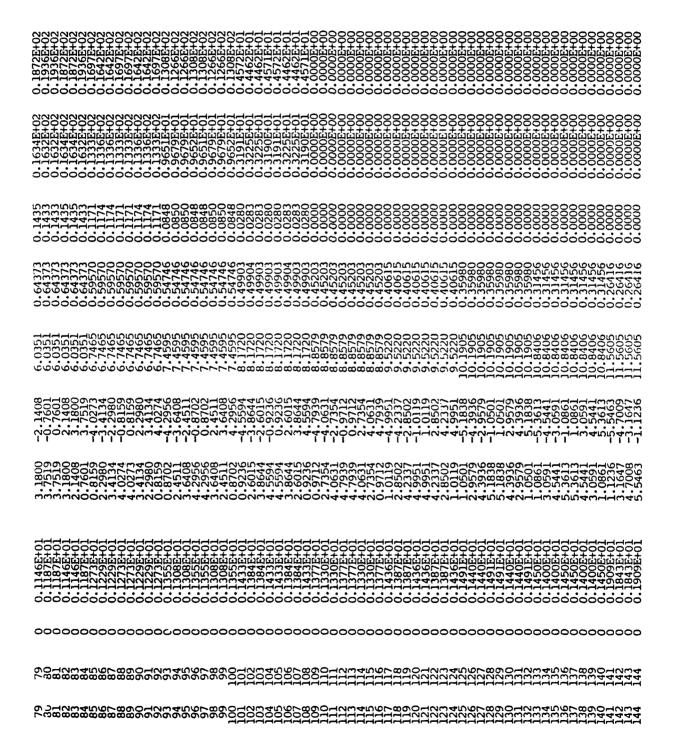
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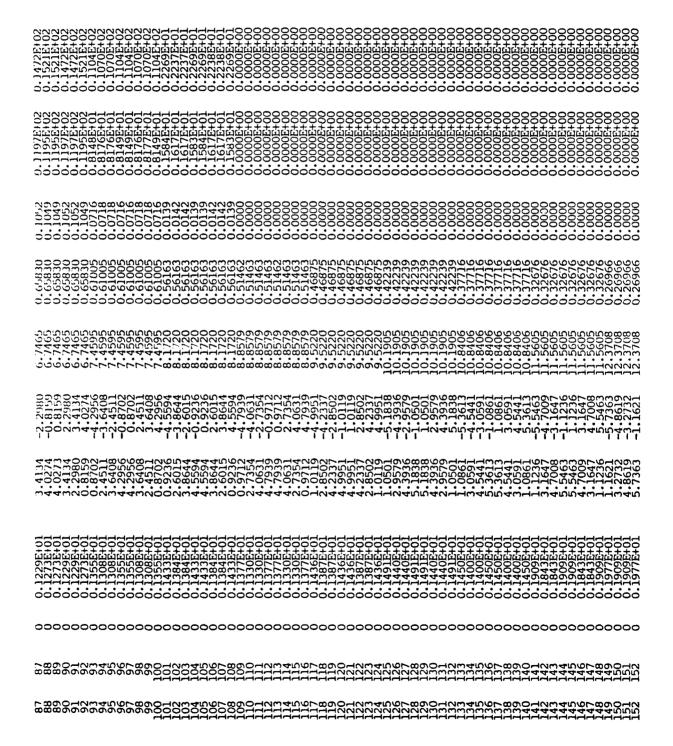
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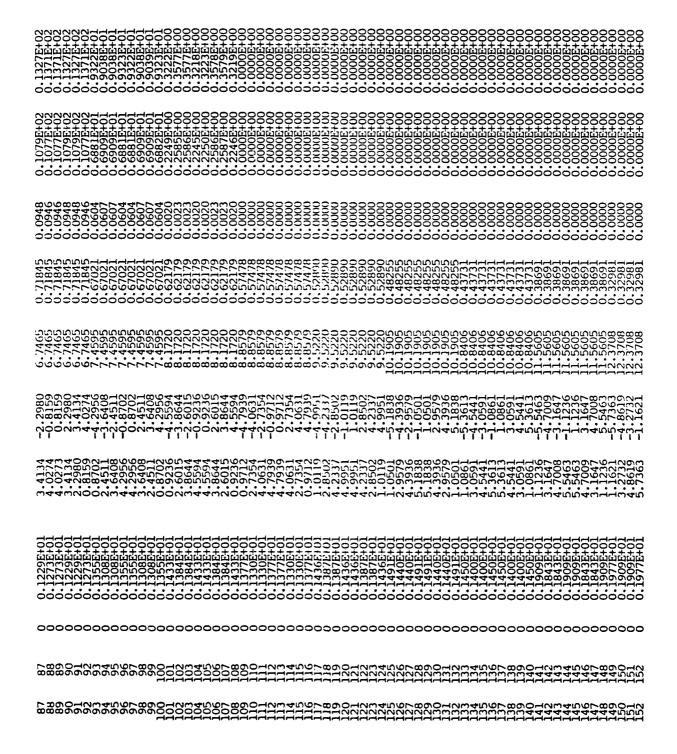
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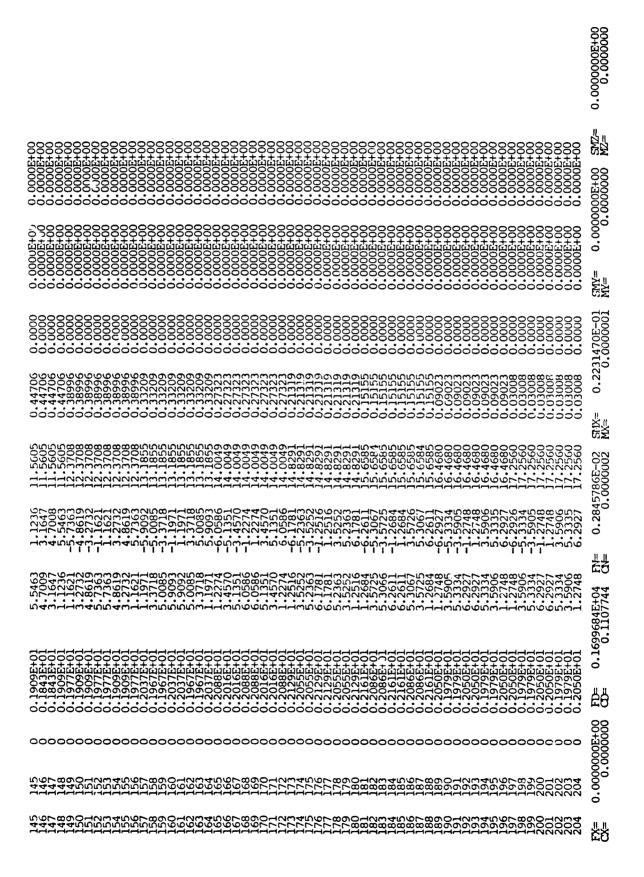
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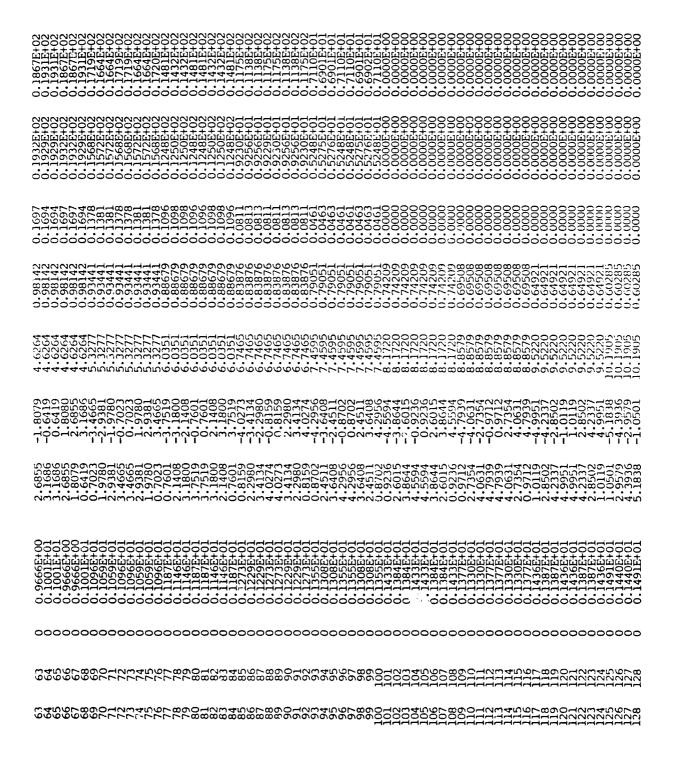


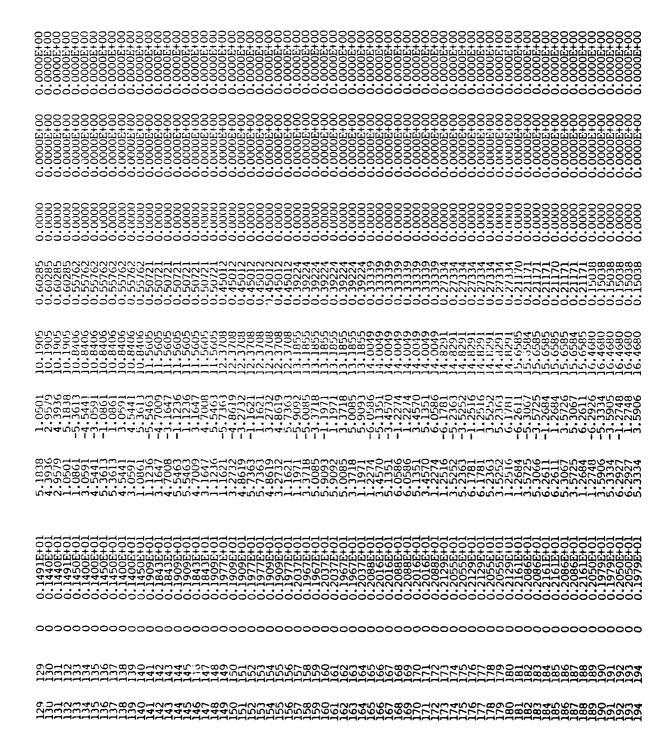
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	SMY= 0.00000 MY= 0.00000	1.2215725 WETTING 000RIENIATION 90	0.1114E+03 0.1114E+03 0.1114E+03 0.8150E+02 0.8150E+02 0.8158E+02 0.8158E+02 0.8156E+02 0.8160E+02 0.8160E+02
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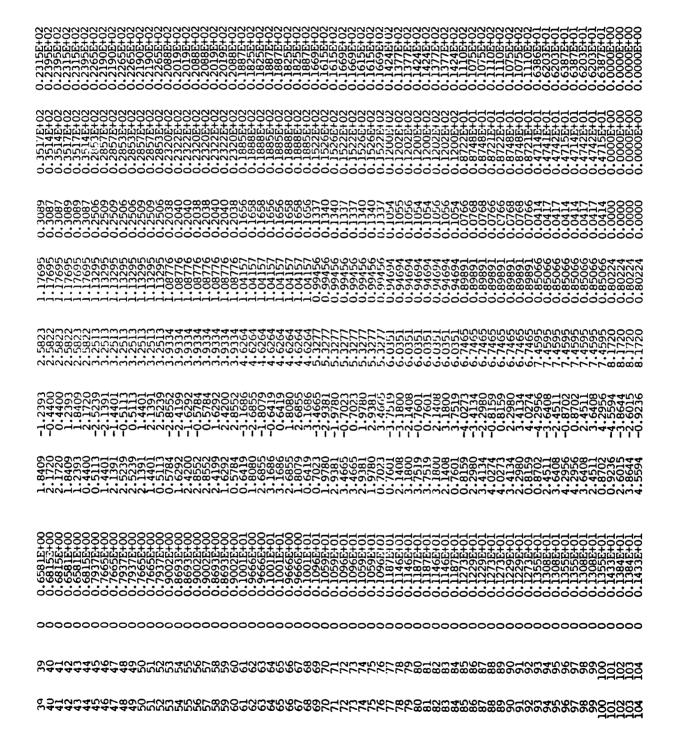
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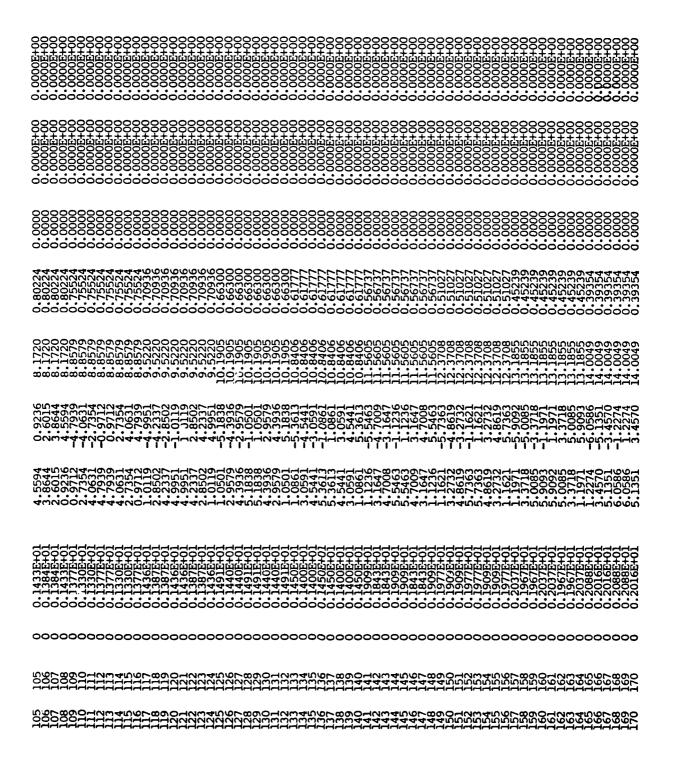




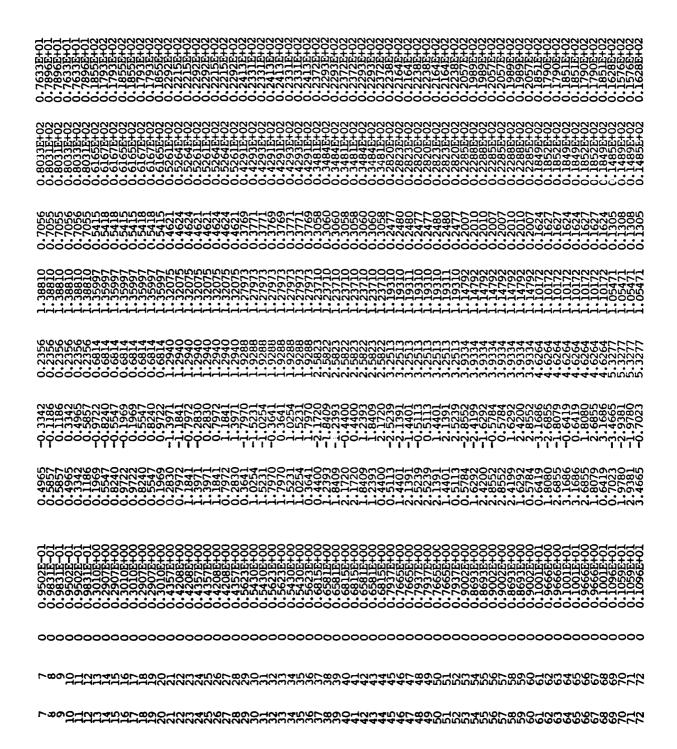


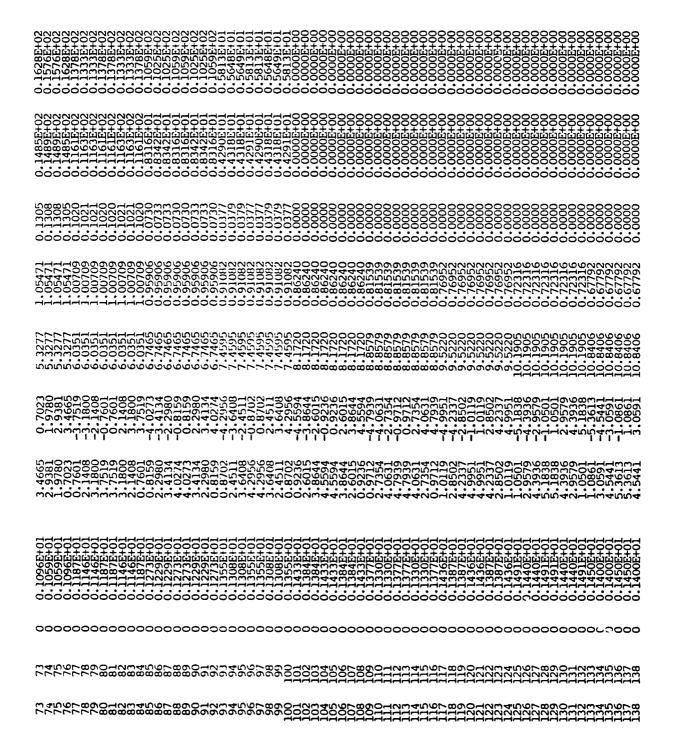
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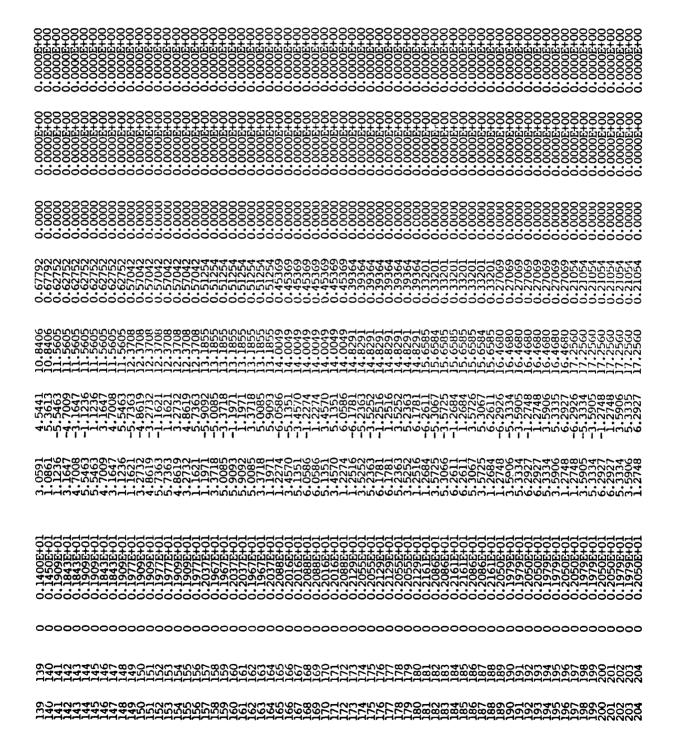




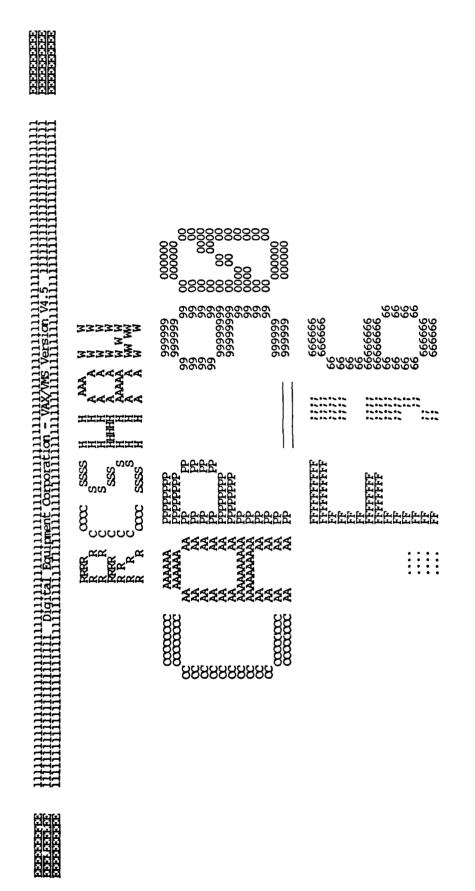
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######################################	0.2457762E-02 0.0000002
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	39684E+04 FNE 3.1049088 CNE
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SYCKNOWN THE PROPERTY OF STANDARD STAND	<u></u>



Job CAP 90 (1176) queued to TAA1 on 27-SEP-1989 10:38 by user RCSHAW, UIC [VMS,RCSHAW], under account VMS at priority 100, started on printer TAA1: On printer TAA1: File DIB3:[RCSHAW]CAP 90.F;6 (206,63.0), last revised on 27—SEP—1989 10:28, is a 6 block sequential file cwied by UIC [VMS,RCSHAW]. The records are variable length with implied (CR) carriage control. The longest record is 84 bytes. 

APPENDIX F
ABAQUS INPUT FILE
CAP90.INP; 10

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*HEADING
1.25-CAL OGIVAL NOSECAP QUATER-SYMMETRIC MODEL W/ INTERNAL BLADES
** NEUTRAL FILE GENERATED ON: 11-OCT-89 14:47:04 PATABA VERSION: 3.0
**
                                                                                            NODE DEFINITIONS
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   *NODE
                                                                                                                                                                                                                                                                  0.305999994E+00,

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0.401172698E+00,

0.401172668E+00,

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1.19TE-03 C. 000E+00 1.562E-03-9.336E+01 1.93EE-03-5.553E+01 2.333E-03-4.737E+01 2.732E-03-4.486E+01 4.732E-03-4.487E+01 4.732E-03-4.487E+01 5.504E-03-4.306E+01 5.895E-03-4.285E+01 5.504E-03-4.306E+01 5.895E-03-4.285E+01 6.264E-03-4.215E+01 6.741E-03-4.377E+01 5.504E-03-4.306E+01 5.895E-03-4.285E+01 6.264E-03-4.215E+01 6.741E-03-4.377E+01 5.504E-03-4.306E+01 5.895E-03-4.285E+01 1.026E-03-3.644E+01 8.722E-03-3.554E+01 7.223E-03-4.094E+01 7.713E-03-4.024E+01 1.026E-03-3.644E+01 8.722E-03-3.554E+01 1.1272-03-3.546E+01 8.212E-03-3.944E+01 8.222E-03-3.554E+01 1.1272-03-3.544E+01 7.773E-03-3.664E+01 1.776E-04 0.000E+00 8.440E-04 0.000E+00 8.40E-04 0.000E+00 8.20E-03 8.80E+01 8.20E-03-3.80E+01 8.20E
```

```
5.895E-03-1.586E+01 6.264E-03-1.526E+01 6.741E-03-1.436E+01 7.223E-03-1.336E+01 7.731E-03-1.336E+01 1.73E-03-1.336E+01 1.73E-03
```

```
1.177E-02 0.000E+00
*STEP, AMPLITUDE=RAMP, LINEAR, INC= 1, CYCLE= 1
TRANSIENT RESPONSE OF NOSECAP TO 1ST WATER-ENTRY STEP PRESSURE
*DYNAMIC
4.45E-05,4.45E-05
*DLOAD, AMPLITUDE=FIG1
EIG1,P,1.
*EL FILE, POSITION=AVERAGED AT NODES
             *EL PRINT, POSITION=AVERAGED AT NODES, ELSET=ELHALF
             *NODE FILE
*NOLE FILE.

*NODE PRINT, NSET=NHALF

UTILE FORMAT, ASCII

*END SIEP
*STEP, AMPLITUDE=RAMP, LINEAR, INC= 1, CYCLE= 1
TRANSIENT RESERONSE OF NOSECAP TO 2ND WATER-ENTRY STEP PRESSURE
*DIVAMIC
1.325E-04,1.325E-04
*DICAD, AMPLITUDE=ELG2
*ELG2, P.1.
*END SIEP
*SIEP AMPLITUDE=ELG2
*ELG2, P.1.
*END SIEP
*SIEP AMPLITUDE=ELG2
*SIEP AMPLITUDE=ELG2
*ICAD, AMPLITUDE=ELG3
*ICAD, AMPLI
             *NODE PRINT, NSET=NHALF
```

```
*PID STEP
**TEP AMPLITUDE=RAMP LINEAR INC= 1 CYCLE= 1
**TENSIANT RESPONSE OF NOSECAP TO 8TH WATER-ENTRY STEP PRESSURE 1
**TENSIANT RESPONSE OF NOSECAP TO 8TH WATER-ENTRY STEP PRESSURE 1
**TENSIANT RESPONSE OF NOSECAP TO 8TH WATER-ENTRY STEP PRESSURE 1
**TENSIANT RESPONSE OF NOSECAP TO 9TH WATER-ENTRY STEP PRESSURE 1
**TENSIANT RESPONSE OF NOSECAP TO 9TH WATER-ENTRY STEP PRESSURE 1
**TERNSIANT RESPONSE OF NOSECAP TO 9TH WATER-ENTRY STEP PRESSURE 1
**TERNSIANT RESPONSE OF NOSECAP TO 9TH WATER-ENTRY STEP PRESSURE 1
**TERNSIANT RESPONSE OF NOSECAP TO 9TH WATER-ENTRY STEP PRESSURE 1
**TERNSIANT RESPONSE OF NOSECAP TO 9TH WATER-ENTRY STEP PRESSURE 1
**TERNSIANT RESPONSE OF NOSECAP TO 9TH WATER-ENTRY STEP PRESSURE 1
**TERNSIANT RESPONSE OF NOSECAP TO 9TH WATER-ENTRY STEP PRESSURE 1
**TERNSIANT RESPONSE OF NOSECAP TO 9TH WATER-ENTRY STEP PRESSURE 1
**TERNSIANT RESPONSE OF NOSECAP TO 9TH WATER-ENTRY STEP PRESSURE 1
**TERNSIANT RESPONSE OF NOSECAP TO 9TH WATER-ENTRY STEP PRESSURE 1
**TERNSIANT RESPONSE OF NOSECAP TO 9TH WATER-ENTRY STEP PRESSURE 1
**TERNSIANT RESPONSE OF NOSECAP TO 9TH WATER-ENTRY STEP PRESSURE 1
**TERNSIANT RESPONSE OF NOSECAP TO 9TH WATER-ENTRY STEP PRESSURE 1
**TERNSIANT RESPONSE OF NOSECAP TO 9TH WATER-ENTRY STEP PRESSURE 1
**TERNSIANT RESPONSE OF NOSECAP TO 9TH WATER-ENTRY STEP PRESSURE 1
**TERNSIANT RESPONSE
```

```
DICAD, AMPLITUDE-ELGA
ELGAD, AMPLITUDE-ELGA

                      *DICAD, AMPLITUDE=EIG3
EIG3 P, 1
*DICAD, AMPLITUDE=EIG4
```

```
*END STEP
*STEP, AMPLITUDE=RAMP, LINEAR, INC= 1, CYCLE= 1
TRANSIENT RESPONSE OF NOSECAP TO 121H WATER-ENTRY STEP PRESSURE
  *STEP, AMPLITUDE=RAMP, LINEAR, INC= 1, CYCLE= 1
TRANSIENT RESPONSE OF NOSECAP TO 12TH WATER-ENTRY STEP PRESSURE
*DYNAMIC
4.049E-04, 4.049E-04
*DLOAD, AMPLITUDE=ELG1
ELG1, P, 1
*DLOAD, AMPLITUDE=ELG3
ELG2, P, 1
*DLOAD, AMPLITUDE=ELG3
ELG3, P, 1
*DLOAD, AMPLITUDE=ELG4
ELG4, P, 1
*DLOAD, AMPLITUDE=ELG5
ELG5, P, 1
*DLOAD, AMPLITUDE=ELG6
ELG6, P, 1
*DLOAD, AMPLITUDE=ELG8
ELG3, P, 1
*DLOAD, AMPLITUDE=ELG8
ELG3, P, 1
*DLOAD, AMPLITUDE=ELG9
ELG9, P, 1
*DLOAD, AMPLITUDE=ELG9
ELG9, P, 1
*DLOAD, AMPLITUDE=ELG1
ELG1, P, 1
*DLOAD, AMPLITUDE=ELG1
ELG1, P, 1
*DLOAD, AMPLITUDE=ELG12
ELG1, P, 1
*END STEP
*STEP, AMPLITUDE=ELG12
*TRANSIENT RESPONSE OF NOSECAP TO 13TH WATER-ENTRY STEP PRESSURE
*DUVANIC
4.057E-04, 4.057E-04
*DLOAD, AMPLITUDE=ELG1
TRANSIENT RESPONSE OF NOSECAP TO 13TH WATER-ENTRY STEP PRESSURE
*DOVINGMIC*
4.057E-04,4.057E-04
*DIOAD, AMPLITUDE=EIG1
EIG1,P,1*
*DIOAD, AMPLITUDE=EIG2
EIG2,P,1*
*DIOAD, AMPLITUDE=EIG3
EIG3,P,1*
*DIOAD, AMPLITUDE=EIG5
EIG5,P,1*
*DIOAD, AMPLITUDE=EIG5
EIG6,P,1*
*DIOAD, AMPLITUDE=EIG7
EIG7,P,1*
*DIOAD, AMPLITUDE=EIG8
EIG8,P,1*
*DIOAD, AMPLITUDE=EIG9
EIG9,P,1*
*DIOAD, AMPLITUDE=EIG9
EIG9,P,1*
*DIOAD, AMPLITUDE=EIG10
EIG10,P,1*
*DIOAD, AMPLITUDE=EIG11
EIG11,P,1*
*DIOAD, AMPLITUDE=EIG12
EIG12,P,1*
*DIOAD, AMPLITUDE=EIG13
EIG13,P,1*
*EIG13,P,1*
**EIG13,P,1*
**EIG1
```

```
EIG1, P, 1.
*DIOAD, AMPLITUDE=EIG2
EIG2, P, 1.
*DIOAD, AMPLITUDE=EIG3
*DICAD, AMPLITUDE=EIG3
EIG3, P, 1.
*DICAD, AMPLITUDE=EIG4
EIG4, P, 1.
*DICAD, AMPLITUDE=EIG5
EIG5, P, 1.
*DICAD, AMPLITUDE=EIG5
EIG5, P, 1.
*DICAD, AMPLITUDE=EIG6
EIG6, P, 1.
*DICAD, AMPLITUDE=EIG7
EIG7, P, 1.
*DICAD, AMPLITUDE=EIG8
EIG8, P, 1.
*DICAD, AMPLITUDE=EIG9
EIG9, P, 1.
*DICAD, AMPLITUDE=EIG10
EIG10, P, 1.
*DICAD, AMPLITUDE=EIG11
EIG11, P, 1.
*DICAD, AMPLITUDE=EIG12
EIG12, P, 1.
*DICAD, AMPLITUDE=EIG12
EIG13, P, 1.
*DICAD, AMPLITUDE=EIG13
EIG13, P, 1.
*DICAD, AMPLITUDE=EIG14
EIG14, P, 1.
*END STEP
*STEP, AMPLITUDE=EIG14
EIG14, P, 1.
*END STEP
*STEP, AMPLITUDE=EIG14
*DICAD, AMPLITUDE=EIG14
*DICAD, AMPLITUDE=EIG14
*DICAD, AMPLITUDE=EIG16
*DICAD, AMPLITUDE=EIG16
*DICAD, AMPLITUDE=EIG16
*DICAD, AMPLITUDE=EIG2
*DICAD, AMPLITUDE=EIG3
EIG1, P, 1.
*DICAD, AMPLITUDE=EIG3
EIG3, P, 1.
*DICAD, AMPLITUDE=EIG3
EIG3, P, 1.
*DICAD, AMPLITUDE=EIG3
EIG3, P, 1.
*DICAD, AMPLITUDE=EIG4
*EIG4, P, 1.
*DICAD, AMPLITUDE=EIG4
*EIG4, P, 1.
*DICAD, AMPLITUDE=EIG5
    EIC4, P, 1
*DLOAD,
                                                              AMPLITUDE=ELG5
  EIGS, P, 1
*DLOAD,
EIGG, P, 1
*DLOAD,
                                                                 AMPLITUDE=ELG6
 *DIOAD, AMPLITUDE=ELG7
ELG7,P,1
*DIOAD, AMPLITUDE=ELG8
*DIOAD, AMPLITUDE=ELG9
ELG9,P,1
*DIOAD, AMPLITUDE=ELG10
ELG10,P,1
*DIOAD, AMPLITUDE=ELG11
ELG11,P,1
*DIOAD, AMPLITUDE=ELG12
ELG12,P,1
*DIOAD, AMPLITUDE=ELG13
ELG13,P,1
*DIOAD, AMPLITUDE=ELG14
ELG14,P,1
*DIOAD, AMPLITUDE=ELG14
ELG14,P,1
*DIOAD, AMPLITUDE=ELG15
ELG15,P,1
*END STEP
*STEP, AMPLITUDE=RAMP, LINEAR, INC= 1, CYCLE= 1
TRANSIENT RESPONSE OF NOSECAP TO 16TH WATER-ENTRY STEP PRESSURE
*DYNAMIC
                                                                 AMPLITUDE=ELG7
```

```
3.877E-04,3.877E-04
*DIOAD, AMPLITUDE=EIG1
EIG1,P,1.
*DIOAD, AMPLITUDE=EIG2
EIG2,P,1.
*DIOAD, AMPLITUDE=EIG3
EIG3,P,1.
*DIOAD, AMPLITUDE=EIG4
      **ELG3.P', 1
*DICAD, AMPLITUDE=ELG4
ELG4.P, 1.
*DICAD, AMPLITUDE=ELG5
ELG5.P, 1.
*DICAD, AMPLITUDE=ELG6
ELG6.P', 1.
*DICAD, AMPLITUDE=ELG7
ELG7.P, 1.
*DICAD, AMPLITUDE=ELG8
ELG8.P, 1.
*DICAD, AMPLITUDE=ELG9
ELG9.P, 1.
*DICAD, AMPLITUDE=ELG9
ELG9.P, 1.
*DICAD, AMPLITUDE=ELG10
ELG10.P, 1.
*DICAD, AMPLITUDE=ELG12
ELG12.P, 1.
*DICAD, AMPLITUDE=ELG13
ELG13.P, 1.
*DICAD, AMPLITUDE=ELG14
ELG13.P, 1.
*DICAD, AMPLITUDE=ELG14
ELG13.P, 1.
*DICAD, AMPLITUDE=ELG15
ELG13.P, 1.
*DICAD, AMPLITUDE=ELG16
ELG16.P, 1.
*DICAD, AMPLITUDE=ELG16
ELG16.P, 1.
*EDICAD, AMPLITUDE=ELG16
ELG16.P, 1.
*END STEP
*STEP, AMPLITUDE=ELG16
ELG16.P, 1.
*STEP, AMPLITUDE=ELG16
E
                                                                                                          04,3.911E-04
AMPLITUDE=EIG1
                                                                                                            AMPLITUDE=ELG2
                                                                                                            AMPLITUDE=ELG3
                                                                                                            AMPLITUDE=ELG4
                                                                                                          AMPLITUDE=ELG5
                                                                                                            AMPLITUDE=ELG6
*DIOAD, AMPLITUDE=EIG7
EIG7, P, 1
*DIOAD, AMPLITUDE=EIG8
EIG8, P, 1
*DIOAD, AMPLITUDE=EIG9
EIG9, P, 1
*DIOAD, AMPLITUDE=EIG10
EIG10, P, 1
*DIOAD, AMPLITUDE=EIG11
EIG11, P, 1
*DIOAD, AMPLITUDE=EIG12
EIG12, P, 1
*DIOAD, AMPLITUDE=EIG13
EIG13, P, 1
*DIOAD, AMPLITUDE=EIG14
EIG14, P, 1
                                                                                                       AMPLITUDE=ELG7
```

```
*DIOAD, AMPLITUDE=EIG15
EIG15,P,1.
*DIOAD, AMPLITUDE=EIG16
EIG16,P,1.
*DIOAD, AMPLITUDE=EIG17
EIG17,P,1.
*END STEP

*STEP, AMPLITUDE=RAMP, LINEAR, INC= 1, CYCLE= 1
TRANSIENT RESPONSE OF NOSECAP TO 18TH WATER-ENTRY STEP PRESSURE
*DYNAMIC
3.690E-04,3.690E-04
*DIOAD, AMPLITUDE=EIG1
EIG1,P,1.
*DIOAD, AMPLITUDE=EIG2
EIG2,P,1.
*DIOAD, AMPLITUDE=EIG3
EIG3,P,1.
*DIOAD, AMPLITUDE=EIG4
EIG4,P,1.
*DIOAD, AMPLITUDE=EIG5
EIG5,P,1.
*DIOAD, AMPLITUDE=EIG6
EIG6,P,1.
*DIOAD, AMPLITUDE=EIG6
EIG6,P,1.
*DIOAD, AMPLITUDE=EIG6
EIG6,P,1.
*DIOAD, AMPLITUDE=EIG6
                                                                             ÄMPLITUDE=ELG6
FIG5, P, 1.
*DICAD, AMPLITUDE=FIG6
FIG6, P, 1.
*DICAD, AMPLITUDE=FIG7
FIG7, P, 1.
*DICAD, AMPLITUDE=FIG8
FIG8, P, 1.
*DICAD, AMPLITUDE=FIG9
FIG9, P, 1.
*DICAD, AMPLITUDE=FIG10
FIG10, P, 1.
*DICAD, AMPLITUDE=FIG11
FIG11, P, 1.
*DICAD, AMPLITUDE=FIG12
FIG12, P, 1.
*DICAD, AMPLITUDE=FIG13
FIG13, P, 1.
*DICAD, AMPLITUDE=FIG14
FIG14, P, 1.
*DICAD, AMPLITUDE=FIG15
FIG15, P, 1.
*DICAD, AMPLITUDE=FIG16
FIG16, P, 1.
*DICAD, AMPLITUDE=FIG16
FIG17, P, 1.
*DICAD, AMPLITUDE=FIG17
FIG17, P, 1.
*DICAD, AMPLITUDE=FIG18
FIG18, P, 1.
**END STEP
```

# APPENDIX G USER INSTRUCTIONS—WATER ENTRY STRUCTURAL TECHNIQUE

# APPENDIX G

# USER INSTRUCTIONS-WATER ENTRY STRUCTURAL TECHNIQUE

These user instructions for WEST are written assuming that the user is familiar with both PATRAN and ABAQUS. Therefore, no effort is expended on their use in these instructions. The user instructions for WEST are divided into three parts:

Part 1 provides the instructions for the generic use of the ENTRY code, without any pre- or post-processing nor finite element analysis of the structure. These instructions are extracted from the original users manual for ENTRY, which is contained in reference 19.

Part 2 lists the special instructions a user will need to generate a finite element mesh that can accommodate or bypass the restrictions imposed by the ENTRY code.

Part 3 provides instructions on how to run the translators PATENTR and ENTPRES, and how to activate the added output options for element pressure- and load-time history files in an ENTRY run.

#### PART 1

#### INSTRUCTIONS FOR THE GENERIC USE OF THE ENTRY CODE

The current version of the ENTRY code can be applied to arbitrary bodies. The grid describing the entry body may contain up to 750 nodes and 500 elements. However, execution will terminate when more than 300 of these elements become submerged. These instructions describe the available program options, necessary input cards and output format.

# Program Options and Required Input

Program input can be divided into three parts. In the first, the basic program options are specified:

Card No.	Variable	Format
1	CONSTANT or VARIABLE body orientation	2 <b>A</b> 4
2	PRINT or DON'T PRINT	3A4
3	ASYMMETRIC or SYMMETRIC mode	3A4

Under the CONSTANT body orientation option the entry model is assumed to retain its initial orientation and velocity throughout the entry process. The natural problem variable in this case is depth rather than time. With little increase in computational time, pressures and forces can be evaluated for a number of different wetting factors,  $C_{\rm W}$ . The VARIABLE body orientation option allows the velocity, orientation, wetting factor and time increments between steps to be varied. The only restriction is that the angular velocity of the body must be small enough to insure that the depth of the body increases monotonically in time. The maximum number of steps is limited to 49.

The PRINT option is used to obtain flow field and element information at each step of the calculation. It is applied only for diagnostic purposes. The second option, DON'T PRINT, is recommended and produces only grid information and the final pressures and forces on the model.

If the SYMMETRIC mode option is used, the entry model is assumed to possess planar symmetry about the y-z plane. The ASYMMETRIC option does not assume any symmetry and hence can be applied to arbitrary bodies. This mode is also used on symmetric bodies where  $\rm V_{\rm X}$  is non-zero.

The second set of input cards describes the entry conditions. The required information differs depending on whether the CONSTANT or VARIABLE body orientation option is used. For the CONSTANT option the following data cards are required:

Card No.	Variable	Format
4 5 6	IMAX, D, VENTRY, ANG, SUMT, HMIN, DH, ALPHA CGL, FCF, ANGB, NNLD NCW, CW(1), CW(2)(CW(NCW))	15,5X,7F10.0 3F10.0, I5 15,5X,7F10.0/
7	omit	(8F10.0)

These variables are defined as follows:

IMAX Number of steps at which pressures and loads are calculated. The present calculative procedure inserts the model into the water in a series of steps, each at a greater depth than the preceding one. When the step count becomes greater than IMAX, execution is terminated.

D Diameter (in inches). This quantity is only used for calculating force coefficients.

VENTRY Entry velocity in (in./sec.).

DH

ANG Orientation of the model (in degrees) relative to the water surface (see Figure G-1).

SUMT Program time limit. This variable is not used in the VAX implementation of this code. Set equal to zero.

Initial body depth (i.e., measured from the lowest point on the body). This parameter is zero if the loads are calculated from the time of initial wetting. Note that if this variable is not zero pressures and forces are first calculated at HMIN+2DH. This parameter allows pressures and forces at a particular depth to be determined without calculating the entire force-time history from initial wetting.

Increment in depth (in inches) between successive steps. It is necessary to coordinate this variable with the specified model grid which is defined on the last set of data cards. The following apply to determining DH:

- a. OBLIQUE ENTRY WITH STANDARD GRID OPTION. DH should be picked so that the average element is submerged in two steps. On models of complex shape this criteria can only be satisfied in the mean and primary consideration should be given to the portion of the body which experiences the greatest load. Generally this will be on elements whose plane is perpendicular to the direction of motion.
- b. VERTICAL ENTRY WITH STANDARD GRID OPTION OR OBLIQUE ENTRY WITH THE OGIVE OPTION. For vertical entry or if the OGIVE grid option is used, elements will have a pair of sides parallel to the water surface. In this case it is important to choose the step size very precisely so that each element

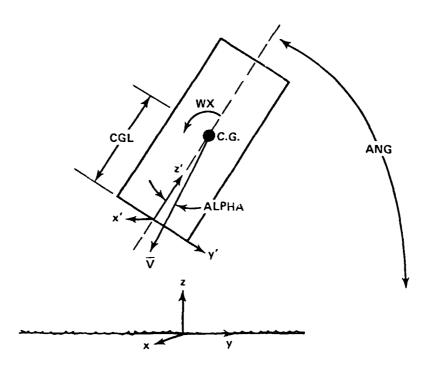


FIG. G-1 TERMS DEFINED

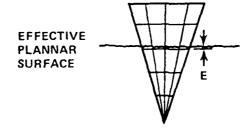


FIG. G-2 PROFILE OF CONE GRID

will be submerged in exactly two steps. To insure that the top row of elements is included in an unmodified state in the code and that the next row of elements is excluded, the actual water surface should fall a small distance  $\epsilon$  above the upper edge of the top row of elements to be included as shown in Figure G-2. Here  $0<\epsilon<\Delta$  h where  $\Delta$  h is defined by

 $4 h = \sqrt{\text{average element area}}$ 

ALPHA Angle of attack (in degrees). See Figure G-1.

CGL z' coordinate of the center of gravity. See Figure G-1.

Pressure correction factor on elements with a modification code of 1. For the oblique entry of blunt bodies (nose length/diameter < 1) set to unity. For other cases use a value of 0.67.

ANGB Yaw angle (in degrees). Velocity components in the x, y, z directions are  $V_{I}$ sin (ANGB),  $-V_{I}$ cos(ANG)cos(ANG+ALPHA) and  $-V_{I}$ cos(ANG)sin(ANG+ALPHA).

NNLD Number of no-load elements.

NCW Number of wetting factors to be used. Since the most appropriate value may not be clear, for little extra computational cost, pressure and loads may be calculated for several different wetting factor values.

Wetting Factor. This parameter describes the rate of surface rise and is equal to the ratio of h/h' defined in Figure G-3. For best results, the test cases reported on in Reference 19 should be used as a guide. An approximate rule for determining this parameter is as follows:

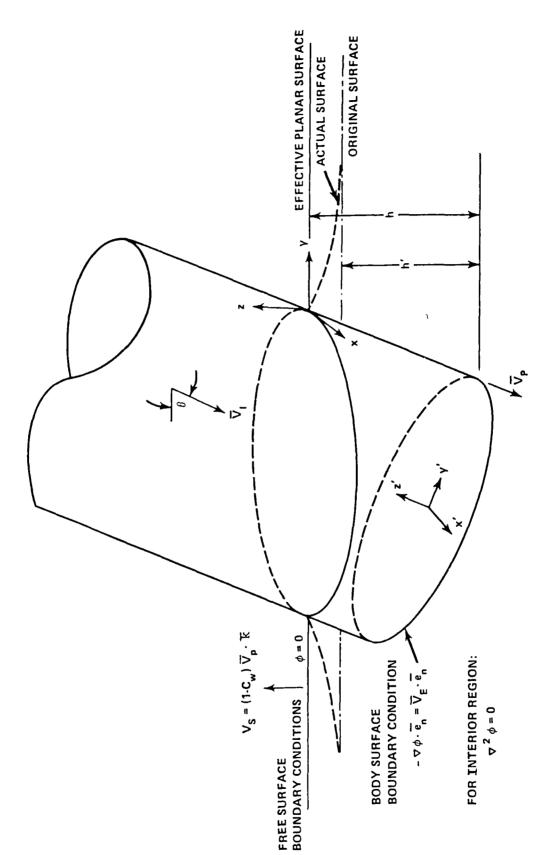


FIG. 6-3 PROBLEM FORMULATION

(1) POINTED BODIES (ALSO INCLUDES SLIGHTLY BLUNTED ONES). Determine the angle,  $\theta_{\text{c}}$ , between the tangent to the body surface and the body axis at both the nosetip and base of the nose. At the nose neglect any effect due to body blunting. Insert the two resulting values of in radians into:

$$C_W = \frac{1}{(1 - .396 \theta_c + .187 \theta_c)^2} - .124 \theta_c)^3$$
 (1)

Average the two calculated values of  $C_W$  to obtain the final one to be used in the code. If ALPHA is non-zero increment  $\theta_{\mathfrak{C}}$  by ALPHA.

- (2) FLAT PLATES. Use a value of 1.45 for ANG>45 degrees and 1.55 for ANG<45 degrees.
- (3) SPHERICAL BODIES. Use a value of 1.55 for near vertical entry and 1.35 for oblique entry.

The classification of an arbitrary body into one of the above categories is a matter of experience. On complex shapes classification should be based on the portion of the body sustaining the majority of the impact loading.

If the VARIABLE body orientation option is used the following data cards are required:

Card No. Variables		Format
4 5 6 7.1	<pre>IMAX, D, VENTRY, ANG, SUMT, HMIN CG., FCF, NNLD NVP VX(1), VY(1), VZ(1), WX(1), CW(1), DH(1)</pre>	I5, 5X, 7F10.0 2F10.0, 10X, I5 I5 6F10.0
7.NVP	VX(NVP), VY(NVP), VZ(NVP), WX(NVP), CW(NVP), DH(NVP)	6F10.0

The variables on cards 4 and 5 are defined above. In this case, VENTRY is only used in determining the force and pressure coefficients and ANG is the initial body orientation. The body velocity, wetting factor, and increment in depth for each step is defined in cards 7.

NVP Number of different steps at which entry conditions are specified.

VX(I), Velocity components in the x, y, z directions of the center VY(I), of gravity (in in./sec.) applied between steps I-l and I. VZ(I)

WX(I) Angular velocity (in degrees/sec.) in the pitch (y-z) plane applied between steps I-1 and I.

- CW(I) Wetting factor applied between the I-l and I step. If the value of this parameter remains constant from step to step use the instruction for determining this variable given in the CONSTANT orientation section. For the vertical entry (VX=VY=0) of pointed bodies an estimate of this parameter for each step can be obtained by:
  - a. Determining the depth of the entry body, H, below the original surface at the start of the step.
  - b. Calculating the angle,  $\theta_{c}$  between a tangent to the body surface and the body axis, z'=H.
  - c. Substituting into equation (1) to determine  $C_{\boldsymbol{W}}$  where

$$\theta_c = \theta_c' + 90 - ANG$$

- d. For blunt bodies, (nose length/diameter < .75), increase this angle by 7% on ogives and decrease it by the same amount on cusps.
- DH(I) Increase in depth (in inches) of the center of gravity between steps I-1 and I. See instructions in the CONSTANT orientation entry section.

The entry velocity at step I is taken to be the average of that at steps I-1 and I. It is only necessary to specify data cards for the first few steps in which the above parameters change. For steps larger than NVP the parameter values at step NVP are used.

The final set of data cards is used to define the grid on the surface of the entry body. The three available options for constructing a grid on the body surface are STANDARD, OGIVE, AND LIST. These can be used singularly, in combination with one another and can be called in arbitrary sequence. The only restriction is that the lowest point on the body should occur on that part of the grid constructed by the first option called. To indicate the desired options, the following input cards are required:

Card No.	Variable	Format
8 9.1 •	N option 1	I5 3A4
9.N	option N	3A4

Here N is the number of options to be used. The recommended options are STANDARD and LIST. Description of the OGIVE option will not be given in these instructions. For information on this option, refer to Reference 19.

The grid representing the surface of the entry body should cover only the nose of the model and not the afterbody. In all cases the pressures on the afterbody are small. Furthermore, on bodies with sharp shoulders such as a disk cylinder, the flow separates at the edge of the model face. If the afterbody is gridded, the flow is required not to separate since the invicid boundary conditions are enforced at the centroid of each element. This is physically unrealistic and hence neglecting the afterbody is appropriate.

A description of the three available options follows. Under no circumstances should right angles be modeled directly. If the body under consideration has such a surface discontinuity, it should be modeled with a 89.9 or 90.1 degree angle.

#### STANDARD

IW(I)

This option is applicable to axisymmetric bodies or axisymmetric portions of arbitrary bodies. The user specifies rings along which nodes are located. Adjacent nodes are combined to form elements. A typical grid for a flat, circular plate is shown in Figure G-4. The required input is:

Card No.	Variables	Format
10 11.1	NROWS, IANG, ISUP R(1), Z(1)	3I5 2F10.0, I5
11.NRO	WS R(NROWS), Z(NROWS), IW(NROWS)	2F10.0, I5
NROWS	Number of grid rings.	
IANG	If IANG=0, only half of the face is gridded as shalf IANG=1, the complete face is gridded.	own.
ISUP	If ISUP=1, the stagnation element (element number removed. This option is used for running pointed For such bodies, R(1) should be very small (i.e., must be finite.  If ISUP=0 this element is included.	objects.
R(I)	Radius of ring I in body fixed coordinates $(x',y')$ inches).	,z', in
Z(I)	z' coordinate of ring (in inches).	

Number of elements in the area between rings I and I-1.

Delete this variable on card 1. If IW=0, elements are

automatically selected so that they are approximately square.

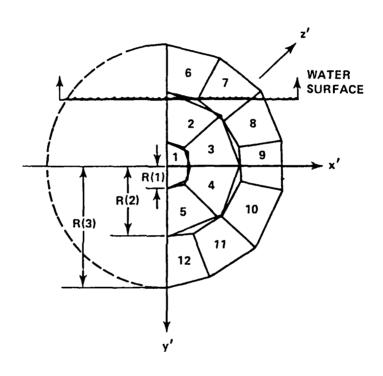


FIG. 6-4 GRID OF A CIRCULAR PLATE

# LIST

This option requires that the user input the list of nodes and elements to be used in the run and hence is applicable to arbitrary bodies. The nodes can be read in any order, however, they are numbered sequentially for internal use in the code. Each element is constructed using nodes from the input list. The identification numbers of nodes defining the four corners of each element must be read in a clockwise order with respect to an observer on the outer surface of the element. The required input cards are:

Card No.	Variable	Format
10 11.1	NP, NE x'(1), y'(1), z'(1)	2I5 3F10.0
11.NP 12.1	x'(NP), y'(NP), z'(NP) IN(1,1), IN(2,1), IN(3,1), IN(4,1)	3F10.0 4I5
12.NE	IN(1,NE), IN(2,NE), IN(3,NE), IN(4,NE)	415
NP	Number of node points to be read in.	
NE	Number of elements to be read.	
x'(I), y'(I) z'(I)	Location of the Ith node in body fixed coordinate z', in inches).	es (x', y',
	Identification number of the nodes defining the sof the element.	four corners

#### PART 2

#### SPECIAL INSTRUCTIONS BEFORE GENERATING FEM

ENTRY includes three options to define the grid on the surface of the entry body: STANDARD, OGIVE, AND LIST. The first two options are associated with the automatic grid generation scheme built into the ENTRY code and are not suitable for WEST application. The LIST option requires the user to input the list of nodes and elements to be used in the runstream, and hence is applicable to arbitrary bodies and linkage with the PATRAN pre-processor. The translator PATENTR was developed to translate the PATRAN neutral file (a PATRAN output ASCII file) defining the finite element mesh of the entry body into the water entry model, with the same mesh but in LIST option format suitable for use in ENTRY. ENTPRES is programmed to convert the ENTRY-calculated element pressures and their arrival times into time-varying load specifications in ABAQUS format for dynamic analysis and in YADAP format for time-history plots on a personal computer. In order to make both translations successful, however, several restrictions imposed by ENTRY need to be followed:

- (1) In dividing the surface of an entry body into rings and subdividing rings into elements, the node and element identification numbers (ID's) must be numbered sequentially, from tip to tail and from left to right. If neither begins with the number one (1), a number offset must be input for both node and element ID at PATENTR run time. The user will be prompted for them at that time.
- (2) Only four-noded quadrilateral (QUAD) elements are permitted in ENTRY. The element connectivities which are generally defined in counter-clockwise convention in FEA, will be reversed by PATENTR for ENTRY. The element pressures calculated by ENTRY are always positive, and are treated as positive only when applied in the outward normal direction. This direction is determined by the element connectivity defined in PATRAN following the right-hand rule. Based on this knowledge, the user must decide whether positive or negative pressures are to apply to the shell elements in the ABAQUS analysis. The user will be prompted for the sign of pressures to be output in an ENTRY run when the element pressure-time history output option is activated.
- (3) Define the axis of the entry body as the z-axis. If a FEM is already built with x- or y-axis defined as the body-axis, PATENTR can convert it into a z-axis model for ENTRY. This is done simply by inputting an 'x' or a 'y' when prompted for the body-axis coordinate in the FEM at PATENTR run time. Note that this conversion is necessary only when running ENTRY. Since pressures are independent of the coordinate system chosen, a user can still keep his own body-axis when running ABAQUS downstream.

- (4) Although it is not necessary to define the tip of the entry body as the origin of the body-axis, it is recommended to do so. This action will permit determination of the pressure arrival time for each ring of elements to be timed from the tip. This will subsequently facilitate the chosing of time-steps in the ABAQUS FEA. Move the origin to the tip by inputting a z-axis shift (positive if the tip of the ENTRY model is above the origin of the PATRAN FEM) when prompted at PATENTR run time.
- (5) It is recommended in the generic use of ENTRY that the increment of depth for water entry be chosen such that the average element is submerged in two steps. To couple with ABAQUS, however, it is necessary to choose the step size precisely so that each ring of elements of the entry body will be submerged in exactly one step. This is because a pressure cannot be applied to only half an element. Since efficient FEM often results in non-uniform mesh density, this situation leads to chosing variable step sizes in ENTRY to properly submerge each ring of elements in a single step. The times required to submerge the variable step-sized rings are computed in ENTRY for given body orientation and entry conditions and can be used to synchronize with the pressure-arrival time for each ring of elements in an ABAQUS analysis.
- (6) The only symmetric condition the ENTRY code can take advantage of is half-symmetric, with the y-z plane chosen as the plane of symmetry, for either vertical or oblique entry. However, an experienced user can take better advantage of symmetry conditions in a subsequent ABAQUS analysis by using a quarter- or eighth-symmetric FEM. This becomes automatic with the element grouping technique implemented in ENTPRES for the case of axi-symmetric bodies entering the water vertically.
- In the case of vertical entry with a zero angle of attack for an axi-symmetric body surface, the resulting axi-symmetric pressure distribution varies only in the axial direction as the body is submerged. Elements in each ring at any given depth will experience the same pressure. To facilitate the element pressure-time history input for ABAQUS, an element grouping technique is implemented in The user is prompted for the number of elements in a ring ENTPRES. that have the same pressure at ENTPRES run time. Therefore, it is recommended to form a FEM with the same number of elements in each ring of the entry body. This grouping limitation excludes the stagnation elements at the tip of the ENTRY model. The user will also be prompted for the number of such elements included in the model. This exclusion increases the flexibility in FEM by allowing the user to use triangular (TRI) shell elements at the tip of the body while using QUAD elements formed by two TRI shell elements for ENTRY calculation. The neartriangle shaped QUAD elements used in ENTRY must therefore be divided into two TRI elements for ABAQUS. The grouping technique implemented in ENTPRES allows this mixed use of QUAD and TRI elements in ENTRY and ABACUS, respectively.

## PART 3

#### COMPUTER RUN STREAM EXAMPLE

PATENTR, ENTPRES, and options in ENTRY are prompted at run time. Options to output pressure-time history files in PATRAN neutral file format, options for ENTPRES' conversion to ABAQUS load format for analysis, and options to produce YADAP format for time-history plots are programmed in an interactive manner. Input/output filenames and control parameters are also prompted at run time. Table G-l is a sample listing of a VAX 11/785 computer run stream that begins after a PATRAN session to generate a FEM of a blunt, 90-degree cone impacting water vertically at 100 ft./sec.. Some questions that are annotated require explanations:

- (1) CONE90.IN is the file that contains two sets of control parameters for water entry. The first set specifies the basic program options for ENTRY, such as constant or variable body entry orientation, option to print step by step flow field and element information, and the entry body's symmetry condition. The second set specifies the water entry conditions such as entry velocity, angle of attack, increment of depth, and wetting factor. This file is then appended by the PATENTR-converted LIST grid file of the cone (CONE.LST) to form the complete input data to run ENTRY. The complete listing of input data for ENTRY is given in Appendix A.
- (2) There are two options to input the LIST grid file for ENTRY. The first option is appending the LIST grid file to the ENTRY run control data file to form a single file for the ENTRY run, as described in note (1) above. The second option is to treat the LIST grid file as a separate external file during the ENTRY run. Since the first option was chosen for this example, the answer to the prompted question is no. If the second option is taken, the answer is yes, and the user will be prompted for the name of the LIST grid file.
- (3) As shown in Figure 3 of the main text, the inclined surface of the cone is divided equally into 10 rings, with 8 QUAD elements each. The flat tip is modeled with four near triangle-shaped QUAD elements, as shown in Figure 4.
- (4) CONE90.INP is an ASCII file output from running the PATABA interface. This file contains all but the loading data required for an ABAQUS analysis. It includes the FEM definition of the cone, analysis type selection, material and section property cards for elements and boundary condition specifications. This file is appended by ENTPRES converted time-varying loads (PRES.ABA) for dynamic analysis with ABAQUS. A complete listing of input data for an ABAQUS execution is given in Appendix C.

```
Table G-1. List of VAX 11/785 sample computer run stream.
```

```
$ RUN PATENTR
 ENTER INPUT FILENAME--PATRAN NEUTRAL FILE W/ EXTENSION
CONE90.NEU
 ENTER OUTPUT FILENAME--ENTRY "LIST" GRID OPTION FILE
CONE90.LST
 IS NODE OR ELEM NUMBER SHIFT REQUIRED FOR ENTRY? Y/N
  IF YES, ENTER NODSFT AND NELSFT IN 215 FORMAT.
  NOTE: NODE & ELEM ID MUST BEGIN WITH 1 & IN SEQUENCE
 ENTER BODY-AXIS CHOSEN IN FINITE ELEM MODEL, X,Y OR Z
  NOTE: NON-Z SYSTEM WILL BE CONVERTED TO Z-X-Y SYSTEM
        WITH Z AS BODY-AXIS & Y-Z AS PLANE OF SYMMETRY
 IS AXIAL OFFSET REQUIRED FOR ENTRY MODEL? Y/N. IF YES,
  ENTER SHIFT (POSITIVE IF TIP IS ABOVE ORIGIN OF FEM)
  NOTE: THIS ALLOWS WETTING TO BE TIMED FROM THE TIP.
END OF INPUT FILE ENCOUNTERED--NORMAL TERMINATION
FORTRAN STOP
$ APPEND CONE90.LST CONE90.IN
                                                                 (1)
 RUN ENTRY
 ENTER INPUT FILENAME INCLUDING EXTENSION
CONE90.IN
 ENTER OUTPUT FILENAME INCLUDING EXTENSION
CONE90.OUT
 ENTER OUTPUT FILENAME FOR ELEMENT PRESSURE-TIME
  HISTORIES IF WANTED, TYPE NO IF NOT WANTED
CONE90.PRE
 ENTER FILENAME FOR TOTAL FORCE-TIME HISTORIES, DRAG &
  NORMAL FORCES & MOMENTS @ CG IF WANTED, TYPE NO IF NOT
CONE90.F
 IS LIST INPUT DEFINING GRID FROM AN EXTERNAL FILE? Y/N
                                                                 (2)
FOLLOWING ENTRIES COMFORM TO PATRAN NEUTRAL FILE FORMAT
  FOR DISTRIBUTED LOADS:
  (1) TITLE CARD (DATA PACKET TYPE 25)
  (2) SUMMARY DATA (DATA PACKET TYPE 26)
  (3) DISTRIBUTED LOADS (DATA PACKET TYPE 6)
 ENTER TITLE OF PRESSURE-FILE (LIMIT 80 CHARS)
PRESSURE-TIME HISTORIES OF 100 FPS VERTICAL ENTRY BLUNT 90-D CONE
 ENTER 6 LOAD COMPONENT FLAGS (ICOMP=0/1) AT 611 FORMAT
   EX: FOR PRESSURE LOADING ON QUAD ELEM, ENTER 001000
001000
 ENTER 8 ELEM NODE FLAGS (NODE=0 OR 1) AT 811 FORMAT.
   1 MEANS THAT NODE IS ON THE LOADED EDGE OR SURFACE.
   EX: FOR A PRESSURE-LOADED QUAD ELEM, ENTER 11110000
11110000
 ENTER 1ST LOAD SET ID (IV=1) & ELEM FACE NO. (NFE=1-6)
   AT 215 FORMAT. NFE=0 FOR A PRESSURE-LOADED QUAD ELEM
 ARE PRESSURES APPLIED TO PATRAN MODEL +(P) OR -(N)? P/N
   NOTE: PRESSURE IS + IN POSITIVE NORMAL FOR QUAD ELEMS
FORTRAN STOP
```

## Table G-1. (continued)

```
$ RUN ENTPRES
 ENTER INPUT FILE OF ELEM PRESSURE HISTORY, EX: CONE.PRE
 ENTER OUTPUT FILE FOR ABAQUS FE ANALYSIS, EX: PRES.ABA
PRES.ABA
DO YOU WANT TAPE9.DAT TIME-HISTORY PLOT FILE? Y/N
  NOTE: TAPE9.DAT FORMAT IS 12E11.4, WITH ELAPSED TIMES
         IN FIRST COLUMN AND UP TO 11 GROUP PRESSURES
         THEREAFTER. IF MORE THAN 11 GROUPS OF OF DATA
         ARE ENCOUNTERED, THEY WILL BE SPREADED OVER TO
         FOR010, FOR011, FOR012.DAT...IN THE SAME FORMAT
Y
 ENTER TITLE FOR TAPE9.DAT PLOT FILE (80 CHARS MAX)
PRESSURE-TIME HISTORIES OF 100 FPS VERTICAL ENTRY BLUNT 90-D CONE
 ENTER NOS. OF STEPS (NSTP, 15) AND NOS. OF ELEMS IN A
   GROUP (NEL, 15) WITH THE SAME PRESSURE IN INPUT FILE
   NOTE: IF ALL ELEMS HAVE DISTINCT PRESSURES, SET NEL=1
                                                                  (3)
 ARE STAGNATION ELEMS INCLUDED IN WATER-ENTRY MODEL? Y/N
Y
 4
                                                                  (3)
   IF YES, ENTER NUMBER OF STAGNATION ELEMS (NSTG,12)
   NOTE: STAGNATION ELEMS ARE TREATED AS GROUP NO. 1
 SUMMARY OF PROCESS:
 MAX NUMBER OF PRESSURIZED ELEMS WITHIN STEPS
 NUMBER OF ELEM GROUPS (WITH EQUAL ELEM PRESSURE)=
 NUMBER OF ELEMS PER GROUP (EXCL STGN ELEM GROUP)=
                                                      8
 NUMBER OF ELEMS IN STGN ELEM GROUP (GROUP NO. 1)=
                                                      4
 NUMBER OF TIME-HISTORY PLOT FILES GENERATED
FORTRAN STOP
$ SET DEF [RCSHAW]
$ APPEND [.ENTPRES]PRES.ABA CONE90.INP
                                                                  (4)
$ @ABAQUS
 IDENTIFIER
                                                            : CONE
 INPUT FILE NAME (W/O .INP)
                                                            : CONE90
 RESTART READ FILE ?
 OLD 'FILE OUTPUT FILE' FILE
 BATCH QUEUE YOU WANT (SYS=<CR> SLOW=S) ?
 GIVE THE AMOUNT OF TIME YOU WANT TO HOLD THE JOB FOR
 THE FORMAT IS HH:MM: ( <RET> TO IGNORE)
Job CONE (queue SYS$BATCH, entry 350) started on SYS$BATCH
Job CONE (queue SYS$BATCH, entry 350) completed
$ DIR CONE
Directory USER2: [RCSHAW]
CONE.COM; 1
                    CONE.DAT;1
                                       CONE.FIL;1
                                                             CONE.LOG; 1
CONE.STA; 1
Total of 5 files.
```

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